

**FAKULTA  
STAVEBNÍ  
ČVUT V PRAZE**

# **DIPLOMA PROJECT**

**Design of multistorey steel car park  
Návrh budovy patrových garáží**

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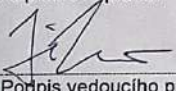
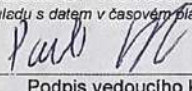
Prague, 2022

## ZADÁNÍ DIPLOMOVÉ PRÁCE

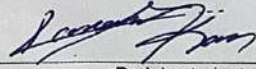
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Název diplomové práce anglicky: Design of multistorey car park	
Pokyny pro vypracování: Design multistorey car park building with steel structural framework and concrete floors. Prepare plan for parking and circulation of cars. Design floor and bracing system. Design typical details and connections and main member sizes. Prepare documentation of structural analysis. Prepare drawings of the typical floor plan, anchoring plan, elevations and typical cross-sections including ramps.	
Seznam doporučené literatury: EN 1993-1-1, Eurocode 3: Design of steel structures - Part1-1: General rules for buildings EN 1993-1-8, Eurocode 3: Design of steel structures - Part 1-8: Design of joints Economical Carparks, A Design Guide, OneSteel <a href="https://www.steelconstruction.info/Car_parks">https://www.steelconstruction.info/Car_parks</a>	
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### Declaration of Authorship

I declare that I have prepared the submitted work independently and that I have listed all the information sources used in accordance with the Methodical Guidelines on the Ethical Preparation of University Thesis.

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Ramazan Koca

Prague, 02.01.2022

# MULTI-STOREY STEEL CAR PARK DESIGN

## Abstract

This thesis deals with design of multi-storey steel carpark which is planned for Prague. The structure has four storey , 12.6 meters height, 34 meters width and 111 meters long.

The steel framework is analysed in SCIA software. All hand calculations have been programmed in Python language by author and displayed in readable form in Jupyter notebook. A snapshot of the Python code is shown in Appendix.

## Keywords

Car Park design, SCIA, Steel structure design, multi-storey carpark, steel framed structure, double tee floor system.

## Acknowledgement

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## 1.Introduction

Multi-storey car parks are essential buildings for cities with high population and for places that people gathered commonly such that train stations, airports, hospitals and shopping centres.

Car Park structures are often built above ground level as permanent structures however underground and basement car parking is used where it is necessary.

Steel construction is one of the best solutions to satisfy all the requirements of fine car park design. Since steel is:

- \* Ideal for long spans – providing column-free parking space
- \* Lightweight – reducing foundation requirements
- \* Robust and fire resistant (fire solutions do exist but it is not a subject of this study)
- \* Fast in construction – particularly relevant where the venue that is to be served is to remain operational during construction, e.g., retail, stations and hospitals
- \* Easily maintained
- \* Vandal resistant
- \* Economic.

A satisfactory steel car park design should fulfil some major objectives. These are:

- \* Utility or function (strength and serviceability)
- \* Safety (permanence)
- \* Economy (Low maintenance)
- \* Elegance
- \* Easy entry and exit to the car park and the parking stalls
- \* Uncomplicated and logical traffic flow around the car park
- \* Unimpeded movement
- \* Light and airy

The aim of this study is to present what are the steps for designing multistorey steel carpark and what is the design process for a given specific geometry of the structure. During designing process, the calculation is done using an advanced tool called Jupyter notebook. This tool is based on object-oriented programming language called Python. Procedures and algorithms were prepared and coded by the author of this thesis. This means that the final product of this thesis is not only the technical documentation itself but also a set of software tools that can be utilized in the future. Advanced analyse is obtained by SCIA Engineering Software, drawings are prepared by using AutoCAD and for 3D illustration TEKLA Structure.

## 2. Design Plan

The structural design plan shows us the phases from initial conception to the final plans and specifications. In this study the phases up to acceptance of structural project is planned, so let us consider the following list of phases and in detail of first three phases in this study:

A- Initial planning

B- Preliminary design

- C- Final design
  - a. Analysis
  - b. Selection and proportioning of elements
  - c. Drawings, specifications and other contract document
- D- Acceptance of project
- E- Construction
- F- Operation and maintenance

## 2.1. Initial planning

Initial planning starts with the requirements of the structure. At the stage of initial design, the geometry of the building, entrance, the capacity of parking space, parking bays and traffic flow are decided as shown below. In our study, building geometry is 34 meters wide and 111 meters long, traffic flow goes two ways with 90 ° angle parking. The parking floors are sloped for traffic to circulate from one level to another, which is shown in drawings at part 4, and satisfy the limit for interior ramp located at the middle of the structure. The full capacity of parking building is 501 vehicles.

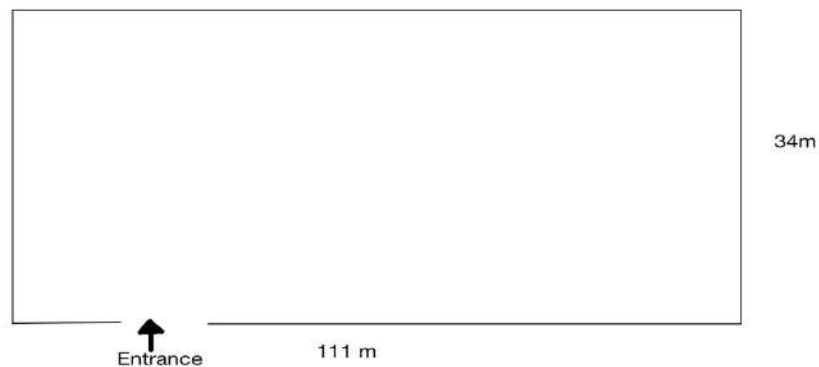


Figure 2. 1: Building geometry and entrance

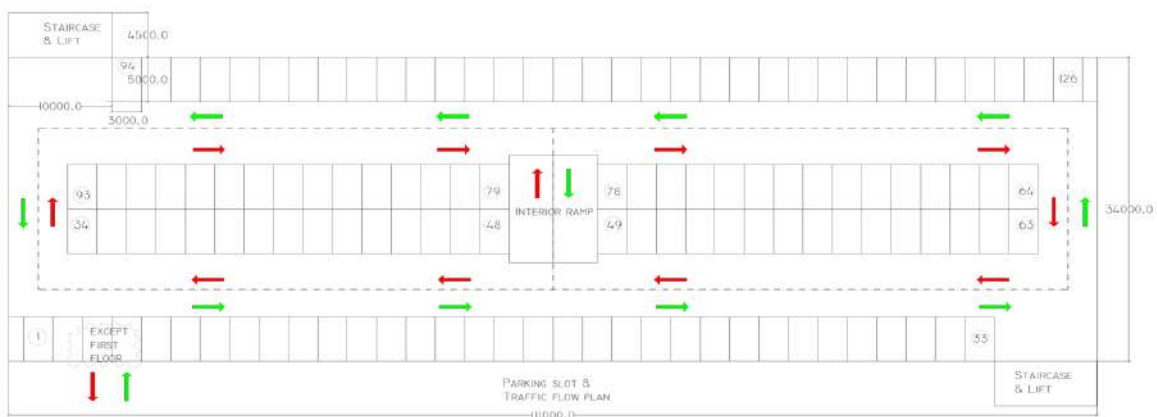


Figure 2. 2: Parking slots and traffic flow

The dimensions of large, standard and small size cars are well established and are given by the Institution of Structural Engineers [1]. According to the table the chosen aisles, parking angle and stall required at least 16.55 m bin width. The minimum dimensions based on a standard car, the bin width, the parking angle and stall width are shown in the following Table 2.1.1.

**Effect of varying parking angle on parking bin requirements**

Parking angle	Stall width (m)	Stall width (m) parallel to aisle	Aisle width (m)	Bin width (m) for stall length 4.80m
45°	2.3	3.25	3.6	13.65
	2.4	3.39		13.80
	2.5	3.54		13.95
60°	2.3	2.66	4.2	14.85
	2.4	2.77		14.95
	2.5	2.89		15.05
75°	2.3	2.38	4.98	15.45
	2.4	2.49		15.50
	2.5	2.59		15.55
90°	All widths	All widths	One way aisle 6.0	15.60
90°	All widths	All widths	Two way aisle 6.95	16.55

*Table 2.1. 1: Effect of varying parking angle on parking bin requirement*

## 2.2. Preliminary design stage

During the study, linear elastic analysis is assumed. All results are obtained and valid at elastic range.

The Standards for the material give values for the minimum yield strength ( $f_y$ ) and ultimate strength( $f_u$ ); it should be noted that the yield strength is normally taken as the design strength. Steel grade was considered as S355 for primary members and S275 for secondary elements.

The following Table 2.2.1 shows value for steel strength at specified thickness range.





The loading on foundations is greatly influenced by the material chosen for the superstructure. Steel is the lightest practical construction material for car parks and will often allow the use of simple foundations however heavier materials will not. On the other hand, foundation calculation depends on soil conditions, but this study is related to steel framework only. Foundation analysis is not part of this project but are shown in drawings just for illustration.

### 2.2.2. Preliminary beam design

The case study, which is shown below both as plan view and 3D view, has 4 stories and 111 m long and 34m wide. The first storey is 3.5m, second and third stories are 3.0 m, and fourth story is 3.1 m height.

The designs are based on the following assumptions:

- \* Design is given for internal and edge beams where appropriate for one bay of a car park
- \* Imposed loading is taken as  $2.5\text{kN/m}^2$
- \* Grade S355 is used for all main and secondary beams
- \* Grade S275 is used for all ties
- \* Approximate weights of steel given are based on a car park 111 m long x 34m wide with car parking spaces at 2.4m wide.
- \* Column sizes are based on the lower length of a 4-storey car park
- \* Imposed load deflection limit:  $\text{Span}/360$
- \* Total load deflection limit:  $\text{Span}/150$
- \* Preliminary design is applied for linear analysis only

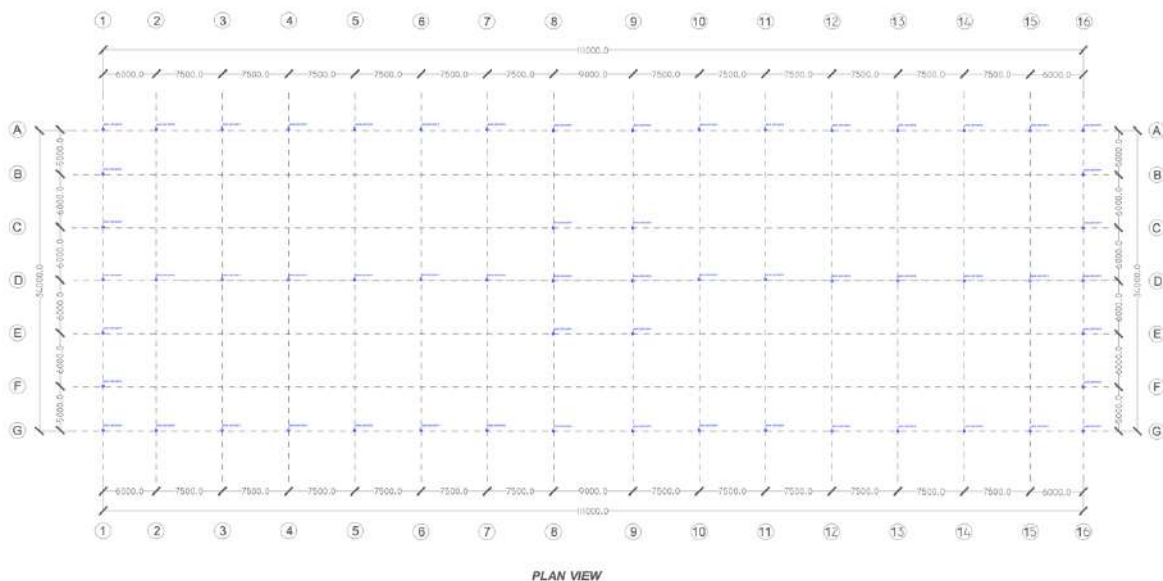


Figure 2.2.2. 1: Plan view

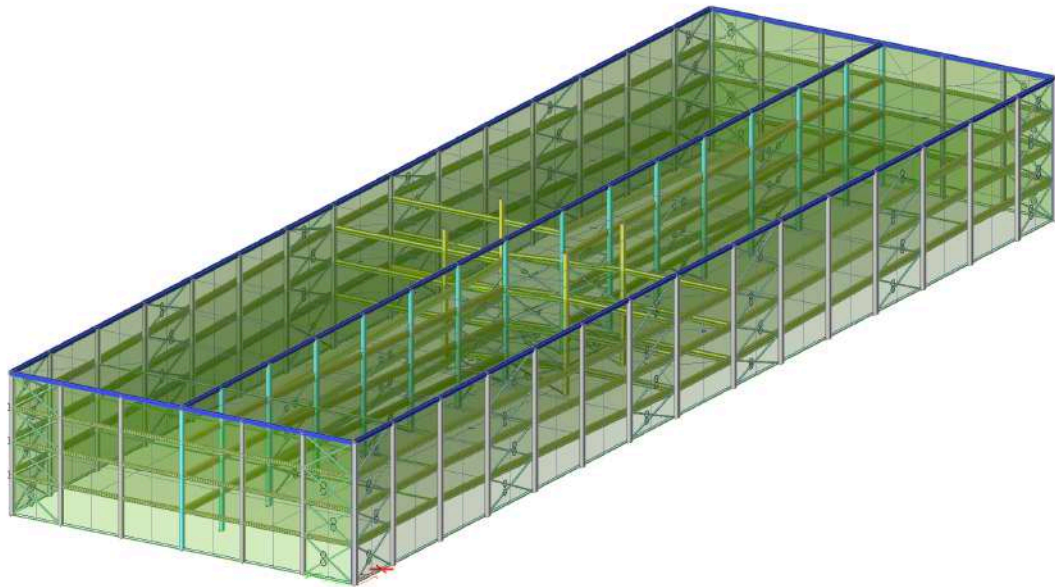


Figure 2.2.2. 2: 3D SCIA Model

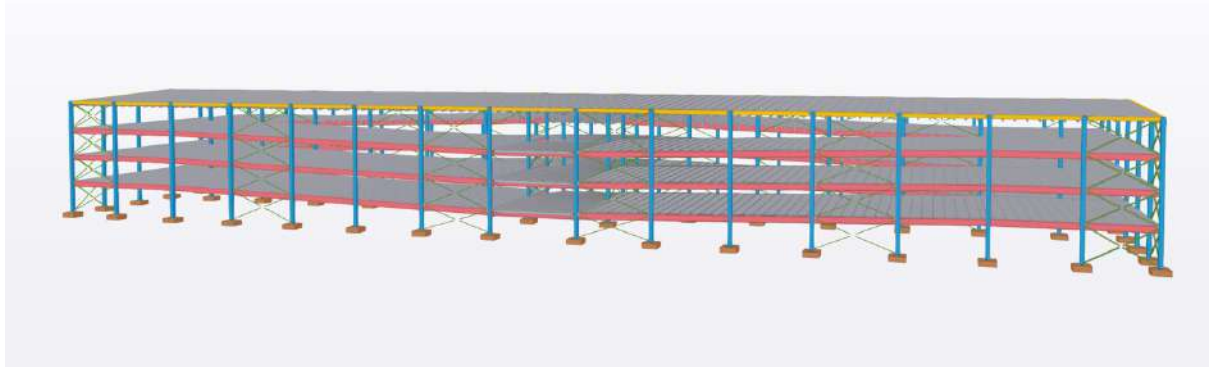
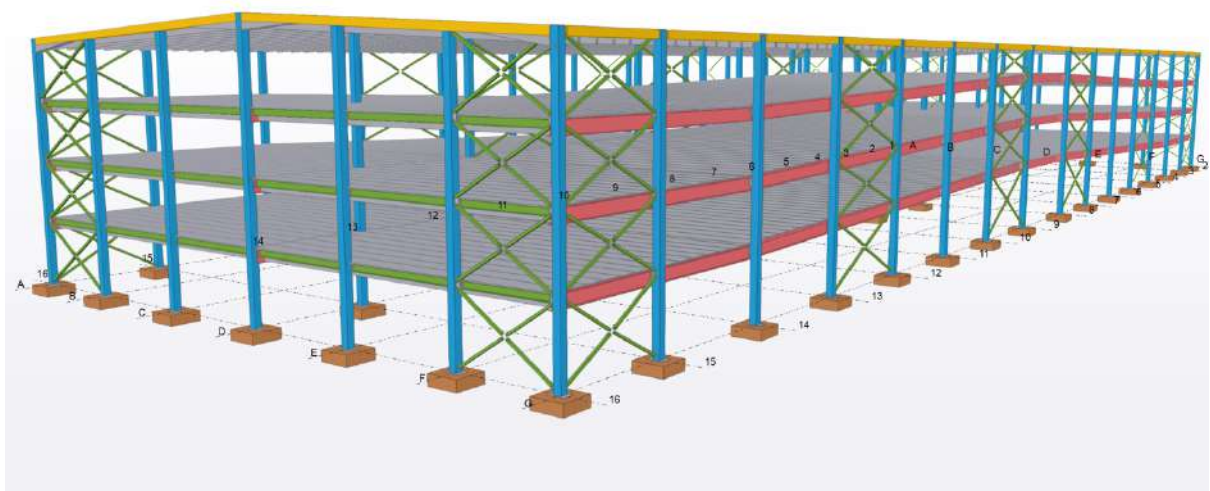


Figure 2.2.2. 3: Tekla model

The continuous beam as shown below has its top flange fully restrained laterally by a precast slab supported on secondary beams. Double tee flooring systems require a minimum of 75mm or span/180, whichever is greater, seating on beams. Concrete supports, double tee slab legs, are on steel beams the end seating can be reduced by 15mm. A construction tolerance of 10mm needs to be added to these figures. Double Tees can be supported at the end on the 55mm thick flange using hanger brackets. This avoids the contractor having to form the beams between the legs and allows the depth of the tee to be almost entirely within the required depth of the supporting beam. Other options are partial leg support and full leg support. The way of double tee flooring system is attached to beam flange is shown below.

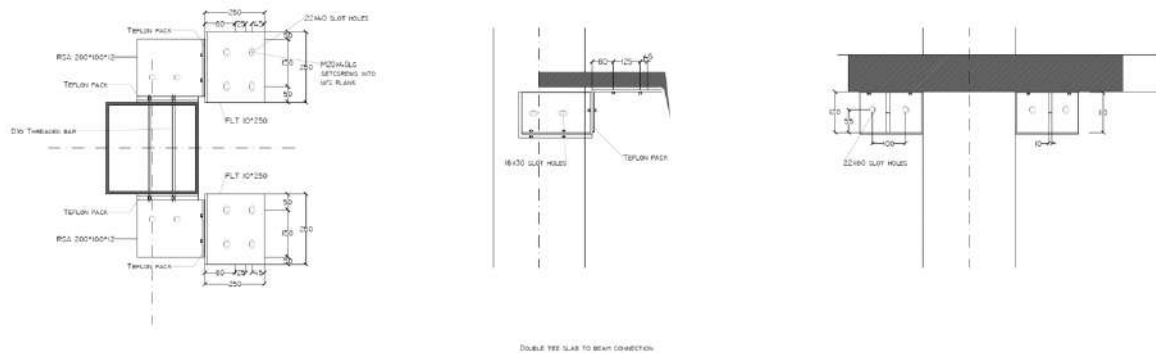


Figure 2.2.2. 4: Double tee floor to beam flange connection

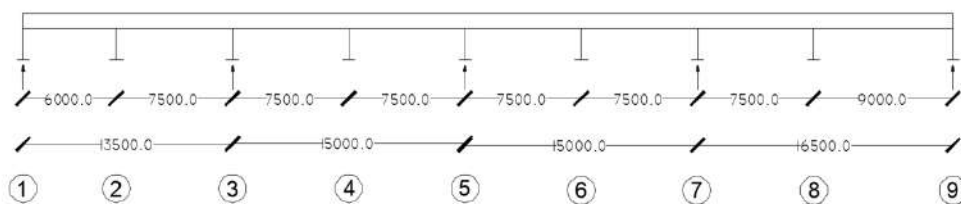


Figure 2.2.2. 5: Simplified continuous beam model

All actions are considered as concentrated loads acting at the nine numbered locations. Only the forces at 2,4,6 and 8 give rise to bending moments and shear forces.

### 2.2.2.1. Actions-Loading

Location: Czech Republic-Prague

Snow load= 0.7 kN/m<sup>2</sup>

Wind Load = 0.32 kN/m<sup>2</sup>

For category F, q<sub>k</sub> may be selected within the range 1.5 to 2.5 kN/m<sup>2</sup> and Q<sub>k</sub> may be selected within the range 10 to 20 kN [2]



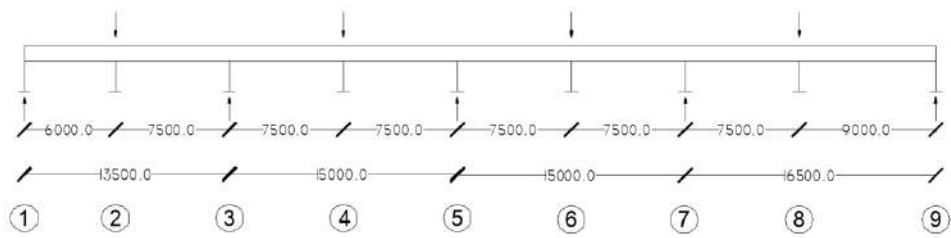
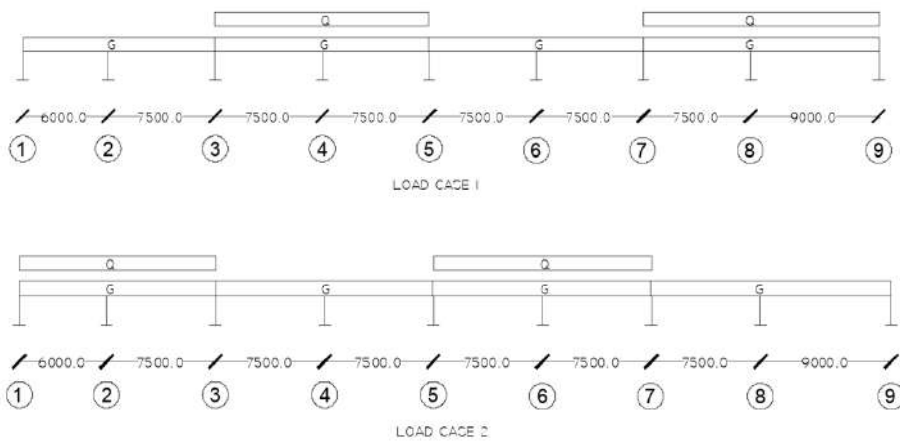


Figure 2.2.2.1. 2: Actions

### 2.2.2.2. Design bending moments and shear forces

For continuous beams with slabs in building without cantilevers where uniformly distributed loads are dominant, it is sufficient to consider only arrangement of actions shown in Figure 2.2.2.2. 1.



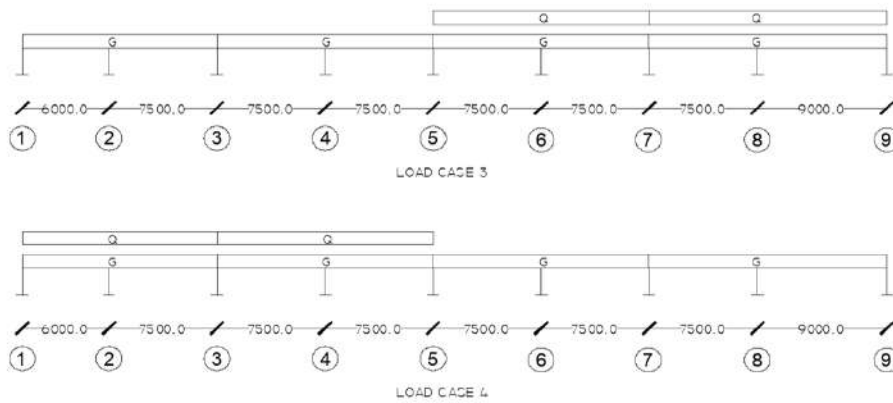


Figure 2.2.4. 1: Load Cases

The design bending moment and shear force diagrams are shown below. From inspection, the most onerous design values are obtained using the load case 1.

Maximum design bending moment occurs at point 8 for load case 1

$$M_{Ed} = 496.94 \text{ kNm (Figure 3.7)}$$

Maximum design shear force occurs at point 8 for load case 1

$$V_{Ed} = 334.18 \text{ kN}$$

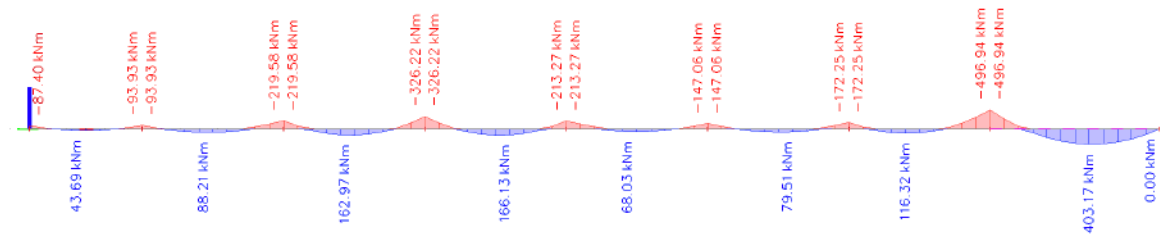


Figure 2.2.4. 2: Bending moment

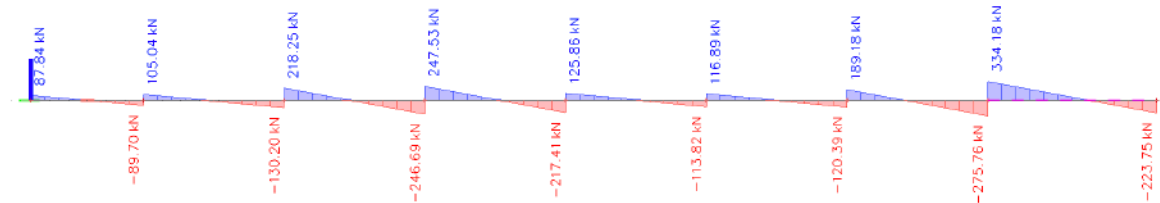


Figure 2.2.4. 3: Shear force

### 2.2.2.3. Beam design

#### Section properties of chosen beam, HEA600

$h = 590.000 \text{ mm}$  (depth)  $b = 300.000 \text{ mm}$  (width)  $t_w = 13.000 \text{ mm}$  (thickness)

$t_f = 25.000 \text{ mm}$ (flange thickness)  $r = 27.000 \text{ mm}$ (root radius)

$m_w = 177.800 \text{ kg} \cdot \text{m}^{-1}$ (weight)

$d = 486.000 \text{ mm}$  (depth between fillets)

$I_y = 1412000000.000 \text{ mm}^4$  (second moment of area)

$i_y = 249.700 \text{ mm}$ (radius of gyration)

$I_z = 112700000.000 \text{ mm}^4$  (second moment of area)

$i_z = 70.500 \text{ mm}$ (radius of gyration)

$I_T = 4075000.000 \text{ mm}^4$ (torsion constant)

$I_w = 8.8790 \cdot 10^{12} \text{ mm}^6$ (warping constant)

$A_v = 9321.000 \text{ mm}^2$

$Wpl_y = 5350000.000 \text{ mm}^3$  (plastic section modulus)

$Wpl_z = 1156000.000 \text{ mm}^3$  (plastic section modulus)

$Wel_y = 4787000.000 \text{ mm}^3$  (elastic section modulus)

$Wel_z = 751400.000 \text{ mm}^3$  (elastic section modulus)

$A = 22646.000 \text{ mm}^2$  (area)

$E = 210.000 \text{ MPa}$ (elasticity modulus)

$f_y = 355.000 \text{ MPa}$ (yield strength)

$f_u = 490.000 \text{ MPa}$ (ultimate strength)

$\gamma_{M0} = 1.0$  (partial safety factor)

$\gamma_{M1} = 1.0$  (partial safety factor)

#### Cross-section classification

$$\varepsilon = \left( \frac{235 \cdot \text{MPa}}{f_y} \right)^{\left(\frac{1}{2}\right)}$$

$$\varepsilon = \left( \frac{235 \cdot \text{MPa}}{355.000 \text{ MPa}} \right)^{\left(\frac{1}{2}\right)}$$

$$\varepsilon = 8.136 \times 10^{-1}$$

#### • Outstand of compression flange

$$c = \frac{b - t_w - 2 \cdot r}{2} = \frac{300.000 \text{ mm} - 13.000 \text{ mm} - 2 \cdot 27.000 \text{ mm}}{2} = 116.500 \text{ mm}$$

$$\frac{c}{t_f} = \frac{116.500 \text{ mm}}{25.000 \text{ mm}} = 4.66$$



The limiting value for Class 1 is  $\frac{c}{t_f} \leq 9\epsilon = 7.8624$

The flange in compression is Class 1

- **Web subject to bending**

$$c_b = 486.000 \text{ mm}$$

$$\frac{c}{t_w} = \frac{116.500 \text{ mm}}{13.000 \text{ mm}} = 8.962$$

The limiting value for Class 1 is  $\frac{c}{t_w} \leq 72\epsilon = 62.8992$

The web under bending is Class 1

### Cross-sectional resistance

- **Shear buckling**

The shear buckling resistance for webs should be verified according to section 5 EN 1993-1-5 [4] if:

$$\frac{h_w}{t_w} > 72 \cdot \frac{\epsilon}{\eta}$$

$$\eta = 1$$

$$h_w = h - 2 \cdot t_f = 590.000 \text{ mm} - 2 \cdot 25.000 \text{ mm} = 540.000 \text{ mm}$$

$$limit = \frac{h_w}{t_w} = \frac{540.000 \text{ mm}}{13.000 \text{ mm}} = 41.538$$

$$limit_2 = 72 \cdot \frac{\epsilon}{\eta} = 72 \cdot \frac{8.136 \times 10^{-1}}{1} = 58.58$$

The shear buckling resistance of the web does not need to be verified.

- **Shear resistance**

$$V_{plRd} = \frac{A_v \cdot \left( \frac{f_y}{(3)^{\left(\frac{1}{2}\right)}} \right)}{\gamma_{M0}}$$

$$V_{plRd} = \frac{9321.000 \text{ mm}^2 \cdot \left( \frac{355.000 \text{ MPa}}{(3)^{\left(\frac{1}{2}\right)}} \right)}{1.0}$$

$$V_{plRd} = 1.910 \text{ MN}$$

$$\text{Maximum design shear } V_{Ed} = 334.18 \text{ kN}$$

If  $\frac{V_{Ed}}{V_{cRd}} < 1.0$  the shear resistance of the section is adequate.

$$V_{Ed} = 334.18 \text{ kN}$$

$$V_{cRd} = 1.910 \text{ MN}$$

$$\text{limit} = \frac{V_{Ed}}{V_{cRd}} = \frac{334.180 \text{ kN}}{1.910 \text{ MN}} = 1.749 \times 10^{-1}$$

The shear resistance of the section is adequate.

- **Resistance to bending**

At the point of maximum bending moment, check if the shear force will reduce the bending moment resistance of the section.

$$\text{limit} = \frac{V_{cRd}}{2} = \frac{1.910 \text{ MN}}{2} = 955.213 \text{ kN}$$

Shear force at maximum bending moment  $V_{Ed} = 334.180 \text{ kN} < V_{cRd} = 955.213 \text{ kN}$

Therefore, no reduction in resistance to bending due to shear is required.

$$M_{cRd} = M_{plRd} = \frac{W_{ply} \cdot f_y}{\gamma_{M0}} = \frac{5350000.000 \text{ mm}^3 \cdot 355.000 \text{ MPa}}{1.0} = 1.899 \text{ MN} \cdot \text{m}$$

The design bending moment is  $M_{Ed} = 496.94 \text{ kNm}$

$$M_{Ed} = 496.94 \cdot \text{kN} \cdot \text{m} = 496.94 \cdot \text{kN} \cdot 1.000 \text{ m} = 496.94 \text{ kN} \cdot \text{m}$$

$$\frac{M_{Ed}}{M_{cRd}} = \frac{496.94 \text{ kN} \cdot \text{m}}{1.899 \text{ MN} \cdot \text{m}} = 2.617 \times 10^{-1}$$

$$\frac{M_{Ed}}{M_{cRd}} < 1.0$$

Therefore, the bending moment capacity is adequate.

- **Resistance to lateral torsional buckling**

$$UV = 0.9$$

$$\beta_w = 1$$

$$C_1 = 3.75$$

$$\bar{\lambda}_{LT0} = 0.4$$

$$\bar{\lambda}_{LT} = 4.01 \times 10^{-1}$$

$$\phi_{LT} = 0.5 \cdot \left( 1 + \alpha \cdot \left( (\bar{\lambda}_{LT} - \bar{\lambda}_{LT0}) + (\beta \cdot (\bar{\lambda}_{LT}))^2 \right) \right)$$

$$\phi_{LT} = 0.5 \cdot \left( 1 + 0.49 \cdot \left( (4.01 \times 10^{-1} - 0.4) + (0.75 \cdot (4.01 \times 10^{-1})^2) \right) \right)$$

$$\phi_{LT} = 5.298 \times 10^{-1}$$

$$\chi_{LT} = \frac{1}{\phi_{LT} + ((\phi_{LT})^2 - (\beta \cdot (\bar{\lambda}_T)^2))^{\frac{1}{2}}}$$

$$\chi_{LT} = \frac{1}{5.298 \times 10^{-1} + ((5.298 \times 10^{-1})^2 - (0.75 \cdot (4.01 \times 10^{-1})^2))^{\frac{1}{2}}}$$

$$\chi_{LT} = 1.0$$

$$M_{bRd} = \chi_{LT} \cdot W_{el} \cdot \frac{f_y}{\gamma_{M1}} = 1 \cdot 4787000.000 \text{ mm}^3 \cdot \frac{355.000 \text{ MPa}}{1.0} = 1.699 \text{ MN} \cdot \text{m}$$

$$\frac{M_{Ed}}{M_{bRd}} = \frac{496.94 \text{ kN} \cdot \text{m}}{1.699 \text{ MN} \cdot \text{m}} = 2.924 \times 10^{-1}$$

$$\frac{M_{Ed}}{M_{bRd}} < 1.0$$

Therefore, the lateral torsional buckling moment capacity is adequate.

The chosen beam HEA600 satisfy the section requirements.

### 2.2.3. Secondary beam design

The importance of structural integrity during the construction stage is significant. In order to provide that secondary beam is used which its top flange fully restrained laterally by a precast slab.

L=9.000 m (for half span)

$$g_f = 2.43 \cdot \frac{\text{kN}}{(\text{m})^2} \cdot L = 2.43 \cdot \frac{\text{kN}}{(\text{m})^2} \cdot 9.000 \text{ m}$$

$$g_f = 21.870 \frac{\text{kN}}{\text{m}} \text{ (Floor dead load)}$$

$$m_b = 960.000 \text{ N/m (SelfWeight, HEA 320(S355), Assumption)}$$

$$g = 22.830 \frac{\text{kN}}{\text{m}} \text{ (Total Dead Load)}$$

$$q_f = 22.500 \frac{\text{kN}}{\text{m}} \text{ (Floor Live load)}$$

$$q = 22.500 \frac{\text{kN}}{\text{m}} \text{ (Total Live Load)}$$

$$w_{uls} = g \cdot 1.35 + q \cdot 1.5 = 22.830 \frac{\text{kN}}{\text{m}} \cdot 1.35 + 22.500 \frac{\text{kN}}{\text{m}} \cdot 1.5$$

$$w_{uls} = 64.570 \frac{\text{kN}}{\text{m}} \text{ (Ultimate Limit State)}$$

$$w_{sls} = g + q = 22.830 \frac{kN}{m} + 22.500 \frac{kN}{m}$$

$$w_{sls} = 45.330 \frac{kN}{m} \text{ (Serviceability Limit State)}$$

### Section properties of chosen beam, HEA320

$$h = 310.000 \text{ mm (depth)} \quad b = 300.000 \text{ mm (width)}$$

$$t_w = 9.000 \text{ mm (thickness)}$$

$$t_f = 15.500 \text{ mm (flange thickness)} \quad r = 27.000 \text{ mm (root radius)}$$

$$m_w = 97.600 \text{ kg} \cdot \text{m}^{-1} \text{ (weight)}$$

$$d = 225.000 \text{ mm (depth between fillets)}$$

$$I_y = 229300000.000 \text{ mm}^4 \text{ (second moment of area)}$$

$$i_y = 135.800 \text{ mm (radius of gyration)}$$

$$I_z = 69850000.000 \text{ mm}^4 \text{ (second moment of area)}$$

$$i_z = 74.900 \text{ mm (radius of gyration)}$$

$$Wpl_y = 1628000.000 \text{ mm}^3 \text{ (plastic section modulus)}$$

$$Wpl_z = 709700.000 \text{ mm}^3 \text{ (plastic section modulus)}$$

$$Wel_y = 1479000.000 \text{ mm}^3 \text{ (elastic section modulus)}$$

$$Wel_z = 465700.000 \text{ mm}^3 \text{ (elastic section modulus)}$$

$$A = 12437.000 \text{ mm}^2 \text{ (area)}$$

$$E = 210.000 \text{ MPa (elasticity modulus)}$$

$$f_y = 355.000 \text{ MPa (yield strength)}$$

$$f_u = 490.000 \text{ MPa (ultimate strength)}$$

$$\gamma_{M0} = 1.0 \text{ (partial safety factor)}$$

$$\gamma_{M1} = 1.0 \text{ (partial safety factor)}$$

### Cross-section classification

$$\varepsilon = \left( \frac{235 \cdot \text{MPa}}{f_y} \right)^{\left(\frac{1}{2}\right)}$$

$$\varepsilon = \left( \frac{235 \cdot \text{MPa}}{355.000 \text{ MPa}} \right)^{\left(\frac{1}{2}\right)}$$

$$\varepsilon = 8.136 \times 10^{-1}$$

#### • Outstand of compression flange

$$c = \frac{b - t_w - 2 \cdot r}{2} = \frac{300.000 \text{ mm} - 9.000 \text{ mm} - 2 \cdot 27.000 \text{ mm}}{2} = 116.500 \text{ mm}$$

$$\frac{c}{t_f} = \frac{116.500 \text{ mm}}{15.500 \text{ mm}} = 7.645$$

The limiting value for Class 1 is  $\frac{c}{t_f} \leq 9\epsilon = 7.645$

The flange in compression is Class 1

- **Web subject to bending**

$$c_b = 225.000 \text{ mm}$$

$$\frac{c}{t_w} = \frac{118.500 \text{ mm}}{9.000 \text{ mm}} = 13.167$$

The limiting value for Class 1 is  $\frac{c}{t_w} \leq 72\epsilon = 62.8992$

The web under bending is Class 1

### Cross-sectional resistance

- **Shear buckling**

The shear buckling resistance for webs should be verified according to section 5 EN 1993-1-5 [4] if:

$$\frac{h_w}{t_w} > 72 \cdot \frac{\epsilon}{\eta}$$

$$\eta = 1$$

$$h_w = h - 2 \cdot t_f = 310.000 \text{ mm} - 2 \cdot 15.500 \text{ mm} = 279.000 \text{ mm}$$

$$\frac{h_w}{t_w} = \frac{279.000 \text{ mm}}{9.000 \text{ mm}} = 31.0$$

$$72 \cdot \frac{\epsilon}{\eta} = 72 \cdot \frac{8.136 \times 10^{-1}}{1} = 58.58$$

The shear buckling resistance of the web does not need to be verified.

- **Shear resistance**

$$\begin{aligned} A_v &= A - (2 \cdot b \cdot t_f) + t_f \cdot (2 \cdot r + t_w) \\ &= 12437.000 \text{ mm}^2 - (2 \cdot 300.00 \text{ mm} \cdot 15.500 \text{ mm}) + 15.500 \text{ mm} (9.00 \text{ mm} \\ &\quad + 2 \cdot 27.000 \text{ mm}) = 4113.500 \text{ mm}^2 \end{aligned}$$

$$V_{plRd} = \frac{A_v \cdot \left( \frac{f_y}{(3)^{\left(\frac{1}{2}\right)}} \right)}{\gamma_{M0}}$$

$$V_{plRd} = \frac{4113.500 \text{ mm}^2 \cdot \left( \frac{355.000 \text{ MPa}}{(3)^{\left(\frac{1}{2}\right)}} \right)}{1.0}$$

$$V_{plRd} = 843.100 \text{ kN}$$

Maximum design shear  $V_{Ed} = 195 \text{ kN}$

If  $\frac{V_{Ed}}{V_{cRd}} < 1.0$  the shear resistance of the section is adequate.

$$V_{Ed} = 195.000 \text{ kN}$$

$$V_{cRd} = 843.100 \text{ kN}$$

$$\frac{V_{Ed}}{V_{cRd}} = \frac{195.000 \text{ kN}}{843.100 \text{ kN}} = 2.313 \times 10^{-1}$$

The shear resistance of the section is adequate.

- **Resistance to bending**

At the point of maximum bending moment, check if the shear force will reduce the bending moment resistance of the section.

$$\frac{V_{cRd}}{2} = \frac{843.100 \text{ kN}}{2} = 421.550 \text{ kN}$$

Shear force at maximum bending moment  $V_{Ed} = 195.00 \text{ kN} < V_{cRd} = 843.100 \text{ kN}$

Therefore, no reduction in resistance to bending due to shear is required.

$$M_{cRd} = M_{plRd} = \frac{W_{ply} \cdot f_y}{\gamma_{M0}} = \frac{1628000.000 \text{ mm}^3 \cdot 355.000 \text{ MPa}}{1.0} = 577.940 \text{ kN} \cdot \text{m}$$

The design bending moment is  $M_{Ed} = 496.94 \text{ kNm}$

$$M_{Ed} = 496.94 \cdot \text{kN} \cdot \text{m}$$

$$limit = \frac{M_{Ed}}{M_{cRd}} = \frac{496.94 \text{ kN} \cdot \text{m}}{577.940 \text{ kN} \cdot \text{m}} = 8.598 \times 10^{-1}$$

$$\frac{M_{Ed}}{M_{cRd}} < 1.0$$

Therefore, the bending moment capacity is adequate.

The chosen beam HEA320 satisfy the section requirements.

## 2.2.4. Preliminary Column Design

### 2.2.4.1. Actions-Loading

The design load area shown in Figure 3.9 presents loading area of expected critical section.

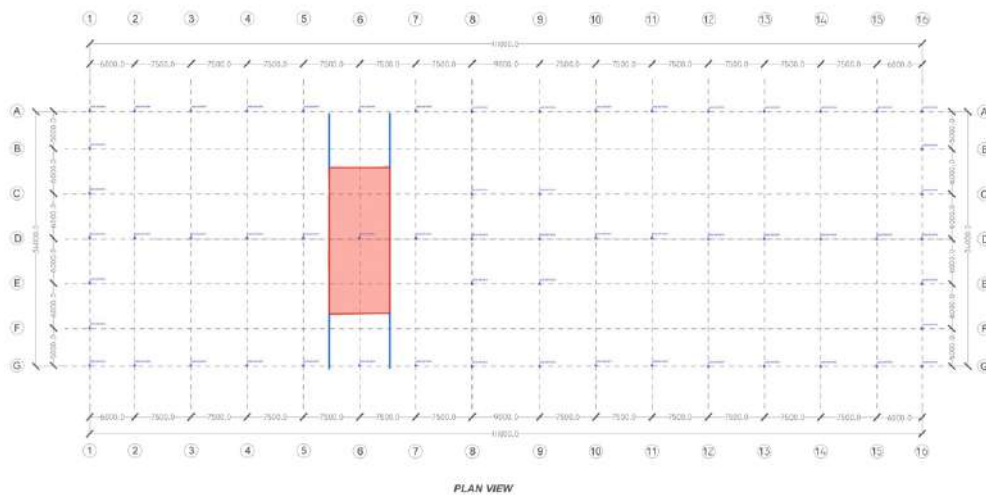


Figure 2.2.4.1. 1: Loading area

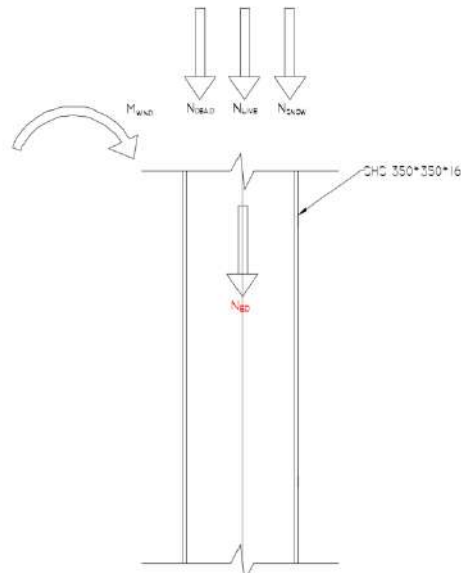


Figure 2.2.4.1. 2: Actions

### Design value of force for ULS

$$\begin{aligned}
 g_k &= 2.430 \text{ kPa} & q_k &= 2.500 \text{ kPa} & g_{sk} &= 700.000 \text{ Pa} \\
 q_w &= 320.000 \text{ Pa} & B &= 7.500 \text{ m} & H &= 8.500 \text{ m} \\
 n_{floor} &= 3 & L_{Total} &= 34.000 \text{ m} & H_{Total} &= 9.500 \text{ m} \\
 g_{steel} &= 785.000 \text{ N/m}^3 & A_{col} &= 7524.000 \text{ mm}^2 \\
 \gamma_G &= 1.35 & \gamma_Q &= 1.5 & \gamma_{M0} &= 1.0 & \psi_{0_{snow}} &= 0.5 \\
 \psi_{0_{wind}} &= 0.6 & b_c &= 3500.000 \text{ mm} \text{ (SHS 350 * 350 * 16)}
 \end{aligned}$$

$$A_{load} = B \cdot H \cdot 2 = 7.500 \text{ m} \cdot 8.500 \text{ m} \cdot 2 = 127.500 \text{ m}^2 \text{ (Load Area)}$$

$$N_{self} = A_{col} \cdot g_{steel} \cdot H_{Total} = 7524.000 \text{ mm}^2 \cdot 785.000 \frac{\text{N}}{\text{m}^3} \cdot 12.600 \text{ m}$$

$$N_{self} = 74.420 \text{ N (Column self weight)}$$

$$N_{dead} = A_{load} \cdot g_k \cdot n_{floor} = 127.500 \text{ m}^2 \cdot 2.430 \text{ kPa} \cdot 3$$

$$N_{dead} = 929.475 \text{ kN (Dead Load)}$$

$$N_{live} = A_{load} \cdot q_k \cdot n_{floor} = 127.500 \text{ m}^2 \cdot 2.500 \text{ kPa} \cdot 3$$

$$N_{live} = 956.250 \text{ kN (Live Load)}$$

$$N_{snow} = A_{load} \cdot g_{sk} = 127.500 \text{ m}^2 \cdot 700.000 \text{ Pa}$$

$$N_{snow} = 89.250 \text{ kN (Snow Load)}$$

$$M_{wind} = q_w \cdot \left( \frac{L_{total}}{2} \right) \cdot \left( \frac{(H_{Total})^2}{2} \right)$$

$$M_{wind} = 320.000 \text{ Pa} \cdot \left( \frac{34.000 \text{ m}}{2} \right) \cdot \left( \frac{(12.600 \text{ m})^2}{2} \right)$$

$$M_{wind} = 431.827 \text{ kN} \cdot \text{m (Bending moment caused by wind load)}$$

$$N_{wind} = \frac{M_{wind}}{B} = \frac{431.827 \text{ kN} \cdot \text{m}}{7.500 \text{ m}}$$

$$N_{wind} = 57.577 \text{ kN (Tension Force)}$$

$$N_{ed} = \gamma_G \cdot (N_{self} + N_{dead}) + \gamma_Q \cdot (N_{live} + N_{snow} \cdot \psi_{0_{snow}} + N_{wind} \cdot \psi_{0_{wind}})$$

$$N_{ed} = 1.35 \cdot (74.420 \text{ N} + 929.475 \text{ kN}) + 1.5 \cdot (956.250 \text{ kN} + 89.250 \text{ kN} \cdot 0.5 + \dots$$

$$\dots + 57.577 \text{ kN} \cdot 0.6)$$

$$N_{ed} = 2.808 \text{ MN (Design Load)}$$

#### 2.2.4.2. Column Design

##### Section properties of chosen section which is SHS 350\*350\*16

$$h = 350.000 \text{ mm (width)}$$

$$b = 350.000 \text{ mm (side dimension)}$$

$$t = 16.000 \text{ mm (thickness)}$$

$$L = 3.500 \text{ m (height)}$$

$$r_0 = 24.000 \text{ mm (outer rounding radius)}$$

$$r_i = 16.000 \text{ mm (inner rounding radius)}$$

$$A = 21101.000 \text{ mm}^2 \text{ (area)}$$

$$A_v = 10551.000 \text{ mm}^2 \text{ (shear area)}$$

$$I = 389400000.000 \text{ mm}^4 \text{ (second moment of inertia)}$$

$$W_{el} = 2225000.000 \text{ mm}^3 \text{ (elastic section modulus)}$$

$$W_{pl} = 2630000.000 \text{ mm}^3 \text{ (plastic section modulus)}$$



$$N_{plRd} = 7.491 \text{ MN (design plastic axial force resistance)}$$

$$V_{plRd} = 2.162 \text{ MN (design plastic shear axial force resistance)}$$

$$M_{elRd} = 789.970 \text{ kN.m (design elastic bending moment resistance)}$$

$$M_{plRd} = 933.530 \text{ kN.m (design plastic bending moment resistance)}$$

$$T_{plRd} = 668.910 \text{ kN (design plastic torsional moment resistance)}$$

$$i = 117.000 \text{ mm (radius of gyration)}$$

$$y_0 = 0$$

$$W_T = 3264000.000 \text{ mm}^3 \text{ (torsion modulus)}$$

$$I_T = 609900000.000 \text{ mm}^4 \text{ (torsion constant)}$$

$$f_y = 355.000 \text{ MPa (yield strength)}$$

$$f_u = 490.000 \text{ MPa (ultimate strength)}$$

$$\gamma_{M0} = 1.0 \text{ (resistance of cross – section)}$$

$$\gamma_{M1} = 1.0 \text{ (resistance of members to instability)}$$

$$G = 81.000 \text{ GPa (shear modulus)}$$

$$\alpha = 0.21 \text{ (buckling curve)}$$

$$\pi = 3.142 \text{ (\pi pi constant)}$$

$$E = 210.000 \text{ GPa (modulus of elasticity)}$$

### Cross-sectional resistance

Compression resistance should verify  $N_{ed}/N_{c,Rd} \leq 1.0$

The design resistance of the cross-section for uniform compression is;

$$N_{cRd} = A \cdot \frac{f_y}{\gamma_{M0}} = 21101.000 \text{ mm}^2 \cdot \frac{355.000 \text{ MPa}}{1.0} = 7.491 \text{ MN (for class 1)}$$

$$\frac{N_{Ed}}{N_{cRd}} = \frac{2.786 \text{ MN}}{7.491 \text{ MN}} = 3.719 \times 10^{-1}$$

$$\frac{N_{Ed}}{N_{cRd}} < 1.00;$$

Therefore, the compression resistance of the cross-section is adequate.

### Member buckling resistance

The design buckling resistance is determined from:

$$L_{cr} = 3.500 \text{ m}$$

$$\varepsilon = 8.136 \times 10^{-1}$$

$$\lambda_1 = 76.399$$

$$\bar{\lambda} = \left( \frac{L_{cr}}{i} \right) \cdot \left( \frac{1}{\lambda_1} \right) = \left( \frac{3.500 \text{ m}}{117.000 \text{ mm}} \right) \cdot \left( \frac{1}{76.399} \right) = 3.916 \times 10^{-1}$$

$$\phi = 0.5 \cdot (1 + \alpha \cdot (\bar{\lambda} - 0.2)) + (\bar{\lambda})^2$$

$$\phi = 0.5 \cdot (1 + 0.21 \cdot (3.916 \times 10^{-1} - 0.2)) + (3.916 \times 10^{-1})^2$$

$$\phi = 5.968 \times 10^{-1}$$

$$\chi = \frac{1}{\phi + ((\phi)^2 - (\bar{\lambda})^2)^{\left(\frac{1}{2}\right)}}$$

$$\chi = \frac{1}{5.968 \times 10^{-1} + ((5.968 \times 10^{-1})^2 - (3.916 \times 10^{-1})^2)^{\left(\frac{1}{2}\right)}}$$

$$\chi = 0.955$$

$$N_{bRd} = \chi \cdot A \cdot \frac{f_y}{\gamma_{M1}} = 0.955 \cdot 21101.000 \text{ mm}^2 \cdot \frac{355.000 \text{ MPa}}{1.0} = 7.154 \text{ MN}$$

$$\frac{N_{Ed}}{N_{bRd}} = \frac{2.808 \text{ MN}}{7.154 \text{ MN}} = 3.925 \times 10^{-1}$$

$$\frac{N_{Ed}}{N_{bRd}} < 1.00;$$

Therefore, the flexural buckling resistance of the section is adequate.

### **Torsional and torsional-flexural buckling resistance**

$$i_0 = 165.463 \text{ mm}$$

$$N_{c_{rT}} = 1.811 \text{ GN}$$

$$\bar{\lambda}_T = \left( \frac{A \cdot f_y}{N_{c_{rT}}} \right)^{\left(\frac{1}{2}\right)}$$

$$\bar{\lambda}_T = \left( \frac{21101.000 \text{ mm}^2 \cdot 355.000 \text{ MPa}}{1.811 \text{ GN}} \right)^{\left(\frac{1}{2}\right)}$$

$$\bar{\lambda}_T = 6.431 \times 10^{-2}$$

$$\phi_T = 0.5 \cdot (1 + \alpha \cdot ((\bar{\lambda}_T - 0.2) + (\bar{\lambda})^2))$$

$$\phi_T = 0.5 \cdot (1 + 0.21 \cdot ((6.431 \times 10^{-2} - 0.2) + (6.431 \times 10^{-2})^2))$$

$$\phi_T = 4.862 \times 10^{-1}$$

$$\chi_T = \frac{1}{\phi_T + ((\phi_T)^2 - (\bar{\lambda}_T)^2)^{\frac{1}{2}}}$$

$$\chi_T = \frac{1}{4.862 \times 10^{-1} + ((4.862 \times 10^{-1})^2 - (6.431 \times 10^{-2})^2)^{\frac{1}{2}}}$$

$$\chi_T = 1.033$$

$$\chi_T = 1.0$$

$$N_{bTRd} = \frac{\chi_T \cdot A \cdot f_y}{\gamma_{M1}} = \frac{1.0 \cdot 21101.000 \text{ mm}^2 \cdot 355.000 \text{ MPa}}{1.0} = 7.491 \text{ MN}$$

$$\frac{N_{Ed}}{N_{bTRd}} < 1.00;$$

Therefore, the torsional buckling resistance of the section is adequate.

### Lateral torsional buckling resistance

Connection eccentricity causes a design major axis bending moment is shown below.

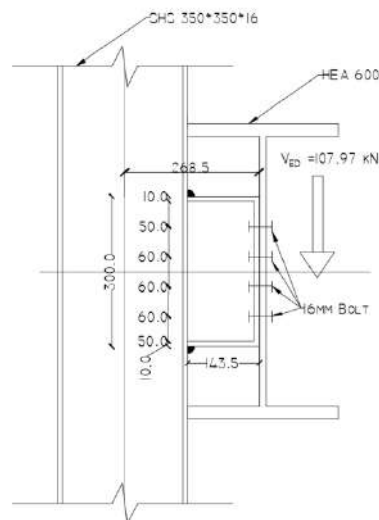


Figure 2.2.4.1. 3: Bending moment due to connection

$$e = 268.500 \text{ mm}$$

$$V_{Ed_6} = 107.97 \text{ kN}$$

$$M_{Ed} = M_{Ed_6} = V_{Ed_6} \cdot e = 107.97 \text{ kN} \cdot 268.500 \text{ mm} = 28.990 \text{ kN} \cdot \text{m}$$

$$UV = 0.9$$

$$\beta_w = 1$$

$$C_1 = 1$$

$$\beta = 0.75$$

$$\bar{\lambda}_{LT0} = 0.4$$

$$\bar{\lambda}_{LT} = 3.524 \times 10^{-1}$$

The slenderness for lateral torsional buckling is less than  $\bar{\lambda}_{LT0}$  the effect of lateral torsional buckling is neglected. [3]

$$\phi_{LT} = 0.5 \cdot \left( 1 + \alpha \cdot \left( (\bar{\lambda}_{LT} - \bar{\lambda}_{LT0}) + (\beta \cdot (\bar{\lambda}_{LT}))^2 \right) \right)$$

$$\phi_{LT} = 0.5 \cdot \left( 1 + 0.21 \cdot \left( (3.524 \times 10^{-1} - 0.4) + (0.75 \cdot (3.524 \times 10^{-1})^2) \right) \right)$$

$$\phi_{LT} = 5.048 \times 10^{-1}$$

$$\chi_{LT} = \frac{1}{\phi_{LT} + ((\phi_{LT})^2 - (\beta \cdot (\bar{\lambda}_{LT})^2))^{\left(\frac{1}{2}\right)}}$$

$$\chi_{LT} = \frac{1}{5.048 \times 10^{-1} + ((5.048 \times 10^{-1})^2 - (0.75 \cdot (3.524 \times 10^{-1})^2))^{\left(\frac{1}{2}\right)}}$$

$$\chi_{LT} = 1.103$$

$$\chi_{LT} = 1.0$$

$$M_{bRd} = \chi_{LT} \cdot W_{el} \cdot \frac{f_y}{\gamma_{M1}} = 1 \cdot 2225000.000 \text{ mm}^3 \cdot \frac{355.000 \text{ MPa}}{1.0} = 789.875 \text{ kN} \cdot \text{m}$$

Combined axial load plus bending

$$condition_3 = \left( \frac{N_{Ed}}{N_{bRd}} \right) + \left( \frac{M_{Ed}}{M_{bRd}} \right) = \left( \frac{2.808 \text{ MN}}{7.154 \text{ MN}} \right) + \left( \frac{28.990 \text{ kN} \cdot \text{m}}{789.875 \text{ kN} \cdot \text{m}} \right) = 4.292 \times 10^{-1}$$

$$\left( \frac{N_{Ed}}{N_{bRd}} \right) + \left( \frac{M_{Ed}}{M_{bRd}} \right) < 1.00$$

Therefore, combined axial load plus bending resistance is adequate. The chosen column section SHS 350x350x16 satisfies the section requirements.

### 2.2.5. Connection Design

Joining beam and column together is done by reverse channel connection where its web is connected to beam by bolting. The connection has eccentricity and it causes bending moment which is shown at column calculation.

**It is assumed that project specifications require class 8.8 bolts.**

$$f_{yb} = 640.000 \text{ MPa}$$

$$f_{ub} = 800.000 \text{ MPa}$$

$$f_y = 355.000 \text{ MPa}$$

$$b = 160.000 \text{ mm}$$

$$d_p = 300.000 \text{ mm}$$

$$t = 10.000 \text{ mm}$$

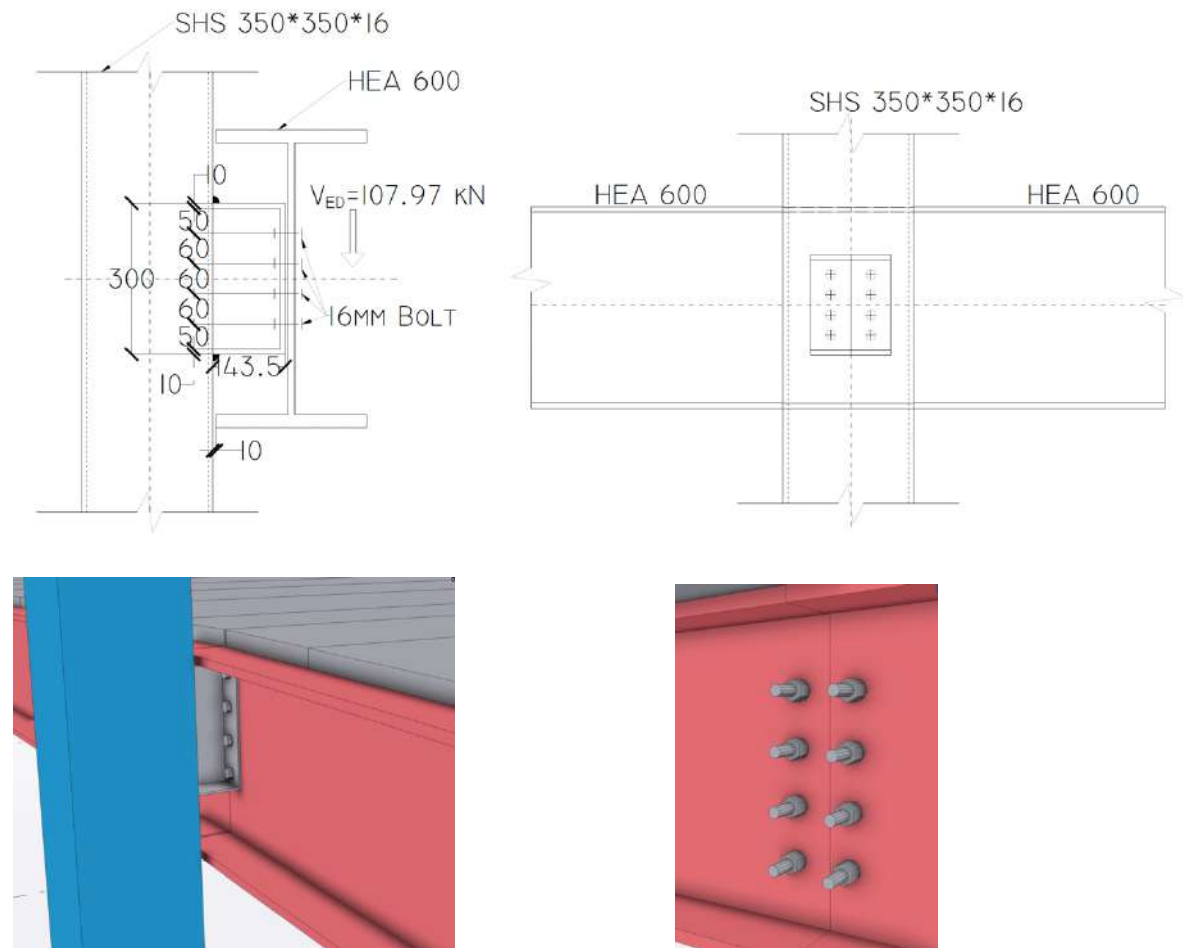


Figure 2.2.5. 1: Connection between column and beam

**Design shear resistance of one fully treated bolt is**

$$\alpha_v = 0.6$$

$$\gamma_{M2} = 1.25$$

$$A_s = 157.000 \text{ mm}^2$$

$$F_{vRd} = \frac{\alpha_v \cdot A_s \cdot f_{ub}}{\gamma_{M2}} = \frac{0.6 \cdot 157.000 \text{ mm}^2 \cdot 800.000 \text{ MPa}}{1.25} = 60.288 \text{ kN}$$

**The design bearing resistance of bolts is evaluated by factors  $k_1$ ,  $\alpha_b$  as governed by the plate**

$$e_2 = 80.000 \text{ mm}$$

$$d_0 = 18.000 \text{ mm}$$

$$k_1 = \min\left(\left(\left(2.8 \cdot \frac{e_2}{d_0}\right) - 1.7\right), 2.5\right) = \min\left(\left(\left(2.8 \cdot \frac{80.000 \text{ mm}}{18.000 \text{ mm}}\right) - 1.7\right), 2.5\right) = 2.5$$

$$e_1 = 50.000 \text{ mm}$$

$$p_1 = 60.000 \text{ mm}$$

$$f_u = 360.000 \text{ MPa}$$

$$\alpha_b = \min\left(\frac{e_1}{3 \cdot d_0}, \left(\frac{p_1}{3 \cdot d_0} - 0.25\right), \left(\frac{f_{ub}}{f_u}\right), 1\right)$$

$$\alpha_b = \min\left(\frac{50.000 \text{ mm}}{3 \cdot 18.000 \text{ mm}}, \left(\frac{60.000 \text{ mm}}{3 \cdot 18.000 \text{ mm}} - 0.25\right), \left(\frac{800.000 \text{ MPa}}{360.000 \text{ MPa}}\right), 1\right)$$

$$\alpha_b = 8.611 \times 10^{-1}$$

Analogous to the bearing resistance of the bolts, the bearing resistance of the beam web may be calculated by factors  $k_1$  and  $\alpha_b$  governed by the plate as follows:

$$e_{1_1} = 170.000 \text{ mm}$$

$$p_1 = 60.000 \text{ mm}$$

$$f_u = 360.000 \text{ MPa}$$

$$\alpha_b = \min\left(\frac{e_{1_1}}{3 \cdot d_0}, \left(\frac{p_1}{3 \cdot d_0} - 0.25\right), \left(\frac{f_{ub}}{f_u}\right), 1\right)$$

$$\alpha_b = \min\left(\frac{170.000 \text{ mm}}{3 \cdot 18.000 \text{ mm}}, \left(\frac{60.000 \text{ mm}}{3 \cdot 18.000 \text{ mm}} - 0.25\right), \left(\frac{800.000 \text{ MPa}}{360.000 \text{ MPa}}\right), 1\right)$$

$$\alpha_b = 8.611 \times 10^{-1}$$

**The design bearing resistance of one bolt is**

$$d = 16.000 \text{ mm}$$

$$t = 10.000 \text{ mm}$$

$$F_{bRd} = k_1 \cdot \alpha_b \cdot d \cdot t \cdot \frac{f_u}{\gamma_{M2}} = 2.5 \cdot 8.611 \times 10^{-1} \cdot 16.000 \text{ mm} \cdot 10.000 \text{ mm} \cdot \frac{360.000 \text{ MPa}}{1.25}$$

$$F_{bRd} = 99.200 \text{ kN}$$

**The design bearing resistance of four bolts are;**

$$V_{rd} = 4 \cdot F_{bRd} = 4 \cdot 99.200 \text{ kN} = 396.800 \text{ kN}$$

$$V_{rd} = 396.8 \text{ kN} > V_{majorEd} = 107.97 \text{ kN}$$

**The connection resistance is satisfactory.**

Eccentricity of acting shear force in bolts creates a bending moment in welds:

The bolts group must resist the following bending moment coming from eccentricity.

$$V_{Ed} = 107.970 \text{ kN}$$

$$e_{\text{boltgroup}} = 268.500 \text{ mm}$$

$$M_{ed} = e_{\text{boltgroup}} \cdot V_{Ed} = 268.500 \text{ mm} \cdot 107.970 \text{ kN} = 28.990 \text{ kN} \cdot \text{m}$$

The moments introduce stress  $\sigma_w$  is plane of the plate:

$$a = 6.000 \text{ mm}$$

$$l = 300.000 \text{ mm}$$

$$W_{elw} = \left( \frac{2 \cdot a \cdot (l)^2}{6} \right) = \left( \frac{2 \times 6.000 \text{ mm} \cdot (300.000 \text{ mm})^2}{6} \right) = 180000.000 \text{ mm}^3$$

$$\sigma_w = \frac{M_{ed}}{W_{elw}} = \frac{28.990 \text{ kN} \cdot \text{m}}{180000.000 \text{ mm}^3} = 161.055 \text{ MPa}$$

This stress is resolved into stress perpendicular and parallel to axes of weld throat calculated as follows:

$$\tau_{\perp} = \sigma_{\perp} = \frac{\sigma_w}{(2)^{1/2}} = \frac{161.055 \text{ MPa}}{(2)^{1/2}} = 113.883 \text{ MPa}$$

$$\tau_{//} = \frac{V_{Ed}}{2 \cdot a \cdot l} = \frac{107.970 \text{ kN}}{2 \times 6.0 \text{ mm} \times 300 \text{ mm}} = 29.992 \text{ MPa}$$

The design resistance of the fillet weld is sufficient if the following are satisfied.

$$\beta_w = 0.9$$

$$C_1 = ((\sigma_L)^2 + 3 \cdot ((\tau_L)^2 - (\tau_{//})^2))^{(1/2)}$$

$$C_1 = ((113.833 \text{ MPa})^2 + 3 \cdot ((113.833 \text{ MPa})^2 - (29.992 \text{ MPa})^2))^{(1/2)}$$

$$C_1 = 233.615 \text{ MPa}$$

$$C_2 = \frac{f_u}{\beta_w \cdot \gamma_{M2}} = \frac{360 \text{ MPa}}{0.9 \times 1.25} = 320 \text{ MPa}$$

$$\sigma_L = 113.883 \text{ MPa}$$

$$C_3 = \frac{f_u}{\gamma_{M2}} = \frac{360 \text{ MPa}}{1.25} = 288 \text{ MPa}$$

$$C_1 = 233.615 \text{ MPa} < C_2 = 320.00 \text{ MPa} \text{ and } \sigma_L = 113.883 \text{ MPa} < C_3 = 288.00 \text{ MPa}$$

The weld is satisfactory.

The design block tearing resistance of a plate is given by summing the resistance on critical area in tension and that in shear.

$$A_{nt} = t. \left( e_2 - \frac{d_0}{2} \right) = 10.00 \text{ mm} . \left( 80.00 \text{ mm} - \frac{18.00 \text{ mm}}{2} \right) = 710.00 \text{ mm}^2$$

$$A_{nv} = t. \left( e_2 + p_1 - d_0 - \frac{d_0}{2} \right) = 10 \text{ mm} . \left( 80.00 \text{ mm} + 60.00 \text{ mm} - 18 \text{ mm} - \frac{18 \text{ mm}}{2} \right) = 1130.00 \text{ mm}^2$$

$$\gamma_{M0} = 1.0$$

$$V_{eff2Rd} = \left( \frac{0.5 \cdot A_{nt} \cdot f_u}{\gamma_{M2}} \right) + \left( \frac{0.5 \cdot A_{nv} \cdot f_y}{(3)^{\left(\frac{1}{2}\right)} \cdot \gamma_{M0}} \right)$$

$$= \left( \frac{0.5 \cdot 710.00 \text{ mm}^2 \cdot 360.00 \text{ MPa}}{1.25} \right) + \left( \frac{1130.00 \text{ mm}^2 \cdot 355 \text{ MPa}}{(3)^{\left(\frac{1}{2}\right)} \cdot 1.0} \right)$$

$$= 333.844 \text{ kN}$$

$V_{eff2Rd} = 333.844 \text{ kN} > V_{Ed} = 107.97 \text{ kN}$  , The design block tearing resistance is satisfactory.

**The shear resistance of gross area:**

$$A_v = 3000.0 \text{ mm}^2$$

$$V_{plRd} = \left( \frac{A_v \cdot f_y}{(3)^{\left(\frac{1}{2}\right)} \cdot \gamma_{M0}} \right) = \frac{3000.00 \text{ mm}^2 \cdot 355 \text{ MPa}}{(3)^{\left(\frac{1}{2}\right)} \cdot 1.0} = 614.878 \text{ kN}$$

$V_{plRd} = 614.878 \text{ kN} > V_{Ed} = 107.97 \text{ kN}$  , The shear resistance of gross area is satisfactory.

For the beam web the design block shear resistance is evaluated in similar way as for the plate.

$$t_w = 13.00 \text{ mm}$$



$$A_{ntw} = t_w \cdot \left( e_2 - \frac{d_0}{2} \right) = 13.00 \text{ mm} \cdot \left( 80.00 \text{ mm} - \frac{18.00 \text{ mm}}{2} \right) = 923.00 \text{ mm}^2$$

$$\begin{aligned} A_{nvw} &= t_w \cdot \left( 120 \text{ mm} + e_1 + p_1 - d_0 - \frac{d_0}{2} \right) \\ &= 13.00 \text{ mm} \cdot \left( 120.00 \text{ mm} + 50.00 \text{ mm} + 60.00 \text{ mm} - 18 \text{ mm} - \frac{18 \text{ mm}}{2} \right) \\ &= 2639.00 \text{ mm}^2 \end{aligned}$$

**The beam shear resistance:**

$$\begin{aligned} V_{eff2Rd} &= \left( \frac{0.5 \cdot A_{ntw} \cdot f_u}{\gamma_{M2}} \right) + \left( \frac{0.5 \cdot A_{nvw} \cdot f_y}{(3)^{\left(\frac{1}{2}\right)} \cdot \gamma_{M0}} \right) \\ &= \left( \frac{0.5 \cdot 923.00 \text{ mm}^2 \cdot 360.00 \text{ MPa}}{1.25} \right) + \left( \frac{2639.00 \text{ mm}^2 \cdot 355 \text{ MPa}}{(3)^{\left(\frac{1}{2}\right)} \cdot 1.0} \right) \\ &= 673.800 \text{ kN} \end{aligned}$$

$V_{eff2Rd} = 673.800 \text{ kN} > V_{Ed} = 107.97 \text{ kN}$  , The beam shear resistance is satisfactory.

The bending resistance is checked for class 3 cross-section which resistance is;

$$M_{elRd} = \left( \frac{W_{elw} \cdot f_y}{\gamma_{M0}} \right) = \left( \frac{180000 \text{ mm}^3 \cdot 355.00 \text{ MPa}}{1.0} \right) = 63.900 \text{ kNm}$$

$M_{elRd} = 63.900 \text{ kNm} > M_{Ed} = 28.99 \text{ kNm}$  ;

Therefore, the connection is satisfactory.

### 2.2.6. Bracing Design

The robustness and stability are a critical structural design consideration of a structure. Multi-storey car park has that problem because it does not have internal walls which help stability. In order to provide stability, bracing member is placed at suitable bays.

**Section properties of chosen section which is SHS 110\*110\*14.2 (S275 JR)**

$b = 120.000 \text{ mm}$  (side dimension)                       $L = 8.276 \text{ m}$ (height)

$A = 5790.000 \text{ mm}^2$  (area)

$f_y = 275.000 \text{ MPa}$      $t = 14.200 \text{ mm}$

$I_y = 10500000.000 \text{ mm}^4$  (second moment of inertia, y – y axis)

$I_z = 10500000.000 \text{ mm}^4$  (second moment of inertia, z – z axis)

$\gamma_{M0} = 1.0$  (resistance of cross – section)

$\gamma_{M1} = 1.0$  (resistance of members to instability)

$\alpha = 0.21$  (buckling curve)

$\pi = 3.142$  ( $\pi$  pi constant)

$E = 210.000$  GPa (modulus of elasticity)

### Cross-sectional resistance

$N_{Ed} = 259.220$  kN

$$N_{cRd} = \left( \frac{A_s \cdot f_y}{\gamma_{M0}} \right) = \left( \frac{5790.00 \text{ mm}^2 \cdot 275.00 \text{ MPa}}{1.0} \right) = 1.592 \text{ MN}$$

$$\left( \frac{N_{Ed}}{N_{cRd}} \right) = \left( \frac{259.220 \text{ kN}}{1.592 \text{ MN}} \right) = 1.628 \times 10^{-1}$$

$\left( \frac{N_{Ed}}{N_{cRd}} \right) < 1.0$ , Resistance in tension is adequate.

### Buckling resistance

$L_{cr} = 8.276$  m

$$N_{cr} = \left( \frac{(\pi)^2 \cdot E \cdot I_z}{(L_{cr})^2} \right) = \frac{(3.142)^2 \cdot 210.000 \text{ GPa} \cdot 10500000.000 \text{ mm}^4}{(8.276 \text{ m})^2} = 317.737 \text{ kN}$$

$$\bar{\lambda}_T = \left( \frac{A \cdot f_y}{N_{cr}} \right)^{\left(\frac{1}{2}\right)}$$

$$\bar{\lambda}_T = \left( \frac{5790.000 \text{ mm}^2 \cdot 275.000 \text{ MPa}}{317.737 \text{ kN}} \right)^{\left(\frac{1}{2}\right)} = 2.239$$

$$\phi = 0.5 \cdot \left( 1 + \alpha \cdot \left( (\bar{\lambda} - 0.2) + (\bar{\lambda})^2 \right) \right)$$

$$\phi = 0.5 \cdot \left( 1 + 0.21 \cdot \left( (2.239 - 0.2) + (2.239)^2 \right) \right)$$

$$\phi = 3.22$$

$$\chi = \frac{1}{\phi + \left( (\phi)^2 - (\bar{\lambda})^2 \right)^{\left(\frac{1}{2}\right)}}$$

$$\chi = \frac{1}{3.22 + \left( (3.22)^2 - (2.239)^2 \right)^{\left(\frac{1}{2}\right)}}$$

$$\chi = 1.807 \times 10^{-1}$$

$$N_{bRd} = \chi \cdot A \cdot \frac{f_y}{\gamma_{M1}} = 1.807 \times 10^{-1} \cdot 5790.000 \text{ mm}^2 \cdot \frac{275.000 \text{ MPa}}{1.0} = 287.734 \text{ kN}$$

$$\left(\frac{N_{Ed}}{N_{bRd}}\right)=\left(\frac{259.220\text{ kN}}{287.734\text{ kN}}\right)=9.009 \times 10^{-1}$$

$\left(\frac{N_{Ed}}{N_{cRd}}\right) < 1.0$ , Buckling resistance is adequate.

Therefore, chosen bracing member 110x110x14.2 is satisfied requirements.

### 2.2.7. Column Base Design

Column is subjected to the self-weight, dead load, snow load and live load with loading area defined in figure. Calculated column is part of bracing system-there are additional forces due to acting in bracing system.

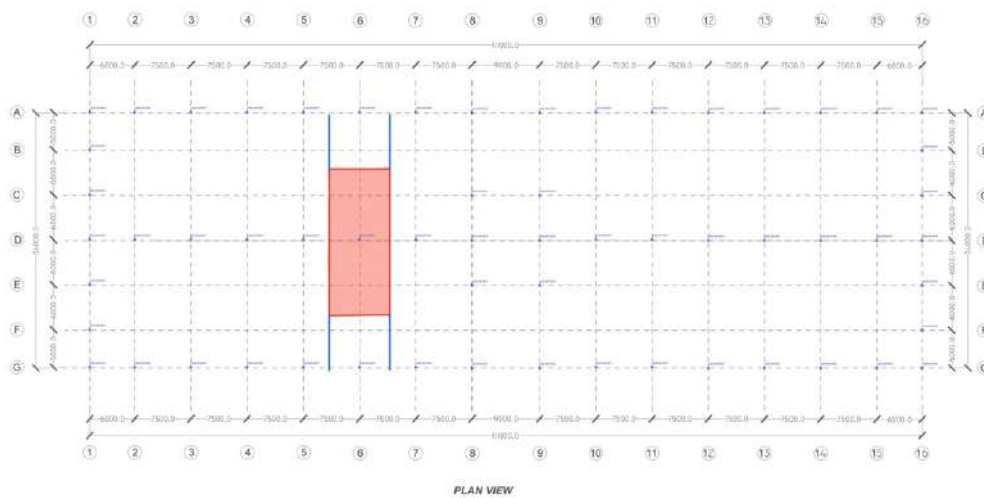


Figure 2.2.7. 1: Calculated column load area

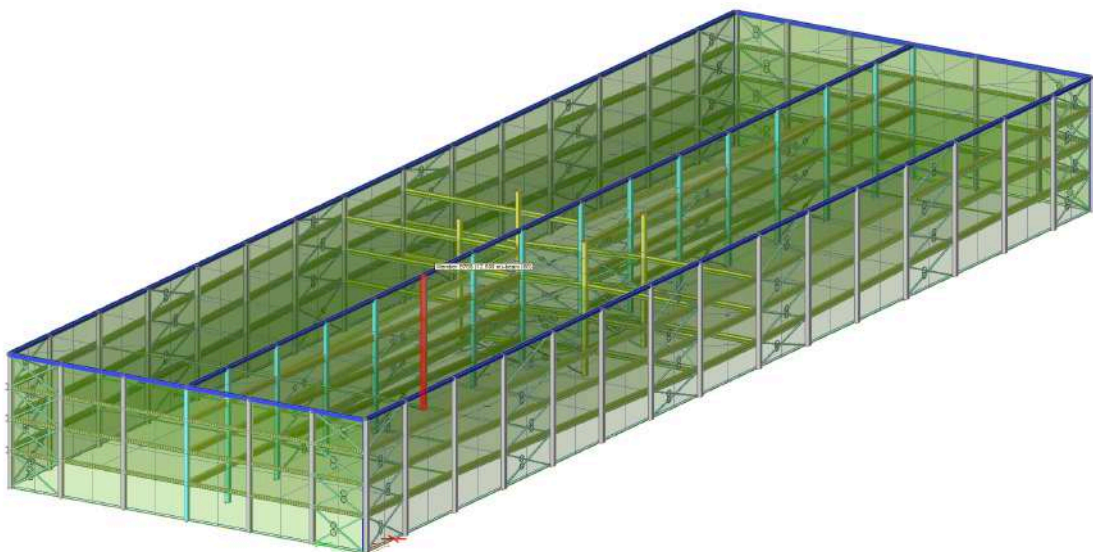


Figure 2.2.7. 2: Calculated column

$$\begin{array}{lll}
g_k = 2.430 \text{ kPa} & q_k = 2.500 \text{ kPa} & g_{sk} = 700.000 \text{ Pa} \\
q_w = 320.000 \text{ Pa} & B = 7.500 \text{ m} & H = 8.500 \text{ m} \\
n_{floor} = 3 & L_{Total} = 34.000 \text{ m} & H_{Total} = 12.600 \text{ m} \\
g_{steel} = 785.000 \text{ N/m}^3 & A_{col} = 7524.000 \text{ mm}^2 & \gamma_G = 1.35 \\
\gamma_Q = 1.5 & \gamma_{M0} = 1.0 & \psi_{0_{snow}} = 0.5 \\
\psi_{0_{wind}} = 0.6 & b_c = 350.000 \text{ mm (SHS 350 * 350 * 16)} &
\end{array}$$

### Maximum normal force in column base

#### Load Area

$$A_{load} = B \cdot H \cdot 2 = 7.500 \text{ m} \cdot 8.500 \text{ m} \cdot 2 = 127.500 \text{ m}^2 \text{ (Load Area)}$$

#### Column self-weight

$$N_{self} = A_{col} \cdot g_{steel} \cdot H_{Total} = 7524.000 \text{ mm}^2 \cdot 785.000 \frac{\text{N}}{\text{m}^3} \cdot 12.600 \text{ m}$$

$$N_{self} = 74.420 \text{ N (Column self weight)}$$

#### Dead Load

$$N_{dead} = A_{load} \cdot g_k \cdot n_{floor} = 127.500 \text{ m}^2 \cdot 2.430 \text{ kPa} \cdot 3$$

$$N_{dead} = 929.475 \text{ kN (Dead Load)}$$

#### Live Load

$$N_{live} = A_{load} \cdot q_k \cdot n_{floor} = 127.500 \text{ m}^2 \cdot 2.500 \text{ kPa} \cdot 3$$

$$N_{live} = 956.250 \text{ kN (Live Load)}$$

#### Snow Load

$$N_{snow} = A_{load} \cdot g_{sk} = 127.500 \text{ m}^2 \cdot 700.000 \text{ Pa}$$

$$N_{snow} = 89.250 \text{ kN (Snow Load)}$$

Bending moment on cantilever substituting truss bracing caused by wind load;

$$M_{wind} = q_w \cdot \left( \frac{L_{total}}{2} \right) \cdot \left( \frac{(H_{Total})^2}{2} \right)$$

$$M_{wind} = 320.000 Pa \cdot \left(\frac{34.000 m}{2}\right) \cdot \left(\frac{(12.600 m)^2}{2}\right)$$

$$M_{wind} = 431.827 kN \cdot m \quad (\text{Bending moment caused by wind load})$$

$$N_{wind} = \frac{M_{wind}}{B} = \frac{431.827 kN \cdot m}{7.500 m}$$

$$N_{wind} = 57.577 kN \quad (\text{Tension Force})$$

Total load combination

$$N_{ed} = \gamma_Q \cdot (N_{self} + N_{dead}) + \gamma_Q \cdot (N_{live} + N_{snow} \cdot \psi_{0_{snow}} + N_{wind} \cdot \psi_{0_{wind}})$$

$$N_{ed} = 1.5 \cdot (74.420 N + 929.475 kN) + 1.5 \cdot (956.250 kN + 89.250 kN \cdot 0.5 + \dots \\ \dots + 57.577 kN \cdot 0.6)$$

$$N_{ed} = 2.947 MN \quad (\text{Design Load})$$

### Column Base Geometry

$$t = 30.000 \text{ mm} \quad b_1 = 450.000 \text{ mm} \quad d_1 = 450.000 \text{ mm}$$

$$b = 500.000 \text{ mm} \quad d = 500.000 \text{ mm} \quad h = 500.000 \text{ mm}$$

$$b_r = 425.000 \text{ mm} \quad f_y = 355.000 \text{ MPa} \quad E = 210.000 \text{ GPa}$$

$$\gamma_0 = 1.0 \quad f_{ck} = 25.000 \text{ MPa} \quad \gamma_c = 1.5$$

### Column Base Check

$$b_2 = \min(b_1 + 2 \cdot b_r, 3 \cdot b_1, b_1 + h)$$

$$b_2 = \min(450.000 \text{ mm} + 2 \cdot 425.000 \text{ mm}, 3 \cdot 450.000 \text{ mm} \dots, \\ \dots, 450.000 \text{ mm} + 500.000 \text{ mm})$$

$$b_2 = 950.000 \text{ mm}$$

$$d_2 = \min(d_1 + 2 \cdot b_r, 3 \cdot d_1, d_1 + h)$$

$$d_2 = \min(450.000 \text{ mm} + 2 \cdot 425.000 \text{ mm}, 3 \cdot 450.000 \text{ mm} \dots, \\ \dots, 450.000 \text{ mm} + 500.000 \text{ mm})$$

$$d_2 = 950.000 \text{ mm}$$

### Stress concentration factor

$$k_j = \left(\frac{b_2 \cdot d_2}{b_1 \cdot d_1}\right)^{\left(\frac{1}{2}\right)} = \left(\frac{950.000 \text{ mm} \cdot 950.000 \text{ mm}}{450.000 \text{ mm} \cdot 450.000 \text{ mm}}\right)^{\left(\frac{1}{2}\right)} = 2.111$$

$$\beta_j = \frac{2}{3} = 6.667 \times 10^{-1} \quad (\text{joint coeff.})$$

### Design value of concrete concentrated pressure strength

$$f_{jd} = \frac{\beta_j \cdot k_j \cdot f_{ck}}{\gamma_c} = \frac{6.667 \times 10^{-1} \cdot 2.111 \cdot 25.000 \text{ MPa}}{1.5} = 23.457 \text{ MPa}$$

### Effective width of flexible plate

$$c = t \cdot \left( \frac{f_y}{3 \cdot f_{jd} \cdot \gamma_{M0}} \right)^{\left(\frac{1}{2}\right)} = 30.000 \text{ mm} \cdot \left( \frac{355.000 \text{ MPa}}{3 \cdot 23.457 \text{ MPa} \cdot 1.0} \right)^{\left(\frac{1}{2}\right)} = 67.381 \text{ mm}$$

$$A_{eff} = 234995.117 \text{ mm}^2$$

### Resistance of base plate

$$N_{bRd} = A_{eff} \cdot f_{jd} = 234995.117 \text{ mm}^2 \cdot 23.457 \text{ MPa} = 5.512 \text{ MN}$$

### Reliability condition

$$\frac{N_{Ed}}{N_{bRd}} = \frac{2.947 \text{ MN}}{5.512 \text{ MN}} = 5.347 \times 10^{-1}$$

$$\frac{N_{Ed}}{N_{bRd}} = 0.5347 < 1.0 \text{ Satisfied}$$

### Minimum normal force in combination with horizontal force in column base

$$V_{Ed} = \gamma_Q \cdot q_w \cdot \left( \frac{L_{Total}}{2} \right) \cdot \left( \frac{H_{Total}}{2} \right)$$

$$V_{Ed} = 1.5 \cdot 320.000 \text{ Pa} \cdot \left( \frac{34.000 \text{ m}}{2} \right) \cdot \left( \frac{12.600 \text{ m}}{2} \right)$$

$$V_{Ed} = 51.408 \text{ kN}$$

Shear resistance of one bolt made of rod with diameter 20 mm, steel S355

$$f_{ub} = 490.000 \text{ MPa} \quad f_{yb} = 355.000 \text{ MPa} \quad A_s = 245.000 \text{ mm}^2$$

$$n_b = 2 \quad \alpha_v = 0.5 \quad \gamma_{M2} = 1.25$$

$$\alpha_b = 3.335 \times 10^{-1}$$

$$F_{1vbRd} = 0.85 \cdot \frac{\alpha_v \cdot f_{ub} \cdot A_s}{\gamma_{M2}}$$

$$F_{1vbRd} = 0.85 \cdot \frac{0.5 \cdot 490.000 \text{ MPa} \cdot 245.000 \text{ mm}^2}{1.25} = 40.817 \text{ kN}$$

$$F_{2vbRd} = \frac{\alpha_b \cdot f_{ub} \cdot A_s}{\gamma_{M2}}$$

$$F_{2vbRd} = \frac{3.335 \times 10^{-1} \cdot 490.000 \text{ MPa} \cdot 245.000 \text{ mm}^2}{1.25} = 32.029 \text{ kN}$$

$$F_{vbRd} = \min(F_{1vbRd}, F_{2vbRd}) = \min(40.817 \text{ kN}, 32.029 \text{ kN}) = 32.029 \text{ kN}$$

### Resistance of two bolts in a column base

$$V_{Rd} = n_b \cdot F_{vbRd} = 2 \cdot 32.029 \text{ kN} = 64.059 \text{ kN}$$

### Reliability condition

$$\frac{V_{Ed}}{V_{Rd}} = \frac{51.408 \text{ kN}}{64.059 \text{ kN}} = 8.025 \times 10^{-1}$$

$$\frac{V_{Ed}}{V_{Rd}} = 0.8025 < 1.0 \text{ Satisfied}$$

The chosen base plate dimensions are satisfied requirements.

### 2.2.3. Final design stage

Before a refined and detailed structural analysis can be carried out, it is necessary to make a final determination of the loads for which the structure is to be designed, including both loads determined from codes of practice and the self-weight of the structure. The analyse of whole structure is done by SCIA Engineer software. Drawings and diagrams are made in AutoCAD and Tekla Structure.

#### 2.2.3.1. Geometry of structure

The structure is 111 meters long and 34 meters wide and column positions and grid plan is shown below.

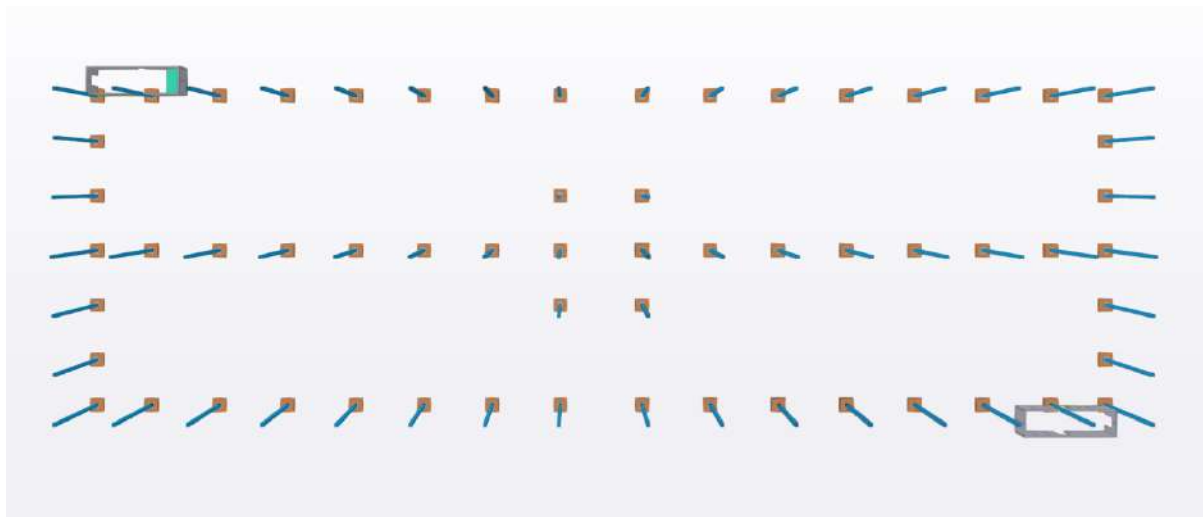
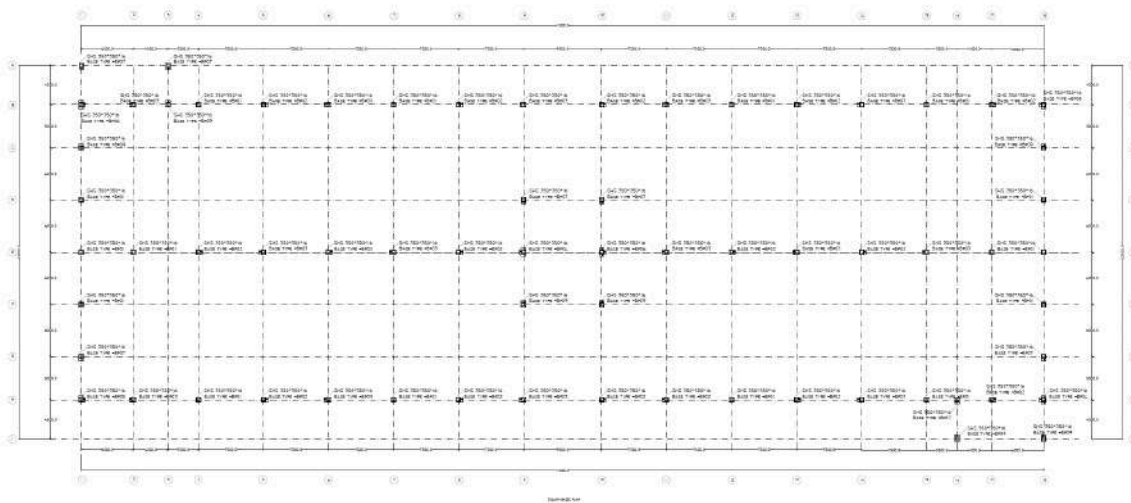
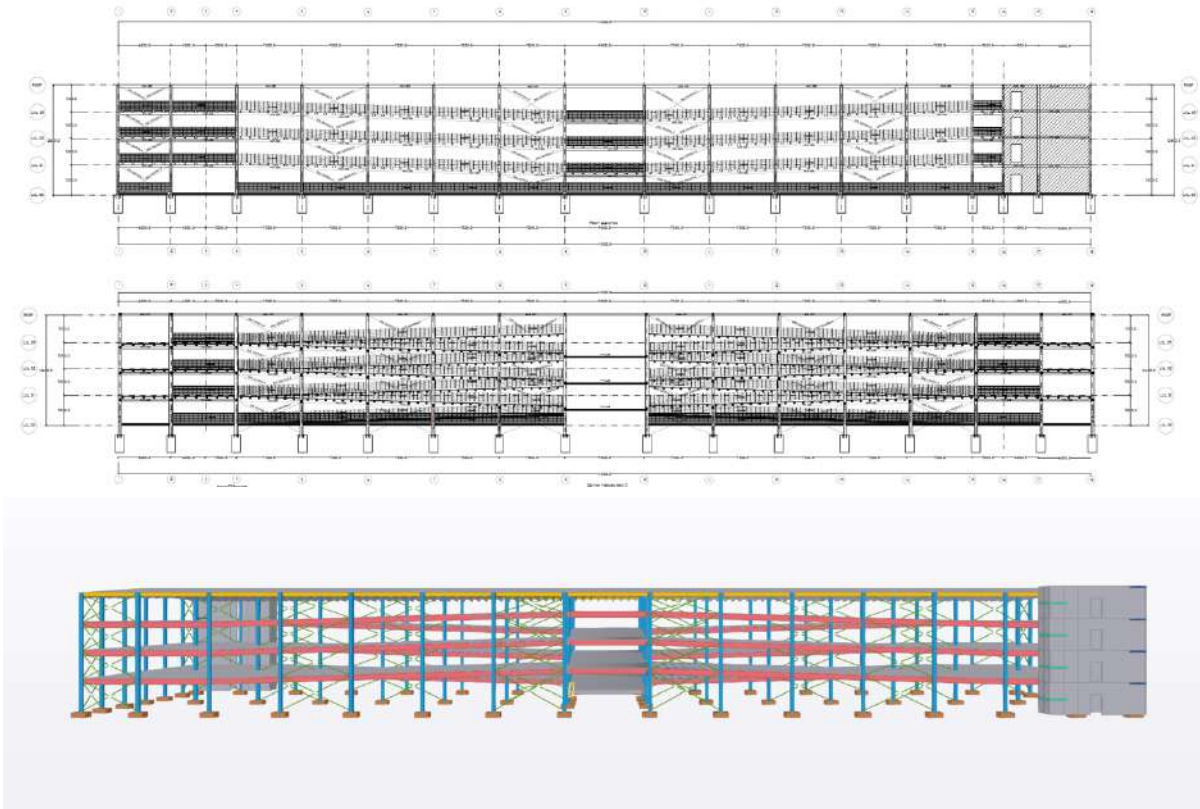
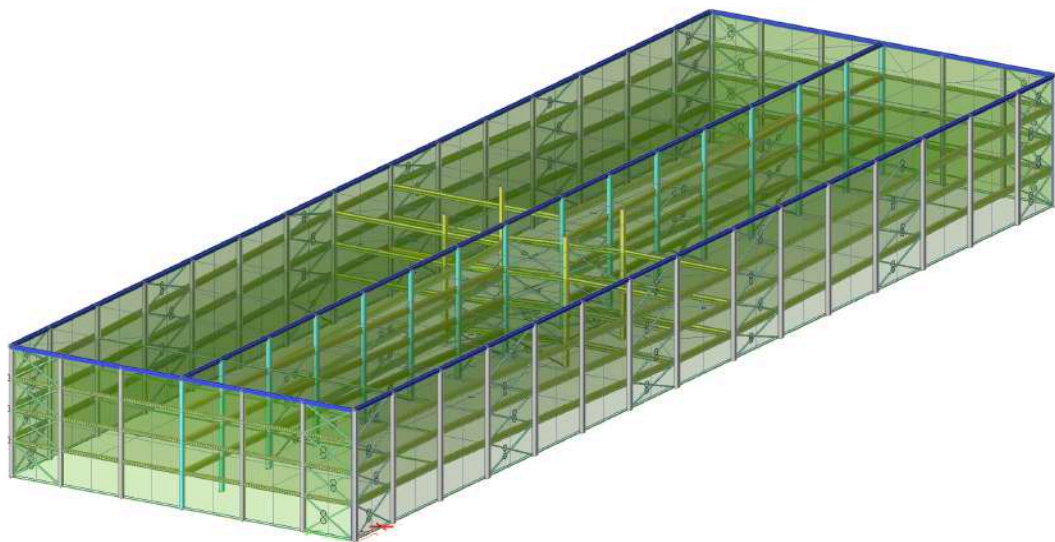


Figure 2.2.3.1. 1: Column positions and grids



*Figure 2.2.3.1. 2: Elevation of structure*



*Figure 2.2.3.1. 3: Geometry of structure*

### **2.2.3.2. Loads**

After geometry of structure is defined, load cases should be applied the structure. There are eight load cases which are self-weight, dead load, live load, snow load and wind generator which is applied four different angles ( $0^\circ, 90^\circ, 180^\circ, 270^\circ$ ).



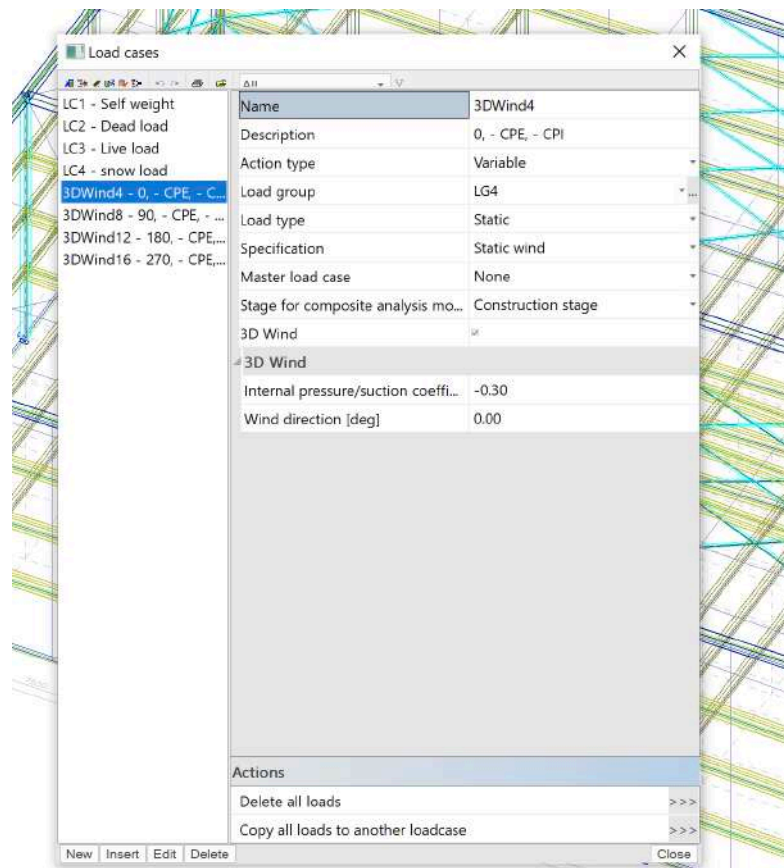


Figure 2.2.3.1. 4: Load cases

Each load case is shown below.

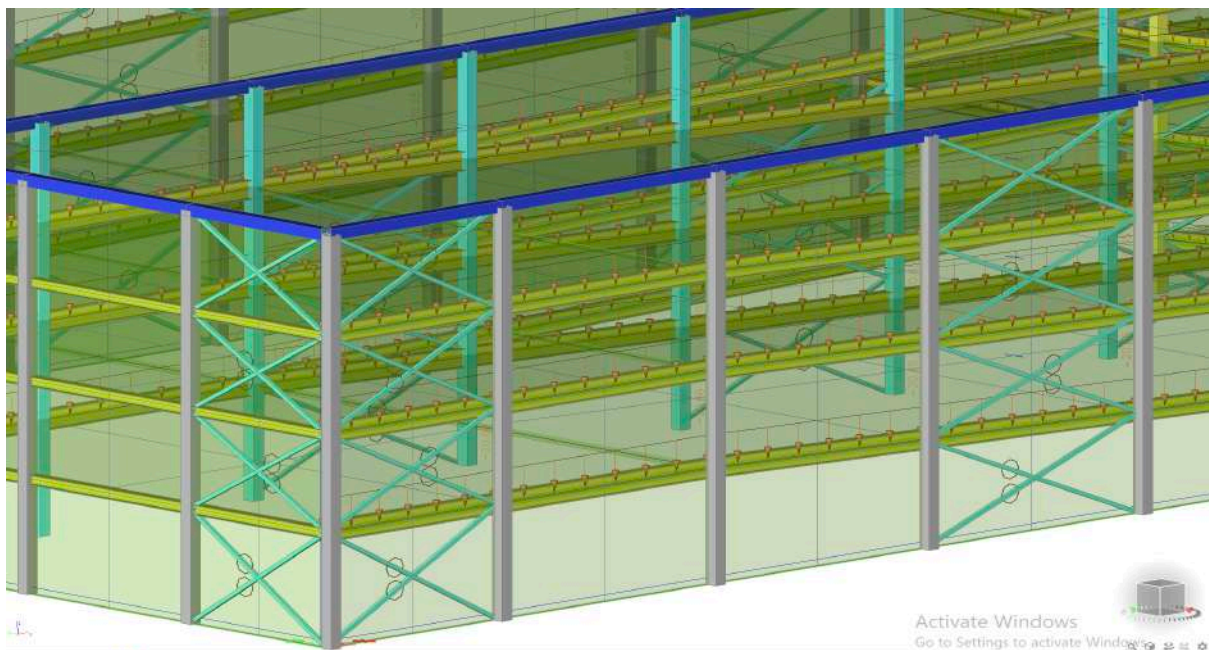


Figure 2.2.3.1. 5: Dead load



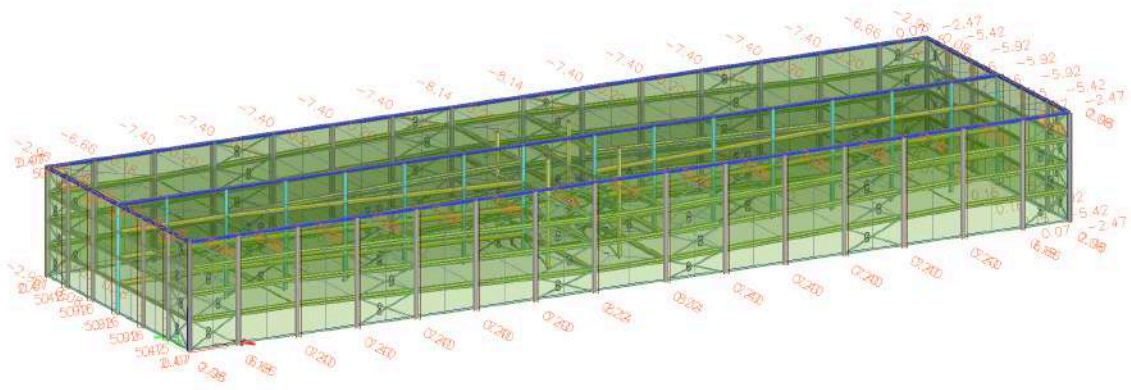


Figure 2.2.3.1. 9: 3D wind 90°

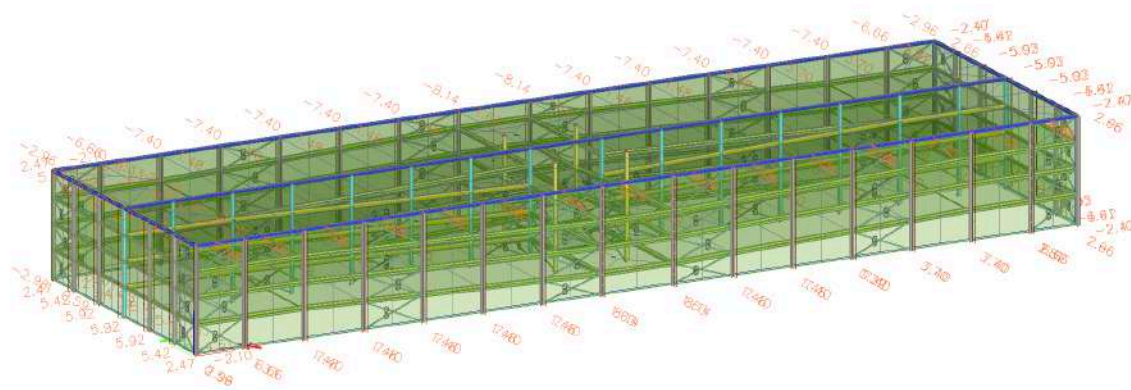


Figure 2.2.3.1. 10: 3D wind 180°

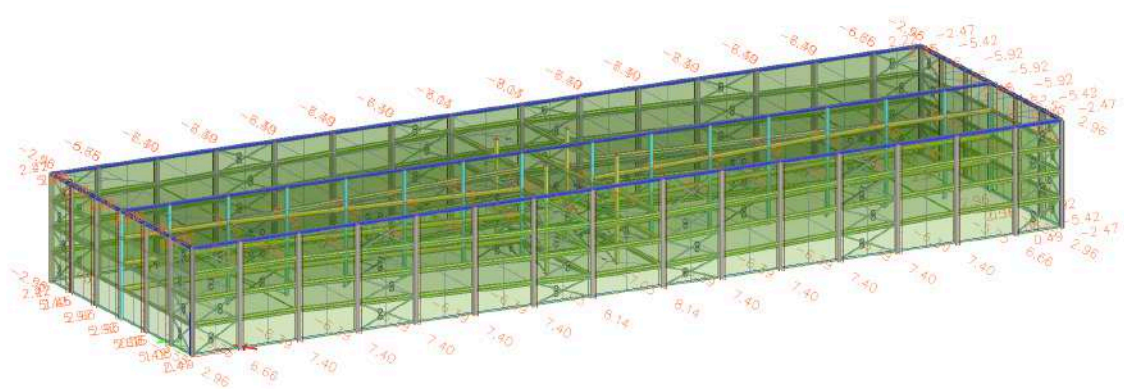


Figure 2.2.3.1. 11: 3D wind 270°

### 2.2.3.3. ULS Member Check

The ultimate limit state is the design for the safety of a structure and its users by limiting the stress that materials experience. In order to comply with engineering demands for strength and stability under design loads, ULS must be fulfilled as an established condition.

The necessity of checking sway mode of structure is important also it should be known that 'sensitive to buckling in a sway mode' does not mean the same as needing to consider second order effects due to the deformation of the structure. It means only that the geometrical deformation of the structure gives rise to additional effects in the members that must be taken into account in design. These additional effects may be only first order effects. If the geometrical deformation significantly affects the structural behaviour then second order effects also need to be considered.

First sway imperfection needs to be evaluated.

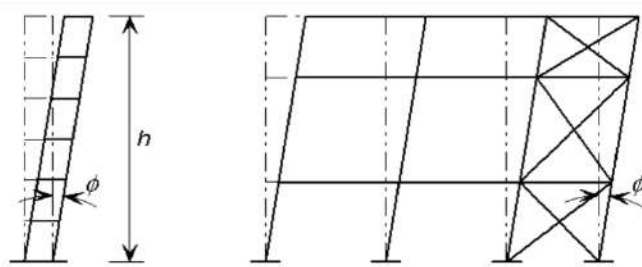


Figure 2.2.3.3. 1: Sway imperfection

$$\Phi = \Phi_0 \times \alpha_h \times \alpha_m$$

$$\Phi_0 = \frac{1}{200} = 0.005$$

$$\alpha_h = \frac{2}{(h)^{\frac{1}{2}}} = \frac{2}{(12.6)^{\frac{1}{2}}} = 0.563 \text{ but } 0.666 \leq \alpha_h \leq 1.0; \alpha_h = 0.666$$

$$\alpha_m = \sqrt{0.5 \left(1 + \frac{1}{m}\right)} = \sqrt{0.5 \left(1 + \frac{1}{3}\right)} = 0.816$$

$$\Phi = \Phi_0 \times \alpha_h \times \alpha_m = 2.72 \times 10^{-3}$$

Equivalent horizontal load

$$H_{Ed} = \Phi \times g \times l = 2.72 \times 10^{-3} \times 64.57 \times 34 = 5.04 \text{ kN}$$

$$\Sigma H_{Ed} = 5.04 + 57.577 = 62.617 \text{ kN}$$

$$\alpha_{cr} = \frac{\Sigma H_{Ed}}{\Sigma V_{Ed}} \times \frac{h}{\delta_{H,Ed}} = \frac{62.617}{2195} \times \frac{12600}{24} = 14.97$$

$\alpha_{cr} > 10$ , therefore 1<sup>st</sup> order linear analysis will be taken into account. [3]

Name	dx [m]	Case	uy [mm]	uy,rel [1/xx]	uz [mm]	uz,rel [1/xx]
By4_5	6	ULS-Set B (auto)/1	-16.9	-1/356	0	0
By41_3	0	ULS-Set B (auto)/2	24	1/136	0	0
B1443	8.500+	ULS-Set B (auto)/3	0.5	1/10000	-30	-1/283
B601	4.500-	ULS-Set B (auto)/4	0	0	-22.1	-1/173
F48_2	5.883	ULS-Set B (auto)/5	0	0	31	1/190

Table 2.2.3.3. 1:Relative deformation(extreme)

### 2.2.3.3.2. Beam Check

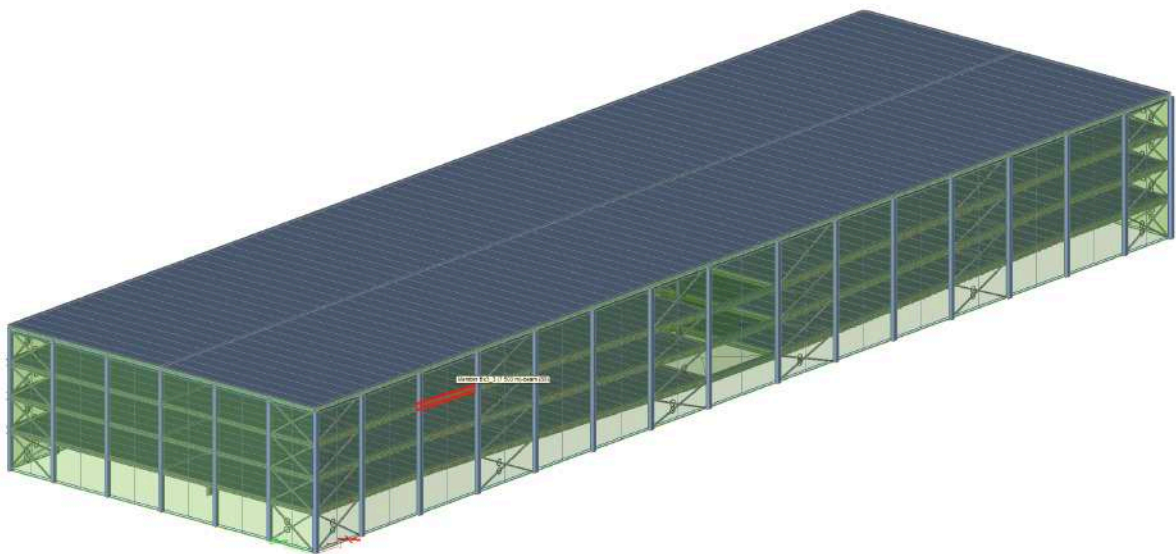


Figure 2.2.3.3. 2:Critical beam

## EC-EN 1993 Steel check ULS

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: All  
 Filter: Cross-section = CS12-Beam2 - HEA600

### EN 1993-1-1 Code Check

National annex: Czech CSN-EN NA

Member Bx3_3	7.503 / 7.503 m	HEA600	S 355	ULS-Set B (auto)	0.83 -
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#### Combination key

ULS-Set B (auto) / 1.15\*LC1 + 1.15\*LC2 + 1.50\*LC3 + 1.50\*LC4 + 0.90\*3DWind16

#### Partial safety factors

$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

#### Material

Yield strength	$f_y$	355.0	MPa
Ultimate strength	$f_u$	490.0	MPa
Fabrication		Rolled	

....SECTION CHECK:...

The critical check is on position 7.503 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	-61.17	kN
Shear force	$V_{y,Ed}$	-1.33	kN
Shear force	$V_{z,Ed}$	-263.12	kN
Torsion	$T_{Ed}$	0.11	kNm
Bending moment	$M_{y,Ed}$	-354.01	kNm
Bending moment	$M_{z,Ed}$	-9.95	kNm

### Classification for cross-section design

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993 1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_\psi$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	7.647e+04	8.676e+04	0.9	0.4	1.0	4.7	7.3	8.1	11.3	1
3	SO	116	25	7.055e+04	6.027e+04	0.9	0.5	1.0	4.7	7.3	8.1	11.9	1
4	I	486	13	6.361e+04	-5.821e+04	-0.9		0.5	37.4	56.2	65.0	92.2	1
5	SO	116	25	-7.107e+04	-8.136e+04								
7	SO	116	25	-6.515e+04	-5.487e+04								

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

### Compression check

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	2.2700e-02	m <sup>2</sup>
Compression resistance	$N_{c,Rd}$	8058.50	kN
Unity check		0.01	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{2.2700 \cdot 10^{-2} [m^2] \times 355.0 [MPa]}{1.00} = 8058.50 [kN] \quad (EC3-1-1: 6.10)$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-61.17 [kN]|}{8058.50 [kN]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.9)$$

### Bending moment check for $M_y$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	5.3333e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Unity check		0.19	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 1893.33 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|-354.01 [kNm]|}{1893.33 [kNm]} = 0.19 \leq 1.00 \quad (EC3-1-1: 6.12)$$

### Bending moment check for $M_z$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,z}$	1.1542e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,z,Rd}$	409.73	kNm
Unity check		0.02	-

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 409.73 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} = \frac{|-9.95 [kNm]|}{409.73 [kNm]} = 0.02 \leq 1.00 \quad (EC3-1-1: 6.12)$$

#### Shear check for $V_y$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	1.5520e-02	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	3180.97	kN
Unity check		0.00	-

$$V_{pl,y,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{1.5520 \cdot 10^{-2} [m^2] \times \frac{355.0 [MPa]}{\sqrt{3}}}{1.00} = 3180.97 [kN] \quad (EC3-1-1: 6.18)$$

$$\text{Unity check} = \frac{|V_{y,Ed}|}{V_{pl,y,Rd}} = \frac{|-1.33 [kN]|}{3180.97 [kN]} = 0.00 \leq 1.00 \quad (EC3-1-1: 6.17)$$

#### Shear check for $V_z$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.3750e-03	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	1921.49	kN
Unity check		0.14	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.3750 \cdot 10^{-3} [m^2] \times \frac{355.0 [MPa]}{\sqrt{3}}}{1.00} = 1921.49 [kN] \quad (EC3-1-1: 6.18)$$

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{pl,z,Rd}} = \frac{|-263.12 [kN]|}{1921.49 [kN]} = 0.14 \leq 1.00 \quad (EC3-1-1: 6.17)$$

#### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	2	
Total torsional moment	$T_{Ed}$	0.7	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.00	-

$$\tau_{Ed} = \frac{T_{Ed}}{I_p} \times \tau_{Ed,unit} = \frac{105.45}{6.284} \times 6.284 [kN/m^2] = 0.7 [MPa]$$

$$\tau_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [MPa]}{\sqrt{3} \times 1.00} = 205.0 [MPa]$$

$$\text{Unity check} = \frac{\tau_{Ed}}{\tau_{Rd}} = \frac{0.7 [MPa]}{205.0 [MPa]} = 0.00 \leq 1.00 \quad (EC3-1-1: 6.23)$$

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

#### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.41)

Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Exponent of bending ratio y	$\alpha$	2.00	
Plastic bending moment	$M_{pl,z,Rd}$	409.73	kNm
Exponent of bending ratio z	$\beta$	1.00	

$$\text{Unity check (6.41)} = 0.03 + 0.02 = 0.06$$

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 1893.33 [kNm] \quad (EC3-1-1: 6.13)$$

$$\alpha = 2.00$$

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 409.73 [kNm] \quad (EC3-1-1: 6.13)$$

$$\beta = 1.00$$

$$\text{Unity check} = \left( \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} \right)^\alpha + \left( \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} \right)^\beta = \left( \frac{|-354.01 [kNm]|}{1893.33 [kNm]} \right)^{2.00} + \left( \frac{|-9.95 [kNm]|}{409.73 [kNm]} \right)^{1.00} = 0.06 \leq 1.00 \quad (EC3-1-1: 6.41)$$

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

**Note:** Since the axial force satisfies both criteria (6.33) and (6.34) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the y-y axis is neglected.

**Note:** Since the axial force satisfies criteria (6.35) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the z-z axis is neglected.

The member satisfies the section check.

....:STABILITY CHECK:....

**Classification for member buckling design**

Decisive position for stability classification: 0,000 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_{\sigma}$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	2.142e+03	2.142e+03	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1
3	SO	116	25	2.142e+03	2.142e+03	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1
4	I	486	13	2.142e+03	2.142e+03	1.0	1.0	1.0	37.4	22.8	27.7	30.9	4
5	SO	116	25	2.142e+03	2.142e+03	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1
7	SO	116	25	2.142e+03	2.142e+03	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1

**Note:** The Classification limits have been set according to Semi-Comp+.

The cross-section is classified as Class 4

**Effective section N-**

**Effective width calculation**

According to EN 1993-1-5 article 4.4

Id	Type	$b_p$ [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_{\sigma}$ [-]	$\lambda_p$ [-]	$\rho$ [-]	$b_e$ [mm]	$b_{e1}$ [mm]	$b_{e2}$ [mm]
1	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
3	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
4	I	486	3.550e+05	3.550e+05	1.0	4.0	0.8	0.9	437	219	219
5	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
7	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		

**Effective section My-**

**Effective width calculation**

According to EN 1993-1-5 article 4.4

Id	Type	$b_p$ [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_{\sigma}$ [-]	$\lambda_p$ [-]	$\rho$ [-]	$b_e$ [mm]	$b_{e1}$ [mm]	$b_{e2}$ [mm]
1	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
3	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
4	I	486	3.054e+05	-3.054e+05	-1.0	23.9	0.3	1.0	243	97	146
5	SO	116	-3.550e+05	-3.550e+05							
7	SO	116	-3.550e+05	-3.550e+05							

**Effective section Mz-**

**Effective width calculation**

According to EN 1993-1-5 article 4.4



**Effective section Mz-  
Effective width calculation**  
According to EN 1993-1-5 article 4.4

Id	Type	b <sub>p</sub> [mm]	σ <sub>1</sub> [kN/m <sup>2</sup> ]	σ <sub>2</sub> [kN/m <sup>2</sup> ]	ψ [-]	k <sub>σ</sub> [-]	λ <sub>p</sub> [-]	ρ [-]	b <sub>e</sub> [mm]	b <sub>e1</sub> [mm]	b <sub>e2</sub> [mm]
1	SO	116	3.550e+05	7.928e+04	0.2	0.5	0.3	1.0	116		
3	SO	116	-7.928e+04	-3.550e+05							
4	I	486	0.000e+00	0.000e+00							
5	SO	116	-7.928e+04	-3.550e+05							
7	SO	116	3.550e+05	7.928e+04	0.2	0.5	0.3	1.0	116		

Effective properties						
Effective area	A <sub>eff</sub>	2.2017e-02	m <sup>2</sup>			
Effective second moment of area	I <sub>eff,y</sub>	1.4123e-03	m <sup>4</sup>	I <sub>eff,z</sub>	1.1271e-04	m <sup>4</sup>
Effective section modulus	W <sub>eff,y</sub>	4.7874e-03	m <sup>3</sup>	W <sub>eff,z</sub>	7.5143e-04	m <sup>3</sup>
Shift of the centroid	e <sub>N,y</sub>	0	mm	e <sub>N,z</sub>	0	mm

**Flexural Buckling check**

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters		yy	zz	
Sway type		sway	non-sway	
System length	L	7.503	22.510	m
Buckling factor	k	3.24	1.00	
Buckling length	l <sub>cr</sub>	24.310	22.509	m
Critical Euler load	N <sub>cr</sub>	4945.03	462.25	kN
Slenderness	λ	97.54	319.03	
Relative slenderness	λ <sub>rel</sub>	1.26	4.11	
Limit slenderness	λ <sub>rel,0</sub>	0.20	0.20	
Buckling curve		a	b	
Imperfection	α	0.21	0.34	
Reduction factor	χ	0.50	0.05	
Buckling resistance	N <sub>b,Rd</sub>	3868.90	426.75	kN

Flexural Buckling verification			
Cross-section effective area	A <sub>eff</sub>	2.2017e-02	m <sup>2</sup>
Buckling resistance	N <sub>b,Rd</sub>	426.75	kN
Unity check		0.14	-

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 1.4100 \cdot 10^{-3} [\text{m}^4]}{24.310 [\text{m}]^2} = 4945.03 [\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 1.1300 \cdot 10^{-4} [\text{m}^4]}{22.509 [\text{m}]^2} = 462.25 [\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{24.310 [\text{m}]}{249 [\text{mm}]} = 97.54$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{22.509 [\text{m}]}{71 [\text{mm}]} = 319.03$$

$$\lambda_{rel,y} = \frac{\lambda_y \times \sqrt{\frac{A_{eff}}{A}}}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{97.54 \times \sqrt{\frac{2.2017 \cdot 10^{-2} [\text{m}^2]}{2.2700 \cdot 10^{-2} [\text{m}^2]}}}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{355.0 [\text{MPa}]}}} = 1.26 \quad (\text{EC3-1-1: 6.51})$$

$$\lambda_{rel,z} = \frac{\lambda_z \times \sqrt{\frac{A_{eff}}{A}}}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{319.03 \times \sqrt{\frac{2.2017 \cdot 10^{-2} [\text{m}^2]}{2.2700 \cdot 10^{-2} [\text{m}^2]}}}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{355.0 [\text{MPa}]}}} = 4.11 \quad (\text{EC3-1-1: 6.51})$$

$$\varphi_y = 0.5 \times [1 + \alpha_y \times (\lambda_{rel,y} - \lambda_{rel,y,0}) + \lambda_{rel,y}^2] = 0.5 \times [1 + 0.21 \times (1.26 - 0.20) + 1.26^2] = 1.40$$

$$\varphi_z = 0.5 \times [1 + \alpha_z \times (\lambda_{rel,z} - \lambda_{rel,z,0}) + \lambda_{rel,z}^2] = 0.5 \times [1 + 0.34 \times (4.11 - 0.20) + 4.11^2] = 9.62$$

$$\chi_y = \min \left( \frac{1}{\varphi_y + \sqrt{\varphi_y^2 - \lambda_{rel,y}^2}}, \frac{1}{\lambda_{rel,y}^2}, 1 \right) = \min \left( \frac{1}{1.40 + \sqrt{1.40^2 - 1.26^2}}, \frac{1}{1.26^2}, 1 \right) = \min(0.50, 0.63, 1) = 0.50 \quad (\text{EC3-1-1: 6.49})$$

$$\chi_z = \min \left( \frac{1}{\varphi_z + \sqrt{\varphi_z^2 - \lambda_{rel,z}^2}}, \frac{1}{\lambda_{rel,z}^2}, 1 \right) = \min \left( \frac{1}{9.62 + \sqrt{9.62^2 - 4.11^2}}, \frac{1}{4.11^2}, 1 \right) = \min(0.05, 0.06, 1) = 0.05 \quad (\text{EC3-1-1: 6.49})$$

$$N_{b,y,Rd} = \frac{\chi_y \times A_{eff} \times f_y}{\gamma_{M1}} = \frac{0.50 \times 2.2017 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 3868.90 [\text{kN}] \quad (\text{EC3-1-1: 6.48})$$

$$N_{b,z,Rd} = \frac{\chi_z \times A_{eff} \times f_y}{\gamma_{M1}} = \frac{0.05 \times 2.2017 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 426.75 [\text{kN}] \quad (\text{EC3-1-1: 6.48})$$

$$N_{b,Rd} = \min(N_{b,y,Rd}; N_{b,z,Rd}) = \min(3868.90[\text{kN}]; 426.75[\text{kN}]) = 426.75[\text{kN}]$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{b,Rd}} = \frac{|-61.17[\text{kN}]|}{426.75[\text{kN}]} = 0.14 \leq 1.00$$

(EC3-1-1: 6.46)

### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** For this I-section the Torsional(-Flexural) buckling resistance is higher than the resistance for Flexural buckling. Therefore Torsional(-Flexural) buckling is not printed on the output.

### Lateral Torsional Buckling check

According to EN 1993-1-1 article 6.3.2.1 & 6.3.2.3 and formula (6.54)

LTB parameters			
Method for LTB curve		Alternative case	
Effective section modulus	$W_{eff,y}$	4.7874e-03	m <sup>3</sup>
Elastic critical moment	$M_{cr}$	548.50	kNm
Relative slenderness	$\lambda_{rel,LT}$	1.76	
Limit slenderness	$\lambda_{rel,LT,0}$	0.40	
LTB curve		b	
Imperfection	$\alpha_{LT}$	0.34	
LTB factor	$\beta$	0.75	
Reduction factor	$\chi_{LT}$	0.32	
Correction factor	$k_c$	0.86	
Correction factor	$f$	1.00	
Modified reduction factor	$\chi_{LT,mod}$	0.32	
Design buckling resistance	$M_{b,Rd}$	548.50	kNm
Unity check		0.65	-

Mcr parameters			
LTB length	$l_{LT}$	22.510	m
Influence of load position		no influence	
Correction factor	$k$	1.00	
Correction factor	$k_w$	1.00	
LTB moment factor	$C_1$	1.35	
LTB moment factor	$C_2$	0.63	
LTB moment factor	$C_3$	0.41	

Mcr parameters			
Shear centre distance	$d_z$	0	mm
Distance of load application	$z_g$	0	mm
Mono-symmetry constant	$\beta_y$	0	mm
Mono-symmetry constant	$z_j$	0	mm

$$M_{cr} = C_1 \times \frac{\pi^2 \times E \times I_x}{l_{LT}^2} \times \left[ \left( \sqrt{\frac{k}{k_w}} \right)^2 \times \frac{I_w}{I_x} + \frac{I_T \times G \times I_x}{\pi^2 \times E \times I_x} + (C_2 \times z_g - C_3 \times z_j)^2 - (C_2 \times z_g - C_3 \times z_j) \right] = 1.35$$

$$\times \frac{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]}{22.510[\text{m}]^2}$$

$$\times \left[ \sqrt{\left( \frac{1.00}{1.00} \right)^2 \times \frac{8.9782 \cdot 10^{-6}[\text{m}^6]}{1.1300 \cdot 10^{-4}[\text{m}^4]} + \frac{22.510[\text{m}]^2 \times 80769.2[\text{MPa}] \times 3.9800 \cdot 10^{-6}[\text{m}^4]}{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4}} + (0.63 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}])^2 - (0.63 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}]) \right]$$

$$= 548.50[\text{kNm}]$$

$$\lambda_{rel,LT} = \sqrt{\frac{W_{eff,y} \times f_y}{M_{cr}}} = \sqrt{\frac{4.7874 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{548.50[\text{kNm}]} = 1.76$$

$$\beta = 0.75$$

$$\chi_{LT} = \min \left( \frac{1}{\varphi_{LT} + \sqrt{\varphi_{LT}^2 - \beta \times \lambda_{rel,LT}^2}}, \frac{1}{\lambda_{rel,LT}^2}, 1 \right) = \min \left( \frac{1}{1.89 + \sqrt{1.89^2 - 0.75 \times 1.76^2}}, \frac{1}{1.76^2}, 1 \right) = \min(0.33, 0.32, 1) = 0.32 \quad (\text{EC3-1-1: 6.57})$$

$$f = \min \left\{ 1 - 0.5 \times (1 - k_c) \times \left[ 1 - 2 \times (\lambda_{rel,LT} - 0.8)^2 \right], 1 \right\} = \min \left\{ 1 - 0.5 \times (1 - 0.86) \times \left[ 1 - 2 \times (1.76 - 0.8)^2 \right], 1 \right\} = \min \{1.06, 1\} = 1.00$$

$$\chi_{LT,mod} = \min \left( \frac{\chi_{LT}}{f}, 1 \right) = \min \left( \frac{0.32}{1.00}, 1 \right) = \min(0.32, 1) = 0.32$$

$$M_{b,Rd} = \chi_{LT,mod} \times W_{eff,y} \times \frac{f_y}{\gamma_{M1}} = 0.32 \times 4.7874 \cdot 10^{-3}[\text{m}^3] \times \frac{355.0[\text{MPa}]}{1.00} = 548.50[\text{kNm}] \quad (\text{EC3-1-1: 6.55})$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{b,Rd}} = \frac{|-354.01[\text{kNm}]|}{548.50[\text{kNm}]} = 0.65 \leq 1.00 \quad (\text{EC3-1-1: 6.54})$$

**Note:** C parameters are determined according to ECCS 119 2006 / Galea 2002.

**Note:** The correction factor  $k_c$  is determined from  $C_1$ .

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

Bending and axial compression check parameters			
Interaction method		alternative method 2	
Cross-section effective area	$A_{eff}$	2.2017e-02	$m^2$
Effective section modulus	$W_{eff,y}$	4.7874e-03	$m^3$
Effective section modulus	$W_{eff,z}$	7.5143e-04	$m^3$
Design compression force	$N_{Ed}$	61.17	kN
Design bending moment (maximum)	$M_{y,Ed}$	-354.01	kNm
Design bending moment (maximum)	$M_{z,Ed}$	-12.65	kNm
Additional moment	$\Delta M_{y,Ed}$	0.00	kNm
Additional moment	$\Delta M_{z,Ed}$	0.00	kNm
Characteristic compression resistance	$N_{Rk}$	7815.95	kN
Characteristic moment resistance	$M_{y,Rk}$	1699.52	kNm
Characteristic moment resistance	$M_{z,Rk}$	266.76	kNm
Reduction factor	$\chi_y$	0.50	
Reduction factor	$\chi_z$	0.05	
Modified reduction factor	$\chi_{LT,mod}$	0.32	
Interaction factor	$k_{yy}$	0.91	
Interaction factor	$k_{yz}$	0.98	
Interaction factor	$k_{zy}$	0.99	
Interaction factor	$k_{zz}$	0.98	

Maximum moment  $M_{y,Ed}$  is derived from beam Bx3\_3 position 7.503 m.  
Maximum moment  $M_{z,Ed}$  is derived from beam Bx5\_3 position 0.000 m.

Interaction method 2 parameters			
Method for interaction factors		Table B.2	
Sway type y		sway	
Equivalent moment factor	$C_{my}$	0.90	
Resulting load type z		point load F	
End moment	$M_{h,z}$	0.00	kNm
Field moment	$M_{s,z}$	-12.65	kNm
Factor	$\alpha_{h,z}$	0.00	
Ratio of end moments	$\psi_z$	1.00	
Equivalent moment factor	$C_{mz}$	0.90	
Resulting load type LT		point load F	
End moment	$M_{h,LT}$	0.00	kNm
Field moment	$M_{s,LT}$	-354.01	kNm

Interaction method 2 parameters			
Factor	$\alpha_{h,LT}$	0.00	
Ratio of end moments	$\psi_{LT}$	1.00	
Equivalent moment factor	$C_{mLT}$	0.90	

Unity check (6.61) =  $0.02 + 0.59 + 0.05 = 0.65$

Unity check (6.62) =  $0.14 + 0.64 + 0.05 = 0.83$

$C_{my} = 0.90$

$$\alpha_{h,z} = \frac{M_{h,z}}{M_{s,z}} = \frac{0.00[\text{kNm}]}{-12.65[\text{kNm}]} = 0.00$$

$$C_{mz} = 0.9 + 0.1 \times \alpha_{h,z} = 0.9 + 0.1 \times 0.00 = 0.90$$

$$\alpha_{h,LT} = \frac{M_{h,LT}}{M_{s,LT}} = \frac{0.00[\text{kNm}]}{-354.01[\text{kNm}]} = 0.00$$

$$C_{mLT} = 0.9 + 0.1 \times \alpha_{h,LT} = 0.9 + 0.1 \times 0.00 = 0.90$$

$$N_{Rk} = A_{eff} \times f_y = 2.2017 \cdot 10^{-2}[\text{m}^2] \times 355.0[\text{MPa}] = 7815.95[\text{kN}]$$

$$M_{y,Rk} = W_{eff,y} \times f_y = 4.7874 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}] = 1699.52[\text{kNm}]$$

$$M_{z,Rk} = W_{eff,z} \times f_y = 7.5143 \cdot 10^{-4}[\text{m}^3] \times 355.0[\text{MPa}] = 266.76[\text{kNm}]$$

$$k_{yy} = \min \left[ C_{my} \times \left( 1 + 0.6 \times \lambda_{rel,y} \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right), C_{my} \times \left( 1 + 0.6 \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right]$$

$$= \min \left[ 0.90 \times \left( 1 + 0.6 \times 1.26 \times \frac{61.17[\text{kN}]}{0.50 \times \frac{7815.95[\text{kN}]}{1.00}} \right), 0.90 \times \left( 1 + 0.6 \times \frac{61.17[\text{kN}]}{0.50 \times \frac{7815.95[\text{kN}]}{1.00}} \right) \right] = \min [0.91, 0.91] = 0.91$$

$$k_{yz} = k_{zz} = 0.98$$

$$k_{zy} = \max \left( 1 - \frac{0.05 \times \lambda_{rel,z}}{C_{mLT} - 0.25} \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}}, 1 - \frac{0.05}{C_{mLT} - 0.25} \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right)$$

$$= \max \left( 1 - \frac{0.05 \times 4.11}{0.90 - 0.25} \times \frac{61.17[\text{kN}]}{0.05 \times \frac{7815.95[\text{kN}]}{1.00}}, 1 - \frac{0.05}{0.90 - 0.25} \times \frac{61.17[\text{kN}]}{0.05 \times \frac{7815.95[\text{kN}]}{1.00}} \right) = \max (0.95, 0.99) = 0.99$$

$$k_{zz} = \min \left[ C_{mz} \times \left( 1 + 0.6 \times \lambda_{rel,z} \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right), C_{mz} \times \left( 1 + 0.6 \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right]$$

$$= \min \left[ 0.90 \times \left( 1 + 0.6 \times 4.11 \times \frac{61.17[\text{kN}]}{0.05 \times \frac{7815.95[\text{kN}]}{1.00}} \right), 0.90 \times \left( 1 + 0.6 \times \frac{61.17[\text{kN}]}{0.05 \times \frac{7815.95[\text{kN}]}{1.00}} \right) \right] = \min [1.22, 0.98] = 0.98$$

$$\text{Unity check (6.61)} = \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{yy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT,mod} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{yz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}} \quad (\text{EC3-1-1: 6.61})$$

$$= \frac{61.17[\text{kN}]}{0.50 \times \frac{7815.95[\text{kN}]}{1.00}} + 0.91 \times \frac{|-354.01[\text{kNm}]| + |0.00[\text{kNm}]|}{0.32 \times \frac{1699.52[\text{kNm}]}{1.00}} + 0.98 \times \frac{|-12.65[\text{kNm}]| + |0.00[\text{kNm}]|}{\frac{266.76[\text{kNm}]}{1.00}} = \mathbf{0.65} \leq \mathbf{1.00}$$

$$\text{Unity check (6.62)} = \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT,mod} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}} \quad (\text{EC3-1-1: 6.62})$$

$$= \frac{61.17[\text{kN}]}{0.05 \times \frac{7815.95[\text{kN}]}{1.00}} + 0.99 \times \frac{|-354.01[\text{kNm}]| + |0.00[\text{kNm}]|}{0.32 \times \frac{1699.52[\text{kNm}]}{1.00}} + 0.98 \times \frac{|-12.65[\text{kNm}]| + |0.00[\text{kNm}]|}{\frac{266.76[\text{kNm}]}{1.00}} = \mathbf{0.83} \leq \mathbf{1.00}$$

Unity check = max(Unity check (6.61), Unity check (6.62)) = max(0.65, 0.83) = **0.83** ≤ **1.00**

### Shear Buckling check

According to EN 1993-1-5 article 5 & 7.1 and formula (5.10) & (7.1)

Shear Buckling parameters			
Buckling field length	a	7.503	m
Web		unstiffened	
Web height	h <sub>w</sub>	540	mm
Web thickness	t	13	mm
Material coefficient	ε	0.81	
Shear correction factor	η	1.20	

Shear Buckling verification		
Web slenderness	h <sub>w</sub> /t	41.54
Web slenderness limit		48.82

$$h_w/t = \frac{h_w}{t} = \frac{540[\text{mm}]}{13[\text{mm}]} = 41.54$$

$$\text{limit } h_w/t = \frac{72 \times \varepsilon}{\eta} = \frac{72 \times 0.81}{1.20} = 48.82$$

**Note:** The web slenderness is such that Shear Buckling effects may be ignored according to EN 1993-1-5 article 5.1(2).

The member satisfies the stability check.

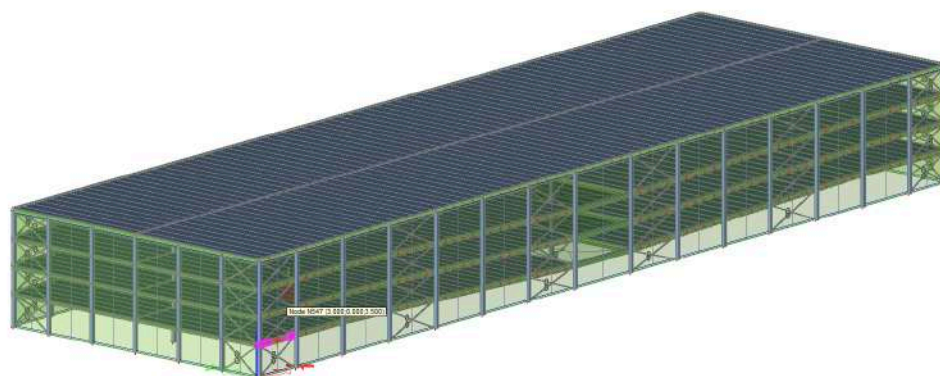


Figure 2.2.3.3. 3: Edge beam

## EC-EN 1993 Steel check ULS

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: Bx1\_1  
 Filter: Cross-section = CS12-Beam2 - HEA600

### EN 1993-1-1 Code Check

National annex: Czech CSN-EN NA

**Member Bx1\_1**   **6.000 / 6.000 m**   **HEA600**   **S 355**   **ULS-Set B (auto)**   **0.29 -**

#### Combination key

ULS-Set B (auto) / 1.15\*LC1 + 1.15\*LC2 + 1.50\*LC3 + 1.50\*LC4 + 0.90\*3DWind16

#### Partial safety factors

$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

#### Material

Yield strength	$f_y$	355.0	MPa
Ultimate strength	$f_u$	490.0	MPa
Fabrication		Rolled	

...:SECTION CHECK:...

The critical check is on position 6.000 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	52.34	kN
Shear force	$V_{y,Ed}$	0.51	kN
Shear force	$V_{z,Ed}$	-198.13	kN
Torsion	$T_{Ed}$	0.05	kNm
Bending moment	$M_{y,Ed}$	-251.70	kNm
Bending moment	$M_{z,Ed}$	6.12	kNm

#### Classification for cross-section design

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_\rho$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	4.622e+04	3.990e+04	0.9	0.5	1.0	4.7	7.3	8.1	11.8	1
3	SO	116	25	4.986e+04	5.618e+04	0.9	0.4	1.0	4.7	7.3	8.1	11.3	1
4	I	486	13	4.100e+04	-4.562e+04	-1.1		0.5	37.4	60.0	69.1	112.4	1
5	SO	116	25	-5.084e+04	-4.452e+04								
7	SO	116	25	-5.448e+04	-6.080e+04								

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

#### Tension check

According to EN 1993-1-1 article 6.2.3 and formula (6.5)

Cross-section area	A	2.2700e-02	m <sup>2</sup>
Plastic tension resistance	$N_{pl,Rd}$	8058.50	kN
Ultimate tension resistance	$N_{u,Rd}$	8008.56	kN
Tension resistance	$N_{t,Rd}$	8008.56	kN
Unity check		0.01	-

$$N_{pl,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{2.2700 \cdot 10^{-2} [m^2] \times 355.0 [MPa]}{1.00} = 8058.50 [kN] \quad (EC3-1-1: 6.6)$$

$$N_{u,Rd} = \frac{0.9 \times A \times f_u}{\gamma_{M2}} = \frac{0.9 \times 2.2700 \cdot 10^{-2} [m^2] \times 490.0 [MPa]}{1.25} = 8008.56 [kN] \quad (EC3-1-1: 6.7)$$

$$N_{t,Rd} = \min(N_{pl,Rd}, N_{u,Rd}) = \min(8058.50 [kN], 8008.56 [kN]) = 8008.56 [kN]$$

$$\text{Unity check} = \frac{N_{Ed}}{N_{t,Rd}} = \frac{52.34 [kN]}{8008.56 [kN]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.5)$$

#### Bending moment check for $M_y$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	5.3333e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Unity check		0.13	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 1893.33 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|-251.70 [kNm]|}{1893.33 [kNm]} = 0.13 \leq 1.00 \quad (EC3-1-1: 6.12)$$

### Bending moment check for $M_z$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,z}$	1.1542e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,z,Rd}$	409.73	kNm
Unity check		0.01	-

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 409.73 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\text{Unity check} = \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} = \frac{|6.12 [\text{kNm}]|}{409.73 [\text{kNm}]} = 0.01 \leq 1.00 \quad (\text{EC3-1-1: 6.12})$$

### Shear check for $V_y$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	1.5520e-02	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	3180.97	kN
Unity check		0.00	-

$$V_{pl,y,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{1.5520 \cdot 10^{-2} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 3180.97 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{y,Ed}|}{V_{c,y,Rd}} = \frac{|0.51 [\text{kN}]|}{3180.97 [\text{kN}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Shear check for $V_z$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.3750e-03	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	1921.49	kN
Unity check		0.10	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.3750 \cdot 10^{-3} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 1921.49 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{c,z,Rd}} = \frac{|-198.13 [\text{kN}]|}{1921.49 [\text{kN}]} = 0.10 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	2	
Total torsional moment	$T_{Ed}$	0.3	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.00	-

$$\tau_{Ed} = |T_{Ed}| \times \tau_{Ed,unit} = |53.94| \times 6.284 [\text{kN/m}^2] = 0.3 [\text{MPa}]$$

$$\tau_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [\text{MPa}]}{\sqrt{3} \times 1.00} = 205.0 [\text{MPa}]$$

$$\text{Unity check} = \frac{\tau_{Ed}}{\tau_{Rd}} = \frac{0.3 [\text{MPa}]}{205.0 [\text{MPa}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.23})$$

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.41)

Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Exponent of bending ratio $y$	$\alpha$	2.00	
Plastic bending moment	$M_{pl,z,Rd}$	409.73	kNm
Exponent of bending ratio $z$	$\beta$	1.00	

$$\text{Unity check (6.41)} = 0.02 + 0.01 = 0.03 -$$

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 1893.33 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\alpha = 2.00$$

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 409.73 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\beta = 1.00$$

$$\text{Unity check} = \left( \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} \right)^\alpha + \left( \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} \right)^\beta = \left( \frac{|-251.70 [\text{kNm}]|}{1893.33 [\text{kNm}]} \right)^{2.00} + \left( \frac{|6.12 [\text{kNm}]|}{409.73 [\text{kNm}]} \right)^{1.00} = 0.03 \leq 1.00 \quad (\text{EC3-1-1: 6.41})$$

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

**Note:** Since the axial force satisfies both criteria (6.33) and (6.34) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the y-y axis is neglected.

**Note:** Since the axial force satisfies criteria (6.35) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the z-z axis is neglected.

The member satisfies the section check.

...:STABILITY CHECK:...:

**Classification for member buckling design**

Decisive position for stability classification: 6.000 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\alpha$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	4.622e+04	3.990e+04	0.9	0.5	1.0	4.7	7.3	8.1	11.8	1
3	SO	116	25	4.986e+04	5.618e+04	0.9	0.4	1.0	4.7	7.3	8.1	11.3	1
4	I	486	13	4.100e+04	-4.562e+04	-1.1		0.5	37.4	60.0	69.1	112.4	1
5	SO	116	25	-5.084e+04	-4.452e+04								
7	SO	116	25	-5.448e+04	-6.080e+04								

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

**Lateral Torsional Buckling check**

According to EN 1993-1-1 article 6.3.2.1 & 6.3.2.3 and formula (6.54)

LTB parameters		
Method for LTB curve		Alternative case
Plastic section modulus	$W_{pl,y}$	5.3333e-03 m <sup>3</sup>
Elastic critical moment	$M_{cr}$	949.28 kNm
Relative slenderness	$\lambda_{rel,LT}$	1.41
Limit slenderness	$\lambda_{rel,LT,0}$	0.40
LTB curve		b
Imperfection	$\alpha_{LT}$	0.34
LTB factor	$\beta$	0.75
Reduction factor	$\chi_{LT}$	0.47
Correction factor	$k_c$	0.88
Correction factor	$f$	0.99
Modified reduction factor	$\chi_{LT,mod}$	0.47
Design buckling resistance	$M_{b,Rd}$	897.39 kNm
Unity check		0.28

Mcr parameters		
LTB length	$l_{LT}$	13.500 m
Influence of load position		no influence
Correction factor	$k$	1.00
Correction factor	$k_w$	1.00
LTB moment factor	$C_1$	1.29
LTB moment factor	$C_2$	0.45
LTB moment factor	$C_3$	0.41
Shear centre distance	$d_z$	0 mm
Distance of load application	$z_g$	0 mm
Mono-symmetry constant	$\beta_y$	0 mm
Mono-symmetry constant	$z_i$	0 mm

$$M_{cr} = C_1 \times \frac{\pi^2 \times E \times I_x}{l_{LT}^2} \times \left[ \left( \frac{k}{k_w} \right)^2 \times \frac{I_w}{I_x} + \frac{I_T^2 \times G \times I_x}{\pi^2 \times E \times I_x} + (C_2 \times z_g - C_3 \times z_i)^2 - (C_2 \times z_g - C_3 \times z_i) \right] = 1.29 \times \frac{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]}{13.500[\text{m}]^2} \times \left[ \left( \frac{1.00}{1.00} \right)^2 \times \frac{8.9782 \cdot 10^{-6}[\text{m}^6]}{1.1300 \cdot 10^{-4}[\text{m}^4]} + \frac{13.500[\text{m}]^2 \times 80769.2[\text{MPa}] \times 3.9800 \cdot 10^{-6}[\text{m}^4]}{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]} + (0.45 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}])^2 - (0.45 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}]) \right] = 949.28[\text{kNm}]$$

$$\lambda_{rel,LT} = \sqrt{\frac{W_{pl,y} \times f_y}{M_{cr}}} = \sqrt{\frac{5.3333 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{949.28[\text{kNm}]} = 1.41$$

$$\beta = 0.75$$

$$\chi_{LT} = \min \left( \frac{1}{\varphi_{LT} + \sqrt{\varphi_{LT}^2 - \beta \times \lambda_{rel,LT}^2}}, \frac{1}{\lambda_{rel,LT}^2} \cdot 1 \right) = \min \left( \frac{1}{1.42 + \sqrt{1.42^2 - 0.75 \times 1.41^2}}, \frac{1}{1.41^2} \cdot 1 \right) = \min(0.47, 0.50, 1) = 0.47 \quad (\text{EC3-1-1: 6.57})$$

$$f = \min \{ 1 - 0.5 \times (1 - k_c) \times [1 - 2 \times (\lambda_{rel,LT} - 0.8)^2], 1 \} = \min \{ 1 - 0.5 \times (1 - 0.88) \times [1 - 2 \times (1.41 - 0.8)^2], 1 \} = \min \{ 0.99, 1 \} = 0.99$$

$$\chi_{LT,mod} = \min \left( \frac{\chi_{LT}}{f}, 1 \right) = \min \left( \frac{0.47}{0.99}, 1 \right) = \min(0.47, 1) = 0.47$$

$$M_{b,Rd} = \chi_{LT,mod} \times W_{pl,y} \times \frac{f_y}{\gamma_{M1}} = 0.47 \times 5.3333 \cdot 10^{-3}[\text{m}^3] \times \frac{355.0[\text{MPa}]}{1.00} = 897.39[\text{kNm}] \quad (\text{EC3-1-1: 6.55})$$

Mcr parameters			
LTB length	$l_{LT}$	13.500	m
Influence of load position		no influence	
Correction factor	$k$	1.00	
Correction factor	$k_w$	1.00	
LTB moment factor	$C_1$	1.29	
LTB moment factor	$C_2$	0.45	
LTB moment factor	$C_3$	0.41	
Shear centre distance	$d_z$	0	mm
Distance of load application	$z_g$	0	mm
Mono-symmetry constant	$\beta_y$	0	mm
Mono-symmetry constant	$z_j$	0	mm

$$M_{Cr} = C_1 \times \frac{\pi^2 \times E \times I_z}{l_{LT}^2} \times \left[ \sqrt{\left(\frac{k}{k_w}\right)^2 \times \frac{I_w}{I_z} + \frac{I_T^2 \times G \times I_z}{\pi^2 \times E \times I_z} + (C_2 \times z_g - C_3 \times z_j)^2} - (C_2 \times z_g - C_3 \times z_j) \right] = 1.29$$

$$\times \frac{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]}{13.500[\text{m}]^2}$$

$$\times \left[ \sqrt{\left(\frac{1.00}{1.00}\right)^2 \times \frac{8.9782 \cdot 10^{-6}[\text{m}^6]}{1.1300 \cdot 10^{-4}[\text{m}^4]} + \frac{13.500[\text{m}]^2 \times 80769.2[\text{MPa}] \times 3.9800 \cdot 10^{-6}[\text{m}^4]}{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]} + (0.45 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}])^2} - (0.45 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}]) \right]$$

$$= 949.28[\text{kNm}]$$

$$\lambda_{rel,LT} = \sqrt{\frac{W_{pl,y} \times f_y}{M_{Cr}}} = \sqrt{\frac{5.3333 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{949.28[\text{kNm}]} = 1.41$$

$$\beta = 0.75$$

$$\chi_{LT} = \min \left( \frac{1}{\varphi_{LT} + \sqrt{\varphi_{LT}^2 - \beta \times \lambda_{rel,LT}^2}}, \frac{1}{\lambda_{rel,LT}^2}, 1 \right) = \min \left( \frac{1}{1.42 + \sqrt{1.42^2 - 0.75 \times 1.41^2}}, \frac{1}{1.41^2}, 1 \right) = \min(0.47, 0.50, 1) = 0.47 \quad (\text{EC3-1-1: 6.57})$$

$$f = \min \left\{ 1 - 0.5 \times (1 - k_c) \times \left[ 1 - 2 \times (\lambda_{rel,LT} - 0.8)^2 \right], 1 \right\} = \min \left\{ 1 - 0.5 \times (1 - 0.88) \times \left[ 1 - 2 \times (1.41 - 0.8)^2 \right], 1 \right\} = \min \{0.99, 1\} = 0.99$$

$$\chi_{LT,mod} = \min \left( \frac{\chi_{LT}}{f}, 1 \right) = \min \left( \frac{0.47}{0.99}, 1 \right) = \min(0.47, 1) = 0.47$$

$$M_{b,Rd} = \chi_{LT,mod} \times W_{pl,y} \times \frac{f_y}{\gamma_{M1}} = 0.47 \times 5.3333 \cdot 10^{-3}[\text{m}^3] \times \frac{355.0[\text{MPa}]}{1.00} = 897.39[\text{kNm}] \quad (\text{EC3-1-1: 6.55})$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{b,Rd}} = \frac{|-251.70[\text{kNm}]|}{897.39[\text{kNm}]} = 0.28 \leq 1.00 \quad (\text{EC3-1-1: 6.54})$$

**Note:** C parameters are determined according to ECCS 119 2006 / Galea 2002.

**Note:** The correction factor  $k_c$  is determined from  $C_1$ .

#### Bending and axial tension check

According to EN 1993-1-3 article 6.3

Normal force	$N_{Ed}$	52.34	kN
Bending moment	$M_{y,Ed}$	-251.70	kNm
Bending moment	$M_{z,Ed}$	6.12	kNm
Tension resistance	$N_{t,Rd}$	8008.56	kN
Bending resistance	$M_{b,y,Rd}$	897.39	kNm
Bending resistance	$M_{c,z,Rd,com}$	409.73	kNm

$$\text{Unity check} = 0.28 + 0.01 - 0.01 = 0.29$$

$$N_{t,Rd} = \min(N_{pl,Rd}, N_{u,Rd}) = \min(8058.50[\text{kN}], 8008.56[\text{kN}]) = 8008.56[\text{kN}]$$

$$M_{b,y,Rd} = \chi_{LT,mod} \times W_{pl,y} \times \frac{f_y}{\gamma_{M1}} = 0.47 \times 5.3333 \cdot 10^{-3}[\text{m}^3] \times \frac{355.0[\text{MPa}]}{1.00} = 897.39[\text{kNm}]$$

$$M_{c,z,Rd,com} = \frac{W_{el,z,com} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{1.00} = 409.73[\text{kNm}]$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{b,y,Rd}} + \frac{|M_{z,Ed}|}{M_{c,z,Rd,com}} - \frac{|N_{Ed}|}{N_{t,Rd}} = \frac{|-251.70[\text{kNm}]|}{897.39[\text{kNm}]} + \frac{|6.12[\text{kNm}]|}{409.73[\text{kNm}]} - \frac{|52.34[\text{kN}]|}{8008.56[\text{kN}]} = 0.29 \leq 1.00$$

#### Shear Buckling check

According to EN 1993-1-5 article 5 & 7.1 and formula (5.10) & (7.1)

Shear Buckling parameters			
Buckling field length	$a$	6.000	m
Web		unstiffened	
Web height	$h_w$	540	mm
Web thickness	$t$	13	mm
Material coefficient	$\epsilon$	0.81	
Shear correction factor	$\eta$	1.20	

Shear Buckling verification		
Web slenderness	$h_w/t$	41.54
Web slenderness limit		48.82

**Note:** The web slenderness is such that Shear Buckling effects may be ignored according to EN 1993-1-5 article 5.1(2).

The member satisfies the stability check.



**EC-EN 1993 Steel check ULS**

Values:  $U_{Coverall}$   
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: Bx8\_1

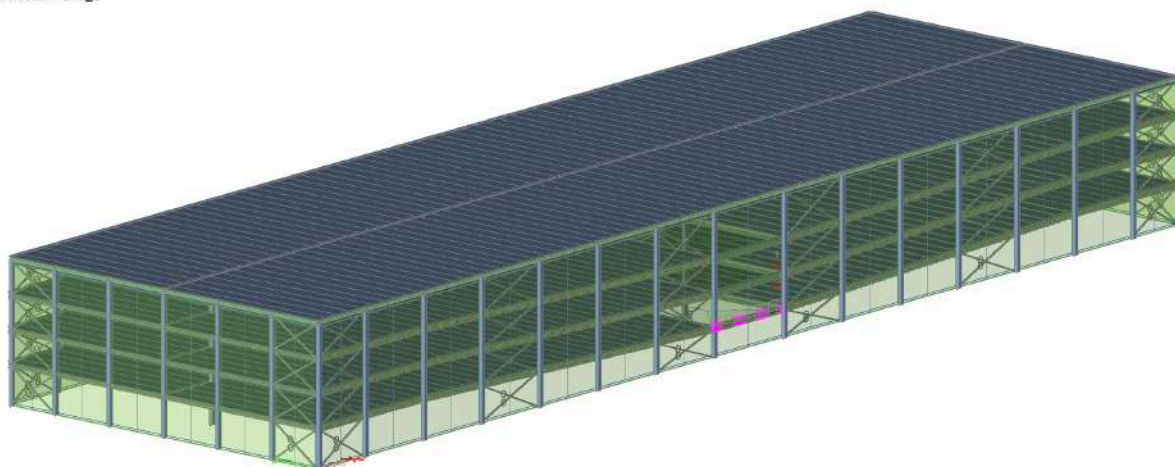


Figure 2.2.3.3. 4: Middle beam

**EC-EN 1993 Steel check ULS**

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: Bx8\_1  
 Filter: Cross-section = CS12-Beam2 - HEA600

**EN 1993-1-1 Code Check**

National annex: Czech CSN-EN NA

<b>Member Bx8_1</b>	<b>0.000 / 9.000 m</b>	<b>HEA600</b>	<b>S 355</b>	<b>ULS-Set B (auto)</b>	<b>0.10 -</b>
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<b>Combination key</b>	
ULS-Set B (auto) / 1.15*LC1 + 1.15*LC2 + 1.05*LC3 + 1.05*LC4 + 1.50*3DWind4	

<b>Partial safety factors</b>	
$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

<b>Material</b>			
Yield strength	$f_y$	355.0	MPa
Ultimate strength	$f_u$	490.0	MPa
Fabrication		Rolled	

....:SECTION CHECK:....

The critical check is on position 0.000 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	-42.50	kN
Shear force	$V_{y,Ed}$	-0.13	kN
Shear force	$V_{z,Ed}$	72.27	kN
Torsion	$T_{Ed}$	0.00	kNm
Bending moment	$M_{y,Ed}$	-167.83	kNm
Bending moment	$M_{z,Ed}$	-0.48	kNm

### Classification for cross-section design

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993 1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_p$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	3.559e+04	3.609e+04	1.0	0.4	1.0	4.7	7.3	8.1	11.2	1
3	SO	116	25	3.531e+04	3.481e+04	1.0	0.4	1.0	4.7	7.3	8.1	11.3	1
4	I	486	13	3.075e+04	-2.700e+04	-0.9		0.5	37.4	56.9	65.7	88.8	1
5	SO	116	25	-3.184e+04	-3.233e+04								
7	SO	116	25	-3.155e+04	-3.106e+04								

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

### Compression check

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	2.2700e-02	m <sup>2</sup>
Compression resistance	$N_{c,Rd}$	8058.50	kN
Unity check		0.01	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{2.2700 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 8058.50 [\text{kN}] \quad (\text{EC3-1-1: 6.10})$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-42.50 [\text{kN}]|}{8058.50 [\text{kN}]} = 0.01 \leq 1.00 \quad (\text{EC3-1-1: 6.9})$$

### Bending moment check for $M_y$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	5.3333e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Unity check		0.09	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 1893.33 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|-167.83 [\text{kNm}]|}{1893.33 [\text{kNm}]} = 0.09 \leq 1.00 \quad (\text{EC3-1-1: 6.12})$$

### Bending moment check for $M_z$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,z}$	1.1542e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,z,Rd}$	409.73	kNm
Unity check		0.00	-

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 409.73 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\text{Unity check} = \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} = \frac{|-0.48 [\text{kNm}]|}{409.73 [\text{kNm}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.12})$$

### Shear check for $V_y$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	1.5520e-02	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	3180.97	kN
Unity check		0.00	-

$$V_{pl,y,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{1.5520 \cdot 10^{-2} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 3180.97 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{y,Ed}|}{V_{c,y,Rd}} = \frac{|-0.13 [\text{kN}]|}{3180.97 [\text{kN}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Shear check for $V_z$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.3750e-03	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	1921.49	kN
Unity check		0.04	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.3750 \cdot 10^{-3} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 1921.49 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{c,z,Rd}} = \frac{|72.27 [\text{kN}]|}{1921.49 [\text{kN}]} = 0.04 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	2	
Total torsional moment	$T_{Ed}$	0.0	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.00	-

$$\tau_{Ed} = |T_{Ed}| \times \tau_{Ed,mit} = |3.73| \times 6.284 [\text{kN/m}^2] = 0.0 [\text{MPa}]$$

$$\tau_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [\text{MPa}]}{\sqrt{3} \times 1.00} = 205.0 [\text{MPa}]$$

$$\text{Unity check} = \frac{\tau_{Ed}}{\tau_{Rd}} = \frac{0.0 [\text{MPa}]}{205.0 [\text{MPa}]} = 0.00 \leq 1.00$$

(EC3-1-1: 6.23)

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

#### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.41)

Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Exponent of bending ratio y	$\alpha$	2.00	
Plastic bending moment	$M_{pl,z,Rd}$	409.73	kNm
Exponent of bending ratio z	$\beta$	1.00	

$$\text{Unity check (6.41)} = 0.01 + 0.00 = 0.01$$

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 1893.33 [\text{kNm}]$$

(EC3-1-1: 6.13)

$$\alpha = 2.00$$

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{1.1542 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 409.73 [\text{kNm}]$$

(EC3-1-1: 6.13)

$$\beta = 1.00$$

$$\text{Unity check} = \left( \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} \right)^\alpha + \left( \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} \right)^\beta = \left( \frac{|-167.83 [\text{kNm}]|}{1893.33 [\text{kNm}]} \right)^{2.00} + \left( \frac{|-0.48 [\text{kNm}]|}{409.73 [\text{kNm}]} \right)^{1.00} = 0.01 \leq 1.00$$

(EC3-1-1: 6.41)

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

**Note:** Since the axial force satisfies both criteria (6.33) and (6.34) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the y-y axis is neglected.

**Note:** Since the axial force satisfies criteria (6.35) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the z-z axis is neglected.

The member satisfies the section check.

#### ....:STABILITY CHECK:....

##### Classification for member buckling design

Decisive position for stability classification: 7.200 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_\sigma$ [-]	$\alpha$ [-]	c/t [-]	Class 1	Class 2	Class 3	Class
										Limit [-]	Limit [-]	Limit [-]	
1	SO	116	25	2.449e+03	3.925e+03	0.6	0.5	1.0	4.7	7.3	8.1	11.7	1
3	SO	116	25	1.600e+03	1.231e+02	0.1	1.4	1.0	4.7	7.3	8.1	20.1	1
4	I	486	13	2.004e+03	1.750e+03	0.9		1.0	37.4	22.8	27.7	32.3	4
5	SO	116	25	1.304e+03	-1.722e+02	-0.1	2.7	0.9	4.7	8.8	9.8	27.9	1
7	SO	116	25	2.154e+03	3.630e+03	0.6	0.5	1.0	4.7	7.3	8.1	11.7	1

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 4

##### Effective section N-

##### Effective width calculation

According to EN 1993-1-5 article 4.4

Id	Type	$b_p$ [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_\sigma$ [-]	$\lambda_p$ [-]	$\rho$ [-]	$b_e$	$b_{e1}$	$b_{e2}$
									[mm]	[mm]	[mm]
1	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
3	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
4	I	486	3.550e+05	3.550e+05	1.0	4.0	0.8	0.9	437	219	219
5	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
7	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		

##### Effective section My-

##### Effective width calculation

According to EN 1993-1-5 article 4.4

Id	Type	$b_p$ [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_\sigma$ [-]	$\lambda_p$ [-]	$\rho$ [-]	$b_e$	$b_{e1}$	$b_{e2}$
									[mm]	[mm]	[mm]
1	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
3	SO	116	3.550e+05	3.550e+05	1.0	0.4	0.3	1.0	116		
4	I	486	3.054e+05	-3.054e+05	-1.0	23.9	0.3	1.0	243	97	146
5	SO	116	-3.550e+05	-3.550e+05							
7	SO	116	-3.550e+05	-3.550e+05							

### Effective section Mz-

### Effective width calculation

According to EN 1993-1-5 article 4.4

Id	Type	b <sub>p</sub> [mm]	σ <sub>1</sub> [kN/m <sup>2</sup> ]	σ <sub>2</sub> [kN/m <sup>2</sup> ]	ψ [-]	k <sub>σ</sub> [-]	λ <sub>p</sub> [-]	ρ [-]	b <sub>e</sub> [mm]	b <sub>e1</sub> [mm]	b <sub>e2</sub> [mm]
1	SO	116	3.550e+05	7.928e+04	0.2	0.5	0.3	1.0	116		
3	SO	116	-7.928e+04	-3.550e+05							
4	I	486	0.000e+00	0.000e+00							
5	SO	116	-7.928e+04	-3.550e+05							
7	SO	116	3.550e+05	7.928e+04	0.2	0.5	0.3	1.0	116		

Effective properties						
Effective area	A <sub>eff</sub>	2.2017e-02	m <sup>2</sup>			
Effective second moment of area	I <sub>eff,y</sub>	1.4123e-03	m <sup>4</sup>	I <sub>eff,z</sub>	1.1271e-04	m <sup>4</sup>
Effective section modulus	W <sub>eff,y</sub>	4.7874e-03	m <sup>3</sup>	W <sub>eff,z</sub>	7.5143e-04	m <sup>3</sup>
Shift of the centroid	e <sub>N,y</sub>	0	mm	e <sub>N,z</sub>	0	mm

### Flexural Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters	yy	zz	
Sway type	sway	non-sway	
System length	L	9.000	m
Buckling factor	k	1.26	0.77
Buckling length	l <sub>cr</sub>	11.335	6.934
Critical Euler load	N <sub>cr</sub>	22745.32	4871.29
Slenderness	λ	45.48	98.28
Relative slenderness	λ <sub>rel</sub>	0.59	1.27
Limit slenderness	λ <sub>rel,0</sub>	0.20	0.20

**Note:** The slenderness or compression force is such that Flexural Buckling effects may be ignored according to EN 1993-1-1 article 6.3.1.2(4).

$$N_{cr,y} = \frac{\pi^2 \times E \times I_{cr,y}}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 1.4100 \cdot 10^{-3}[\text{m}^4]}{11.335[\text{m}]^2} = 22745.32[\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_{cr,z}}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]}{6.934[\text{m}]^2} = 4871.29[\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{11.335[\text{m}]}{249[\text{mm}]} = 45.48$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{6.934[\text{m}]}{71[\text{mm}]} = 98.28$$

$$\lambda_{rel,y} = \frac{\lambda_y \times \sqrt{\frac{A_{eff}}{A}}}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{45.48 \times \sqrt{\frac{2.2017 \cdot 10^{-2}[\text{m}^2]}{2.2700 \cdot 10^{-2}[\text{m}^2]}}}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{355.0[\text{MPa}]}}} = 0.59$$

(EC3-1-1: 6.51)

$$\lambda_{rel,z} = \frac{\lambda_z \times \sqrt{\frac{A_{eff}}{A}}}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{98.28 \times \sqrt{\frac{2.2017 \cdot 10^{-2}[\text{m}^2]}{2.2700 \cdot 10^{-2}[\text{m}^2]}}}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{355.0[\text{MPa}]}}} = 1.27$$

(EC3-1-1: 6.51)

### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** For this I-section the Torsional(-Flexural) buckling resistance is higher than the resistance for Flexural buckling. Therefore Torsional(-Flexural) buckling is not printed on the output.

### Lateral Torsional Buckling check

According to EN 1993-1-1 article 6.3.2.1 & 6.3.2.3 and formula (6.54)

LTB parameters			
Method for LTB curve		Alternative case	
Effective section modulus	W <sub>eff,y</sub>	4.7874e-03	m <sup>3</sup>
Elastic critical moment	M <sub>cr</sub>	4734.28	kNm
Relative slenderness	λ <sub>rel,LT</sub>	0.60	
Limit slenderness	λ <sub>rel,LT,0</sub>	0.40	

**Note:** The slenderness or bending moment is such that Lateral Torsional Buckling effects may be ignored according to EN 1993-1-1 article 6.3.2.2(4).

Mcr parameters			
LTB length	$l_{LT}$	9.000	m
Influence of load position		no influence	
Correction factor	$k$	1.00	
Correction factor	$k_w$	1.00	
LTB moment factor	$C_1$	3.75	
LTB moment factor	$C_2$	1.66	
LTB moment factor	$C_3$	0.41	
Shear centre distance	$d_z$	0	mm
Distance of load application	$z_g$	0	mm
Mono-symmetry constant	$\beta_y$	0	mm
Mono-symmetry constant	$z_j$	0	mm

$$M_{cr} = C_1 \times \frac{\pi^2 \times E \times I_z}{l_{LT}^2} \times \left[ \sqrt{\left(\frac{k}{k_w}\right)^2 \times \frac{I_w}{I_z} + \frac{I_{LT}^2 \times G \times I_z}{\pi^2 \times E \times I_z} + (C_2 \times z_g - C_3 \times z_j)^2} - (C_2 \times z_g - C_3 \times z_j) \right] = 3.75$$

$$\times \frac{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]}{9.000[\text{m}]^2}$$

$$\times \left[ \sqrt{\left(\frac{1.00}{1.00}\right)^2 \times \frac{8.9782 \cdot 10^{-6}[\text{m}^6]}{1.1300 \cdot 10^{-4}[\text{m}^4]} + \frac{9.000[\text{m}]^2 \times 80769.2[\text{MPa}] \times 3.9800 \cdot 10^{-6}[\text{m}^2]}{\pi^2 \times 210000.0[\text{MPa}] \times 1.1300 \cdot 10^{-4}[\text{m}^4]} + (1.66 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}])^2} - (1.66 \times 0[\text{mm}] - 0.41 \times 0[\text{mm}]) \right]$$

$$= 4734.28[\text{kNm}]$$

$$\lambda_{rel,LT} = \sqrt{\frac{W_{eff,y} \times f_y}{M_{cr}}} = \sqrt{\frac{4.7874 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{4734.28[\text{kNm}]} = 0.60$$

**Note:** C parameters are determined according to ECCS 119 2006 / Galea 2002.

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

Bending and axial compression check parameters			
Interaction method		alternative method 2	
Cross-section effective area	$A_{eff}$	2.2017e-02	m <sup>2</sup>
Effective section modulus	$W_{eff,y}$	4.7874e-03	m <sup>3</sup>
Effective section modulus	$W_{eff,z}$	7.5143e-04	m <sup>3</sup>
Design compression force	$N_{Ed}$	42.50	kN
Design bending moment (maximum)	$M_{y,Ed}$	-167.83	kNm
Design bending moment (maximum)	$M_{z,Ed}$	-1.67	kNm
Additional moment	$\Delta M_{y,Ed}$	0.00	kNm
Additional moment	$\Delta M_{z,Ed}$	0.00	kNm
Characteristic compression resistance	$N_{Rk}$	7815.95	kN
Characteristic moment resistance	$M_{y,Rk}$	1699.52	kNm
Characteristic moment resistance	$M_{z,Rk}$	266.76	kNm

Bending and axial compression check parameters			
Reduction factor	$\chi_y$	1.00	
Reduction factor	$\chi_z$	1.00	
Modified reduction factor	$\chi_{LT,mod}$	1.00	
Interaction factor	$k_{yy}$	0.90	
Interaction factor	$k_{yz}$	0.72	
Interaction factor	$k_{zy}$	0.72	
Interaction factor	$k_{zz}$	0.72	

Maximum moment  $M_{y,Ed}$  is derived from beam Bx8\_1 position 0.000 m.  
Maximum moment  $M_{z,Ed}$  is derived from beam Bx8\_1 position 9.000 m.

Interaction method 2 parameters			
Method for interaction factors		Table B.1	
Sway type y		sway	
Equivalent moment factor	$C_{my}$	0.90	
Resulting load type z		linear moment M	
Ratio of end moments	$\psi_z$	0.29	
Equivalent moment factor	$C_{mz}$	0.72	
Resulting load type LT		line load q	
End moment	$M_{h,LT}$	-167.83	kNm
Field moment	$M_{s,LT}$	54.85	kNm
Factor	$\alpha_{s,LT}$	-0.33	
Ratio of end moments	$\psi_{LT}$	0.55	
Equivalent moment factor	$C_{mLT}$	0.40	

$$\text{Unity check (6.61)} = 0.01 + 0.09 + 0.00 = 0.10 -$$

$$\text{Unity check (6.62)} = 0.01 + 0.07 + 0.00 = 0.08 -$$

$$C_{my} = 0.90$$

$$C_{mz} = \max(0.6 + 0.4 \times \psi_z, 0.4) = \max(0.6 + 0.4 \times 0.29, 0.4) = \max(0.72, 0.4) = 0.72$$

$$\alpha_{s,LT} = \frac{M_{s,LT}}{M_{h,LT}} = \frac{54.85[\text{kNm}]}{-167.83[\text{kNm}]} = -0.33$$

$$C_{mLT} = \max(0.1 - 0.8 \times \alpha_{s,LT}, 0.4) = \max(0.1 - 0.8 \times -0.33, 0.4) = \max(0.36, 0.4) = 0.40$$

$$N_{Rk} = A_{eff} \times f_y = 2.2017 \cdot 10^{-2}[\text{m}^2] \times 355.0[\text{MPa}] = 7815.95[\text{kN}]$$

$$M_{y,Rk} = W_{eff,y} \times f_y = 4.7874 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}] = 1699.52[\text{kNm}]$$

$$M_{z,Rk} = W_{eff,z} \times f_y = 7.5143 \cdot 10^{-4}[\text{m}^3] \times 355.0[\text{MPa}] = 266.76[\text{kNm}]$$

$$k_{yy} = \min \left[ C_{my} \times \left( 1 + 0.6 \times \lambda_{rel,y} \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right), C_{my} \times \left( 1 + 0.6 \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right]$$

$$= \min \left[ 0.90 \times \left( 1 + 0.6 \times 0.59 \times \frac{42.50[\text{kN}]}{1.00 \times \frac{7815.95[\text{kN}]}{1.00}} \right), 0.90 \times \left( 1 + 0.6 \times \frac{42.50[\text{kN}]}{1.00 \times \frac{7815.95[\text{kN}]}{1.00}} \right) \right] = \min [0.90, 0.90] = 0.90$$

$$k_{yz} = k_{zz} = 0.72$$

$$k_{zy} = 0.8 \times k_{yy} = 0.8 \times 0.90 = 0.72$$

$$k_{zz} = \min \left[ C_{mz} \times \left( 1 + 0.6 \times \lambda_{rel,z} \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right), C_{mz} \times \left( 1 + 0.6 \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right]$$

$$= \min \left[ 0.72 \times \left( 1 + 0.6 \times 1.27 \times \frac{42.50[\text{kN}]}{1.00 \times \frac{7815.95[\text{kN}]}{1.00}} \right), 0.72 \times \left( 1 + 0.6 \times \frac{42.50[\text{kN}]}{1.00 \times \frac{7815.95[\text{kN}]}{1.00}} \right) \right] = \min [0.72, 0.72] = 0.72$$

$$\text{Unity check (6.61)} = \frac{|N_{Ed}|}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{yy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\lambda_{LT,mod} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{yz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{42.50[\text{kN}]}{1.00 \times \frac{7815.95[\text{kN}]}{1.00}} + 0.90 \times \frac{|-167.83[\text{kNm}]| + |0.00[\text{kNm}]|}{1.00 \times \frac{1699.52[\text{kNm}]}{1.00}} + 0.72 \times \frac{|-1.67[\text{kNm}]| + |0.00[\text{kNm}]|}{\frac{266.76[\text{kNm}]}{1.00}} = \mathbf{0.10} \leq \mathbf{1.00}$$

(EC3-1-1: 6.61)

$$\text{Unity check (6.62)} = \frac{|N_{Ed}|}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\lambda_{LT,mod} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{42.50[\text{kN}]}{1.00 \times \frac{7815.95[\text{kN}]}{1.00}} + 0.72 \times \frac{|-167.83[\text{kNm}]| + |0.00[\text{kNm}]|}{1.00 \times \frac{1699.52[\text{kNm}]}{1.00}} + 0.72 \times \frac{|-1.67[\text{kNm}]| + |0.00[\text{kNm}]|}{\frac{266.76[\text{kNm}]}{1.00}} = \mathbf{0.08} \leq \mathbf{1.00}$$

(EC3-1-1: 6.62)

$$\text{Unity check} = \max(\text{Unity check (6.61), Unity check (6.62)}) = \max(0.10, 0.08) = \mathbf{0.10} \leq \mathbf{1.00}$$

#### Shear Buckling check

According to EN 1993-1-5 article 5 & 7.1 and formula (5.10) & (7.1)

Shear Buckling parameters			
Buckling field length	a	9.000	m
Web		unstiffened	
Web height	$h_w$	540	mm
Web thickness	t	13	mm
Material coefficient	$\varepsilon$	0.81	
Shear correction factor	$\eta$	1.20	

Shear Buckling verification		
Web slenderness	$h_w/t$	41.54
Web slenderness limit		48.82

$$h_w/t = \frac{h_w}{t} = \frac{540[\text{mm}]}{13[\text{mm}]} = 41.54$$

$$\text{limit } h_w/t = \frac{72 \times \varepsilon}{\eta} = \frac{72 \times 0.81}{1.20} = 48.82$$

**Note:** The web slenderness is such that Shear Buckling effects may be ignored according to EN 1993-1-5 article 5.1(2).

The member satisfies the stability check.

**EC-EN 1993 Steel check ULS**  
 Values: **UC<sub>overall</sub>**  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: Bx8\_1

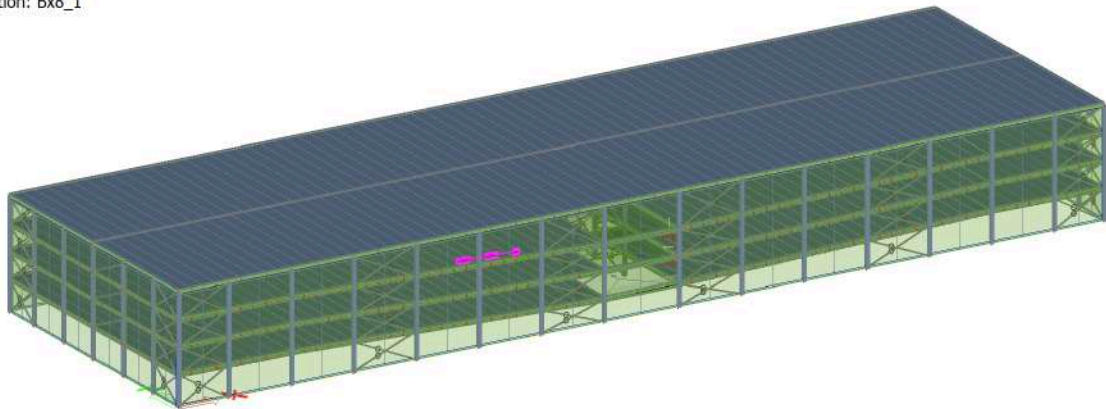


Figure 2.2.3.3. 5: Interior beam

**EC-EN 1993 Steel check ULS**

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: Bx22\_1  
 Filter: Cross-section = CS12-Beam2 - HEA600

**EN 1993-1-1 Code Check**

National annex: Czech CSN-EN NA

<b>Member Bx22_1</b>	<b>3.752 / 7.503 m</b>	<b>HEA600</b>	<b>S 355</b>	<b>ULS-Set B (auto)</b>	<b>0.30 -</b>
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<b>Combination key</b>
ULS-Set B (auto) / 1.15*LC1 + 1.15*LC2 + 1.50*LC3 + 1.50*LC4 + 0.90*3DWind12

<b>Partial safety factors</b>	
$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

<b>Material</b>		
Yield strength $f_y$	355.0	MPa
Ultimate strength $f_u$	490.0	MPa
Fabrication	Rolled	

....:SECTION CHECK:....

The critical check is on position 3.752 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	51.59	kN
Shear force	$V_{y,Ed}$	0.00	kN
Shear force	$V_{z,Ed}$	0.00	kN
Torsion	$T_{Ed}$	-0.18	kNm
Bending moment	$M_{y,Ed}$	405.05	kNm
Bending moment	$M_{z,Ed}$	0.00	kNm

The critical check is on position 3.752 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	51.59	kN
Shear force	$V_{y,Ed}$	0.00	kN
Shear force	$V_{z,Ed}$	0.00	kN
Torsion	$T_{Ed}$	-0.18	kNm
Bending moment	$M_{y,Ed}$	405.05	kNm
Bending moment	$M_{z,Ed}$	0.00	kNm

#### Classification for cross-section design

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_{tr}$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	-8.330e+04	-8.330e+04								
3	SO	116	25	-8.330e+04	-8.330e+04								
4	I	486	13	-7.197e+04	6.742e+04	-1.1		0.5	37.4	60.0	69.1	107.8	1
5	SO	116	25	7.875e+04	7.875e+04	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1
7	SO	116	25	7.875e+04	7.875e+04	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

#### Tension check

According to EN 1993-1-1 article 6.2.3 and formula (6.5)

Cross-section area	A	2.2700e-02	m <sup>2</sup>
Plastic tension resistance	$N_{pl,Rd}$	8058.50	kN
Ultimate tension resistance	$N_{u,Rd}$	8008.56	kN
Tension resistance	$N_{t,Rd}$	8008.56	kN
Unity check		0.01	-

$$N_{pl,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{2.2700 \cdot 10^{-2} [m^2] \times 355.0 [MPa]}{1.00} = 8058.50 [kN] \quad (EC3-1-1: 6.6)$$

$$N_{u,Rd} = \frac{0.9 \times A \times f_u}{\gamma_{M2}} = \frac{0.9 \times 2.2700 \cdot 10^{-2} [m^2] \times 490.0 [MPa]}{1.25} = 8008.56 [kN] \quad (EC3-1-1: 6.7)$$

$$N_{t,Rd} = \min(N_{pl,Rd}, N_{u,Rd}) = \min(8058.50 [kN], 8008.56 [kN]) = 8008.56 [kN]$$

$$\text{Unity check} = \frac{N_{Ed}}{N_{t,Rd}} = \frac{51.59 [kN]}{8008.56 [kN]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.5)$$

#### Bending moment check for $M_y$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	5.3333e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Unity check		0.21	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 1893.33 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{405.05 [kNm]}{1893.33 [kNm]} = 0.21 \leq 1.00 \quad (EC3-1-1: 6.12)$$

#### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	2	
Total torsional moment	$T_{Ed}$	1.1	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.01	-

$$\tau_{Ed} = |T_{Ed}| \times \tau_{Ed,unit} = |-181.66| \times 6.284 [kN/m^2] = 1.1 [MPa]$$

$$\tau_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [MPa]}{\sqrt{3} \times 1.00} = 205.0 [MPa]$$

$$\text{Unity check} = \frac{\tau_{Ed}}{\tau_{Rd}} = \frac{1.1 [MPa]}{205.0 [MPa]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.23)$$

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

#### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.31)

Plastic bending moment	$M_{pl,y,Rd}$	1893.33	kNm
Unity check		0.21	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{5.3333 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 1893.33 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{405.05 [kNm]}{1893.33 [kNm]} = 0.21 \leq 1.00 \quad (EC3-1-1: 6.31)$$

**Note:** Since the axial force satisfies both criteria (6.33) and (6.34) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the y-y axis is neglected.

The member satisfies the section check.



....:STABILITY CHECK:....

**Classification for member buckling design**

Decisive position for stability classification: 3.752 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_b$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	116	25	-8.330e+04	-8.330e+04								
3	SO	116	25	-8.330e+04	-8.330e+04								
4	I	486	13	-7.197e+04	6.742e+04	-1.1		0.5	37.4	60.0	69.1	107.8	1
5	SO	116	25	7.875e+04	7.875e+04	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1
7	SO	116	25	7.875e+04	7.875e+04	1.0	0.4	1.0	4.7	7.3	8.1	11.4	1

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

**Lateral Torsional Buckling check**

According to EN 1993-1-1 article 6.3.2.1 & 6.3.2.3 and formula (6.54)

LTB parameters			
Method for LTB curve		Alternative case	
Plastic section modulus	$W_{pl,y}$	5.3333e-03	m <sup>3</sup>
Elastic critical moment	$M_{cr}$	1856.08	kNm
Relative slenderness	$\lambda_{rel,LT}$	1.01	
Limit slenderness	$\lambda_{rel,LT,0}$	0.40	
LTB curve		b	
Imperfection	$\alpha_{LT}$	0.34	
LTB factor	$\beta$	0.75	
Reduction factor	$\chi_{LT}$	0.69	
Correction factor	$k_c$	0.94	
Correction factor	$f$	0.97	
Modified reduction factor	$\chi_{LT,mod}$	0.71	
Design buckling resistance	$M_{b,Rd}$	1348.82	kNm
Unity check		0.30	-

Mcr parameters			
LTB length	$l_{LT}$	7.503	m
Influence of load position		no influence	
Correction factor	$k$	1.00	
Correction factor	$k_w$	1.00	
LTB moment factor	$C_1$	1.13	
LTB moment factor	$C_2$	0.45	
LTB moment factor	$C_3$	0.53	
Shear centre distance	$c_z$	0	mm
Distance of load application	$z_g$	0	mm
Mono-symmetry constant	$\beta_y$	0	mm
Mono-symmetry constant	$z_j$	0	mm

$$M_{cr} = C_1 \times \frac{\pi^2 \times E \times I_z}{l_{LT}^2} \times \left[ \sqrt{\left( \frac{k}{k_w} \right)^2 \times \frac{I_w}{I_z} + \frac{I_{Tz}^2 \times G \times I_t}{\pi^2 \times E \times I_z} + (C_2 \times z_g - C_3 \times z_j)^2} - (C_2 \times z_g - C_3 \times z_j) \right] = 1.13$$

$$\times \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 1.1300 \cdot 10^{-4} [\text{m}^4]}{7.503 [\text{m}]^2}$$

$$\times \left[ \sqrt{\left( \frac{1.00}{1.00} \right)^2 \times \frac{8.9782 \cdot 10^{-6} [\text{m}^6]}{1.1300 \cdot 10^{-4} [\text{m}^4]} + \frac{7.503 [\text{m}]^2 \times 80769.2 [\text{MPa}] \times 3.9800 \cdot 10^{-6} [\text{m}^4]}{\pi^2 \times 210000.0 [\text{MPa}] \times 1.1300 \cdot 10^{-4} [\text{m}^4]} + (0.45 \times 0 [\text{mm}] - 0.53 \times 0 [\text{mm}])^2} - (0.45 \times 0 [\text{mm}] - 0.53 \times 0 [\text{mm}]) \right]$$

$$= 1856.08 [\text{kNm}]$$

$$\lambda_{rel,LT} = \sqrt{\frac{W_{pl,y} \times f_y}{M_{cr}}} = \sqrt{\frac{5.3333 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1856.08 [\text{kNm}]} = 1.01$$

$$\beta = 0.75$$

$$\chi_{LT} = \min \left( \frac{1}{\varphi_{LT} + \sqrt{\varphi_{LT}^2 - \beta \times \lambda_{rel,LT}^2}}, \frac{1}{\lambda_{rel,LT}^2}, 1 \right) = \min \left( \frac{1}{0.99 + \sqrt{0.99^2 - 0.75 \times 1.01^2}}, \frac{1}{1.01^2}, 1 \right) = \min (0.69, 0.98, 1) = 0.69 \quad (\text{EC3-1-1: 6.57})$$

$$f = \min \left\{ 1 - 0.5 \times (1 - k_c) \times \left[ 1 - 2 \times (\lambda_{rel,LT} - 0.8)^2 \right], 1 \right\} = \min \left\{ 1 - 0.5 \times (1 - 0.94) \times \left[ 1 - 2 \times (1.01 - 0.8)^2 \right], 1 \right\} = \min \{0.97, 1\} = 0.97$$

$$\chi_{LT,mod} = \min \left( \frac{\chi_{LT}}{f}, 1 \right) = \min \left( \frac{0.69}{0.97}, 1 \right) = \min (0.71, 1) = 0.71$$

$$M_{b,Rd} = \chi_{LT,mod} \times W_{pl,y} \times \frac{f_y}{\gamma_{M1}} = 0.71 \times 5.3333 \cdot 10^{-3} [\text{m}^3] \times \frac{355.0 [\text{MPa}]}{1.00} = 1348.82 [\text{kNm}] \quad (\text{EC3-1-1: 6.55})$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{b,Rd}} = \frac{405.05 [\text{kNm}]}{1348.82 [\text{kNm}]} = 0.30 \leq 1.00 \quad (\text{EC3-1-1: 6.54})$$

**Note:** C parameters are determined according to ECCS 119 2006 / Galea 2002.

**Note:** The correction factor  $k_c$  is determined from C1.

The member satisfies the stability check.

### 2.2.3.3.3. Secondary Beam Check

**EC-EN 1993 Steel check ULS**  
 Values:  $UC_{overall}$   
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: B776

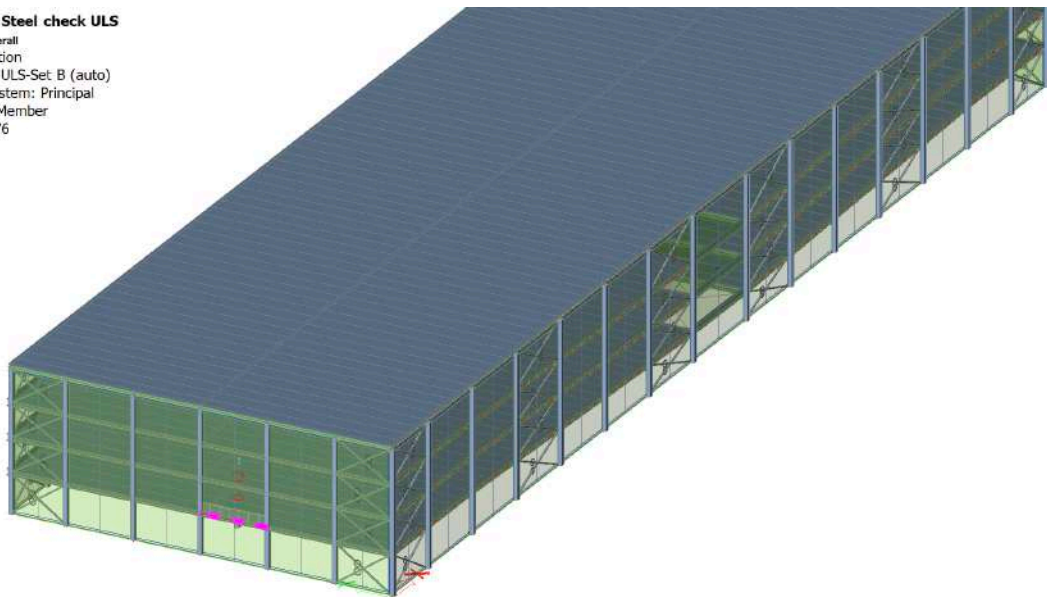


Figure 2.2.3.3. 6: Edge secondary beam

#### EC-EN 1993 Steel check ULS

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: B776

#### EN 1993-1-1 Code Check

National annex: Czech CSN-EN NA

Member B776	2.571 / 6.000 m	HEA320	S 355	ULS-Set B (auto)	0.01 -
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<b>Combination key</b>	
ULS-Set B (auto) / 1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 0.90*3DWind8	

<b>Partial safety factors</b>	
$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

<b>Material</b>			
Yield strength	$f_y$	355.0	MPa
Ultimate strength	$f_u$	490.0	MPa
Fabrication		Rolled	

....:SECTION CHECK:....

The critical check is on position 2.571 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	-16.91	kN
Shear force	$V_{y,Ed}$	0.00	kN
Shear force	$V_{z,Ed}$	0.55	kN
Torsion	$T_{Ed}$	0.04	kNm
Bending moment	$M_{y,Ed}$	5.68	kNm
Bending moment	$M_{z,Ed}$	0.00	kNm

### Classification for cross-section design

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_{\sigma}$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	118	16	-2.290e+03	-2.290e+03								
3	SO	118	16	-2.290e+03	-2.290e+03								
4	I	225	9	-1.429e+03	4.148e+03	-0.3		0.5	25.0	56.5	65.3	58.0	1
5	SO	118	16	5.009e+03	5.009e+03	1.0	0.4	1.0	7.6	7.3	8.1	11.4	2
7	SO	118	16	5.009e+03	5.009e+03	1.0	0.4	1.0	7.6	7.3	8.1	11.4	2

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 2

### Compression check

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	1.2400e-02	m <sup>2</sup>
Compression resistance	$N_{c,Rd}$	4402.00	kN
Unity check		0.00	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{1.2400 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 4402.00 [\text{kN}] \quad (\text{EC3-1-1: 6.10})$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-16.91 [\text{kN}]|}{4402.00 [\text{kN}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.9})$$

### Bending moment check for $M_y$

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	1.6292e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	578.36	kNm
Unity check		0.01	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{1.6292 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 578.36 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|5.68 [\text{kNm}]|}{578.36 [\text{kNm}]} = 0.01 \leq 1.00 \quad (\text{EC3-1-1: 6.12})$$

### Shear check for $V_z$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	4.0765e-03	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	835.52	kN
Unity check		0.00	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{4.0765 \cdot 10^{-3} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 835.52 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{pl,z,Rd}} = \frac{|0.55 [\text{kN}]|}{835.52 [\text{kN}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	2	
Total torsional moment	$T_{Ed}$	0.5	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.00	-

$$T_{Ed} = |T_{Ed}| \times T_{Ed,unit} = |38.11| \times 1.436 \cdot 10^{-1} [\text{kN/m}^2] = 0.5 [\text{MPa}]$$

$$T_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [\text{MPa}]}{\sqrt{3} \times 1.00} = 205.0 [\text{MPa}]$$

$$\text{Unity check} = \frac{T_{Ed}}{T_{Rd}} = \frac{0.5 [\text{MPa}]}{205.0 [\text{MPa}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.23})$$

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.31)

Plastic bending moment	$M_{pl,y,Rd}$	578.36	kNm
Unity check		0.01	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{1.6292 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}]}{1.00} = 578.36 [\text{kNm}] \quad (\text{EC3-1-1: 6.13})$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|5.68 [\text{kNm}]|}{578.36 [\text{kNm}]} = 0.01 \leq 1.00 \quad (\text{EC3-1-1: 6.31})$$

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

**Note:** Since the axial force satisfies both criteria (6.33) and (6.34) of EN 1993-1-1 article 6.2.9.1(4) its effect on the moment resistance about the y-y axis is neglected.

The member satisfies the section check.

.....STABILITY CHECK:.....

**Classification for member buckling design**

Decisive position for stability classification: 2.571 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\sigma$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	SO	118	16	-2.290e+03	-2.290e+03								
3	SO	118	16	-2.290e+03	-2.290e+03								
4	I	225	9	-1.429e+03	4.148e+03	-0.3		0.5	25.0	56.5	65.3	58.0	1
5	SO	118	16	5.009e+03	5.009e+03	1.0	0.4	1.0	7.6	7.3	8.1	11.4	2
7	SO	118	16	5.009e+03	5.009e+03	1.0	0.4	1.0	7.6	7.3	8.1	11.4	2

**Note:** The Classification limits have been set according to Semi-Comp+.

The cross-section is classified as Class 2

**Flexural Buckling check**

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters	yy	zz	
Sway type	sway	non-sway	
System length	L	6.000	m
Buckling factor	k	1.00	
Buckling length	$l_{cr}$	6.000	m
Critical Euler load	$N_{cr}$	13184.15	kN
Slenderness	$\lambda$	44.15	
Relative slenderness	$\lambda_{rel}$	0.58	
Limit slenderness	$\lambda_{rel,0}$	0.20	

**Note:** The slenderness or compression force is such that Flexural Buckling effects may be ignored according to EN 1993-1-1 article 6.3.1.2(4).

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 2.2900 \cdot 10^{-4} [\text{m}^4]}{6.000 [\text{m}]^2} = 13184.15 [\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 6.9900 \cdot 10^{-5} [\text{m}^4]}{6.000 [\text{m}]^2} = 4024.53 [\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{6.000 [\text{m}]}{136 [\text{mm}]} = 44.15$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{6.000 [\text{m}]}{75 [\text{mm}]} = 79.91$$

$$\lambda_{rel,y} = \frac{\lambda_y}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{44.15}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{355.0 [\text{MPa}]}}} = 0.55 \quad (\text{EC3-1-1: 6.50})$$

$$\lambda_{rel,z} = \frac{\lambda_z}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{79.91}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{355.0 [\text{MPa}]}}} = 1.05 \quad (\text{EC3-1-1: 6.50})$$

**Torsional(-Flexural) Buckling check**

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** For this I-section the Torsional(-Flexural) buckling resistance is higher than the resistance for Flexural buckling. Therefore Torsional(-Flexural) buckling is not printed on the output.

**Lateral Torsional Buckling check**

According to EN 1993-1-1 article 6.3.2.1 & 6.3.2.3 and formula (6.54)

LTB parameters		Alternative case	
Method for LTB curve			
Plastic section modulus	$W_{pl,y}$	1.6292e-03	m <sup>3</sup>
Elastic critical moment	$M_{cr}$	943.89	kNm
Relative slenderness	$\lambda_{rel,LT}$	0.78	
Limit slenderness	$\lambda_{rel,LT,0}$	0.40	

**Note:** The slenderness or bending moment is such that Lateral Torsional Buckling effects may be ignored according to EN 1993-1-1 article 6.3.2.2(4).

Mcr parameters			
LTB length	$l_{LT}$	6.000	m
Influence of load position		no influence	
Correction factor	k	1.00	
Correction factor	$k_w$	1.00	
LTB moment factor	$C_1$	1.13	
LTB moment factor	$C_2$	0.45	
LTB moment factor	$C_3$	0.53	
Shear centre distance	$d_z$	0	mm
Distance of load application	$z_g$	0	mm
Mono-symmetry constant	$\beta_y$	0	mm
Mono-symmetry constant	$z_j$	0	mm

$$M_{cr} = C_1 \times \frac{\pi^2 \times E \times I_y}{L_{cr}^2} \times \left[ \sqrt{\left(\frac{k}{k_w}\right)^2 \times \frac{I_w}{I_y} + \frac{I_y^2 \times G \times I_t}{\pi^2 \times E \times I_y} + (C_2 \times z_g - C_3 \times z_j)^2} - (C_2 \times z_g - C_3 \times z_j) \right] = 1.13$$

$$\times \frac{\pi^2 \times 210000.0[\text{MPa}] \times 6.9900 \cdot 10^{-5}[\text{m}^4]}{6.000[\text{m}]^2}$$

$$\times \left[ \sqrt{\left(\frac{1.00}{1.00}\right)^2 \times \frac{1.5124 \cdot 10^{-6}[\text{m}^6]}{6.9900 \cdot 10^{-5}[\text{m}^4]} + \frac{6.000[\text{m}]^2 \times 80769.2[\text{MPa}] \times 1.0800 \cdot 10^{-6}[\text{m}^4]}{\pi^2 \times 210000.0[\text{MPa}] \times 6.9900 \cdot 10^{-5}[\text{m}^4]} + (0.45 \times 0[\text{mm}] - 0.53 \times 0[\text{mm}])^2} - (0.45 \times 0[\text{mm}] - 0.53 \times 0[\text{mm}]) \right]$$

$$= 943.89[\text{kNm}]$$

$$\lambda_{rel,LT} = \sqrt{\frac{W_{pl,y} \times f_y}{M_{cr}}} = \sqrt{\frac{1.6292 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{943.89[\text{kNm}]} = 0.78$$

**Note:** C parameters are determined according to ECCS 119 2006 / Galea 2002.

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

Bending and axial compression check parameters			
Interaction method		alternative method 2	
Cross-section area	A	1.2400e-02	m <sup>2</sup>
Plastic section modulus	W <sub>pl,y</sub>	1.6292e-03	m <sup>3</sup>
Design compression force	N <sub>Ed</sub>	16.91	kN
Design bending moment (maximum)	M <sub>y,Ed</sub>	5.68	kNm
Design bending moment (maximum)	M <sub>z,Ed</sub>	0.00	kNm
Characteristic compression resistance	N <sub>Rk</sub>	4402.00	kN
Characteristic moment resistance	M <sub>y,Rk</sub>	578.36	kNm
Reduction factor	χ <sub>y</sub>	1.00	
Reduction factor	χ <sub>z</sub>	1.00	
Modified reduction factor	χ <sub>LT,mod</sub>	1.00	
Interaction factor	k <sub>yy</sub>	0.90	
Interaction factor	k <sub>zy</sub>	0.54	

Maximum moment M<sub>y,Ed</sub> is derived from beam 8776 position 2.571 m.  
Maximum moment M<sub>z,Ed</sub> is derived from beam 8776 position 0.000 m.

Interaction method 2 parameters			
Method for interaction factors		Table B.1	
Sway type y		sway	
Equivalent moment factor	C <sub>my</sub>	0.90	
Resulting load type LT		line load q	
End moment	M <sub>h,LT</sub>	0.00	kNm
Field moment	M <sub>s,LT</sub>	5.68	kNm
Factor	α <sub>h,LT</sub>	0.00	
Ratio of end moments	ψ <sub>LT</sub>	1.00	
Equivalent moment factor	C <sub>mLT</sub>	0.95	

Unity check (6.61) = 0.00 + 0.01 + 0.00 = 0.01 -

Unity check (6.62) = 0.00 + 0.01 + 0.00 = 0.01 -

C<sub>my</sub> = 0.90

$$\alpha_{h,LT} = \frac{M_{h,LT}}{M_{s,LT}} = \frac{0.00[\text{kNm}]}{5.68[\text{kNm}]} = 0.00$$

$$C_{mLT} = 0.95 + 0.05 \times \alpha_{h,LT} = 0.95 + 0.05 \times 0.00 = 0.95$$

$$N_{Rk} = A \times f_y = 1.2400 \cdot 10^{-2}[\text{m}^2] \times 355.0[\text{MPa}] = 4402.00[\text{kN}]$$

$$M_{y,Rk} = W_{pl,y} \times f_y = 1.6292 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}] = 578.36[\text{kNm}]$$

$$k_{yy} = \min \left\{ C_{my} \times \left[ 1 + (\lambda_{rel,y} - 0.2) \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right], C_{my} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right\}$$

$$= \min \left\{ 0.90 \times \left[ 1 + (0.58 - 0.2) \times \frac{16.91[\text{kN}]}{1.00 \times \frac{4402.00[\text{kN}]}{1.00}} \right], 0.90 \times \left( 1 + 0.8 \times \frac{16.91[\text{kN}]}{1.00 \times \frac{4402.00[\text{kN}]}{1.00}} \right) \right\} = \min \{0.90, 0.90\} = 0.90$$

$$k_{zy} = 0.6 \times k_{yy} = 0.6 \times 0.90 = 0.54$$

$$\text{Unity check (6.61)} = \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{yy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT,mod} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{yz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{16.91[\text{kN}]}{1.00 \times \frac{4402.00[\text{kN}]}{1.00}} + 0.90 \times \frac{5.68[\text{kNm}] + |0.00[\text{kNm}]|}{1.00 \times \frac{578.36[\text{kNm}]}{1.00}} + 0.60 \times \frac{0.00[\text{kNm}] + |0.00[\text{kNm}]|}{1.00 \times \frac{251.46[\text{kNm}]}{1.00}} = 0.01 \leq 1.00 \quad (\text{EC3-1-1: 6.61})$$

$$\text{Unity check (6.62)} = \frac{|N_{Ed}|}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\lambda_{LT,mod} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{16.91[\text{kN}]}{1.00 \times \frac{4402.00[\text{kN}]}{1.00}} + 0.54 \times \frac{5.68[\text{kNm}] + |0.00[\text{kNm}]|}{1.00 \times \frac{578.36[\text{kNm}]}{1.00}} + 1.01 \times \frac{0.00[\text{kNm}] + |0.00[\text{kNm}]|}{1.00 \times \frac{251.46[\text{kNm}]}{1.00}} = 0.01 \leq 1.00 \quad (\text{EC3-1-1: 6.62})$$

Unity check = max(Unity check (6.61), Unity check (6.62)) = max(0.01, 0.01) = 0.01 ≤ 1.00

### Shear Buckling check

According to EN 1993-1-5 article 5 & 7.1 and formula (5.10) & (7.1)

Shear Buckling parameters			
Buckling field length	a	6.000	m
Web		unstiffened	
Web height	h <sub>w</sub>	279	mm
Web thickness	t	9	mm
Material coefficient	ε	0.81	
Shear correction factor	η	1.20	

Shear Buckling verification		
Web slenderness	h <sub>w</sub> /t	31.00
Web slenderness limit		48.82

$$h_w/t = \frac{h_w}{t} = \frac{279[\text{mm}]}{9[\text{mm}]} = 31.00$$

$$\text{limit } h_w/t = \frac{72 \times \varepsilon}{\eta} = \frac{72 \times 0.81}{1.20} = 48.82$$

**Note:** The web slenderness is such that Shear Buckling effects may be ignored according to EN 1993-1-5 article 5.1(2).

The member satisfies the stability check.

## 2.2.3.3.4. Column Check

### EC-EN 1993 Steel check ULS

Values: **UC<sub>Overall</sub>**  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: B779

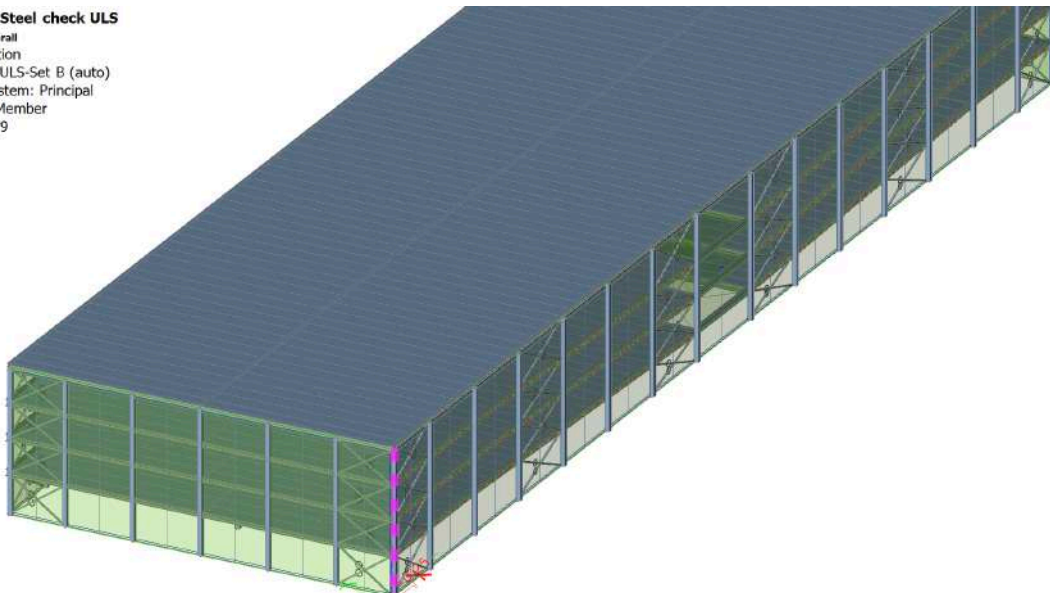


Figure 2.2.3.3. 7: Corner column

**EN 1993-1-1 Code Check**

National annex: Czech CSN-EN NA

<b>Member B779</b>	<b>12.600 / 12.600 m</b>	<b>SHS350/350/14.2</b>	<b>S 355</b>	<b>ULS-Set B (auto)</b>	<b>0.18 -</b>
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<b>Combination key</b>	
ULS-Set B (auto) / 1.15*LC1 + 1.15*LC2 + 1.05*LC3 + 1.05*LC4 + 1.50*3DWind4	

<b>Partial safety factors</b>	
γ <sub>M0</sub> for resistance of cross-sections	1.00
γ <sub>M1</sub> for resistance to instability	1.00
γ <sub>M2</sub> for resistance of net sections	1.25

<b>Material</b>			
Yield strength	f <sub>y</sub>	355.0	MPa
Ultimate strength	f <sub>u</sub>	490.0	MPa
Fabrication		Rolled	

.....SECTION CHECK:.....

The critical check is on position 12.600 m

Internal forces		Calculated	Unit
Normal force	N <sub>Ed</sub>	-636.53	kN
Shear force	V <sub>y,Ed</sub>	-8.80	kN
Shear force	V <sub>z,Ed</sub>	-33.66	kN
Torsion	T <sub>Ed</sub>	10.61	kNm
Bending moment	M <sub>y,Ed</sub>	-54.62	kNm
Bending moment	M <sub>z,Ed</sub>	-10.41	kNm

**Classification for cross-section design**

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	σ <sub>1</sub> [kN/m <sup>2</sup> ]	σ <sub>2</sub> [kN/m <sup>2</sup> ]	Ψ [-]	k <sub>σ</sub> [-]	α [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	307	14	6.435e+04	5.526e+04	0.9		1.0	21.6	22.8	27.7	32.5	1
3	I	307	14	5.264e+04	4.950e+03	0.1		1.0	21.6	22.8	27.7	45.1	1
5	I	307	14	3.167e+03	1.225e+04	0.3		1.0	21.6	22.8	27.7	41.6	1
7	I	307	14	1.488e+04	6.256e+04	0.2		1.0	21.6	22.8	27.7	42.0	1

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

**Compression check**

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	1.8900e-02	m <sup>2</sup>
Compression resistance	N <sub>c,Rd</sub>	6709.50	kN
Unity check		0.09	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{1.8900 \cdot 10^{-2} [m^2] \times 355.0 [MPa]}{1.00} = 6709.50 [kN] \quad (EC3-1-1: 6.10)$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-636.53[\text{kN}]|}{6709.50[\text{kN}]} = 0.09 \leq 1.00$$

(EC3-1-1: 6.9)

**Bending moment check for  $M_y$**   
According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	2.3640e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	839.22	kNm
Unity check		0.07	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{2.3640 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{1.00} = 839.22[\text{kNm}]$$

(EC3-1-1: 6.13)

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|-54.62[\text{kNm}]|}{839.22[\text{kNm}]} = 0.07 \leq 1.00$$

(EC3-1-1: 6.12)

**Bending moment check for  $M_z$**   
According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,z}$	2.3640e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,z,Rd}$	839.22	kNm
Unity check		0.01	-

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{2.3640 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}]}{1.00} = 839.22[\text{kNm}]$$

(EC3-1-1: 6.13)

$$\text{Unity check} = \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} = \frac{|-10.41[\text{kNm}]|}{839.22[\text{kNm}]} = 0.01 \leq 1.00$$

(EC3-1-1: 6.12)

**Shear check for  $V_y$**   
According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.4500e-03	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	1936.87	kN
Unity check		0.00	-

$$V_{pl,y,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.4500 \cdot 10^{-3}[\text{m}^2] \times \frac{355.0[\text{MPa}]}{\sqrt{3}}}{1.00} = 1936.87[\text{kN}]$$

(EC3-1-1: 6.18)

$$\text{Unity check} = \frac{|V_{y,Ed}|}{V_{c,y,Rd}} = \frac{|-8.80[\text{kN}]|}{1936.87[\text{kN}]} = 0.00 \leq 1.00$$

(EC3-1-1: 6.17)

**Shear check for  $V_z$**   
According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.4500e-03	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	1936.87	kN
Unity check		0.02	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.4500 \cdot 10^{-3}[\text{m}^2] \times \frac{355.0[\text{MPa}]}{\sqrt{3}}}{1.00} = 1936.87[\text{kN}]$$

(EC3-1-1: 6.18)

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{c,z,Rd}} = \frac{|-33.66[\text{kN}]|}{1936.87[\text{kN}]} = 0.02 \leq 1.00$$

(EC3-1-1: 6.17)

**Torsion check**  
According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	1	
Total torsional moment	$T_{Ed}$	3.3	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.02	-

$$T_{Ed} = |T_{Ed}| \times T_{Ed,mit} = |10612.76| \times 3.123 \cdot 10^{-1}[\text{kN/m}^2] = 3.3[\text{MPa}]$$

$$T_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0[\text{MPa}]}{\sqrt{3} \times 1.00} = 205.0[\text{MPa}]$$

$$\text{Unity check} = \frac{T_{Ed}}{T_{Rd}} = \frac{3.3[\text{MPa}]}{205.0[\text{MPa}]} = 0.02 \leq 1.00$$

(EC3-1-1: 6.23)

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

**Combined bending, axial force and shear force check**  
According to EN 1993-1-1 article 6.2.9.1 and formula (6.41)

Design plastic moment resistance reduced due to $N_{Ed}$	$M_{N,y,Rd}$	839.22	kNm
Exponent of bending ratio $y$	$\alpha$	1.68	
Design plastic moment resistance reduced due to $N_{Ed}$	$M_{N,z,Rd}$	839.22	kNm
Exponent of bending ratio $z$	$\beta$	1.68	



Unity check (6.41) = 0.01 + 0.00 = 0.01 -

$$M_{N,y,Rd} = \min \left[ \frac{M_{pl,y,Rd} \times (1-n)}{1-0.5 \times \text{ratio}_{A,w}}, M_{pl,y,Rd} \right] = \min \left[ \frac{839.22[\text{kNm}] \times (1-0.09)}{1-0.5 \times 0.47}, 839.22[\text{kNm}] \right] = \min [995.60[\text{kNm}], 839.22[\text{kNm}]] = 839.22[\text{kNm}] \quad (\text{EC3-1-1: 6.39})$$

$$\alpha = \max \left[ \min \left( \frac{1.66}{1-1.13 \times n^2}, 6 \right), 1 \right] = \max \left[ \min \left( \frac{1.66}{1-1.13 \times 0.09^2}, 6 \right), 1 \right] = \max [\min (1.68, 6), 1] = 1.68$$

$$M_{N,z,Rd} = \min \left[ \frac{M_{pl,z,Rd} \times (1-n)}{1-0.5 \times \text{ratio}_{A,t}}, M_{pl,z,Rd} \right] = \min \left[ \frac{839.22[\text{kNm}] \times (1-0.09)}{1-0.5 \times 0.47}, 839.22[\text{kNm}] \right] = \min [995.60[\text{kNm}], 839.22[\text{kNm}]] = 839.22[\text{kNm}] \quad (\text{EC3-1-1: 6.40})$$

$$\beta = \max \left[ \min \left( \frac{1.66}{1-1.13 \times n^2}, 6 \right), 1 \right] = \max \left[ \min \left( \frac{1.66}{1-1.13 \times 0.09^2}, 6 \right), 1 \right] = \max [\min (1.68, 6), 1] = 1.68$$

$$\text{Unity check} = \left( \frac{|M_{y,Ed}|}{M_{N,y,Rd}} \right)^\alpha + \left( \frac{|M_{z,Ed}|}{M_{N,z,Rd}} \right)^\beta = \left( \frac{-54.62[\text{kNm}]}{839.22[\text{kNm}]} \right)^{1.68} + \left( \frac{-10.41[\text{kNm}]}{839.22[\text{kNm}]} \right)^{1.68} = \mathbf{0.01} \leq \mathbf{1.00} \quad (\text{EC3-1-1: 6.41})$$

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

The member satisfies the section check.

....:STABILITY CHECK:....

**Classification for member buckling design**

Decisive position for stability classification: 9.100 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\sigma$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class [-]
1	I	307	14	-3.420e+03	1.438e+04	-0.2		0.8	21.6	29.8	35.8	54.2	1
3	I	307	14	1.757e+04	6.877e+04	0.3		1.0	21.6	22.8	27.7	41.7	1
5	I	307	14	7.031e+04	5.251e+04	0.7		1.0	21.6	22.8	27.7	33.9	1
7	I	307	14	4.932e+04	-1.877e+03	0.0		1.0	21.6	23.9	28.9	48.3	1

**Note:** The Classification limits have been set according to Semi-Comp+.

The cross-section is classified as Class 1

**Flexural Buckling check**

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters		yy	zz	
Sway type		sway	non-sway	
System length	L	3.500	3.500	m
Buckling factor	k	1.95	0.58	
Buckling length	$l_{cr}$	6.840	2.020	m
Critical Euler load	$N_{cr}$	15597.86	178802.49	kN
Slenderness	$\lambda$	50.11	14.80	
Relative slenderness	$\lambda_{rel}$	0.66	0.19	
Limit slenderness	$\lambda_{rel,0}$	0.20	0.20	
Buckling curve	a	a		
Imperfection	$\alpha$	0.21	0.21	
Reduction factor	$\chi$	0.87	1.00	
Buckling resistance	$N_{b,Rd}$	5820.57	6709.50	kN

Flexural Buckling verification			
Cross-section area	A	1.8900e-02	m <sup>2</sup>
Buckling resistance	$N_{b,Rd}$	5820.57	kN
Unity check		0.11	-

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 3.5210 \cdot 10^{-4}[\text{m}^4]}{6.840[\text{m}]^2} = 15597.86[\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 3.5210 \cdot 10^{-4}[\text{m}^4]}{2.020[\text{m}]^2} = 178802.49[\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{6.840[\text{m}]}{136[\text{mm}]} = 50.11$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{2.020[\text{m}]}{136[\text{mm}]} = 14.80$$

$$\lambda_{rel,y} = \frac{\lambda_y}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{50.11}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{355.0[\text{MPa}]}}} = 0.66 \quad (\text{EC3-1-1: 6.50})$$

$$\lambda_{rel,z} = \frac{\lambda_z}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{14.80}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{355.0[\text{MPa}]}}} = 0.19 \quad (\text{EC3-1-1: 6.50})$$

$$\varphi_y = 0.5 \times [1 + \alpha_y \times (\lambda_{rel,y} - \lambda_{rel,y,0}) + \lambda_{rel,y}^2] = 0.5 \times [1 + 0.21 \times (0.66 - 0.20) + 0.66^2] = 0.67$$

$$\varphi_z = 0.5 \times [1 + \alpha_z \times (\lambda_{rel,z} - \lambda_{rel,z,0}) + \lambda_{rel,z}^2] = 0.5 \times [1 + 0.21 \times (0.19 - 0.20) + 0.19^2] = 0.52$$

$$\chi_y = \min \left( \frac{1}{\varphi_y + \sqrt{\varphi_y^2 - \lambda_{rel,y}^2}}, \frac{1}{\lambda_{rel,y}^2}, 1 \right) = \min \left( \frac{1}{0.76 + \sqrt{0.76^2 - 0.66^2}}, \frac{1}{0.66^2}, 1 \right) = \min(0.87, 2.32, 1) = 0.87 \quad (\text{EC3-1-1: 6.49})$$

$$\chi_z = \min \left( \frac{1}{\varphi_z + \sqrt{\varphi_z^2 - \lambda_{rel,z}^2}}, \frac{1}{\lambda_{rel,z}^2}, 1 \right) = \min \left( \frac{1}{0.52 + \sqrt{0.52^2 - 0.19^2}}, \frac{1}{0.19^2}, 1 \right) = \min(1.00, 26.65, 1) = 1.00 \quad (\text{EC3-1-1: 6.49})$$

$$N_{b,y,Rd} = \frac{\chi_y \times A \times f_y}{\gamma_{M1}} = \frac{0.87 \times 1.8900 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 5820.57 [\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,z,Rd} = \frac{\chi_z \times A \times f_y}{\gamma_{M1}} = \frac{1.00 \times 1.8900 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 6709.50 [\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,Rd} = \min(N_{b,y,Rd}, N_{b,z,Rd}) = \min(5820.57 [\text{kN}], 6709.50 [\text{kN}]) = 5820.57 [\text{kN}]$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{b,Rd}} = \frac{|-636.53 [\text{kN}]|}{5820.57 [\text{kN}]} = 0.11 \leq 1.00 \quad (\text{EC3-1-1: 6.46})$$

### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** The cross-section concerns a RHS section which is not susceptible to Torsional(-Flexural) Buckling.

### Lateral Torsional Buckling check

According to EN 1993-1-1 article 6.3.2.1

**Note:** The cross-section concerns an RHS section with 'h / b < 10 / λ<sub>rel,z</sub>'.  
This section is thus not susceptible to Lateral Torsional Buckling.

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

Bending and axial compression check parameters			
Interaction method		alternative method 2	
Cross-section area	A	1.8900e-02	m <sup>2</sup>
Plastic section modulus	W <sub>pl,y</sub>	2.3640e-03	m <sup>3</sup>
Plastic section modulus	W <sub>pl,z</sub>	2.3640e-03	m <sup>3</sup>
Design compression force	N <sub>Ed</sub>	636.53	kN
Design bending moment (maximum)	M <sub>y,Ed</sub>	58.64	kNm
Design bending moment (maximum)	M <sub>z,Ed</sub>	20.39	kNm
Characteristic compression resistance	N <sub>Rk</sub>	6709.50	kN
Characteristic moment resistance	M <sub>y,Rk</sub>	839.22	kNm
Characteristic moment resistance	M <sub>z,Rk</sub>	839.22	kNm
Reduction factor	χ <sub>y</sub>	0.87	
Reduction factor	χ <sub>z</sub>	1.00	
Reduction factor	χ <sub>LT</sub>	1.00	
Interaction factor	k <sub>yy</sub>	0.94	
Interaction factor	k <sub>yz</sub>	0.24	
Interaction factor	k <sub>zy</sub>	0.57	
Interaction factor	k <sub>zz</sub>	0.40	

Maximum moment M<sub>y,Ed</sub> is derived from beam B779 position 9.100 m.

Maximum moment M<sub>z,Ed</sub> is derived from beam B779 position 9.100 m.

Interaction method 2 parameters			
Method for interaction factors		Table B.1	
Sway type y		sway	
Equivalent moment factor	C <sub>my</sub>	0.90	
Resulting load type z		linear moment M	
Ratio of end moments	ψ <sub>z</sub>	-0.51	
Equivalent moment factor	C <sub>mz</sub>	0.40	
Resulting load type LT		line load q	
End moment	M <sub>h,LT</sub>	58.64	kNm
Field moment	M <sub>s,LT</sub>	3.14	kNm
Factor	α <sub>s,LT</sub>	0.05	
Ratio of end moments	ψ <sub>LT</sub>	-0.93	
Equivalent moment factor	C <sub>mLT</sub>	0.40	

$$\text{Unity check (6.61)} = 0.11 + 0.07 + 0.01 = 0.18 -$$

$$\text{Unity check (6.62)} = 0.09 + 0.04 + 0.01 = 0.14 -$$

$$C_{my} = 0.90$$

$$C_{mz} = \max(0.6 + 0.4 \times \psi_z, 0.4) = \max(0.6 + 0.4 \times -0.51, 0.4) = \max(0.40, 0.4) = 0.40$$

$$\alpha_{s,LT} = \frac{M_{s,LT}}{M_{h,LT}} = \frac{3.14 [\text{kNm}]}{58.64 [\text{kNm}]} = 0.05$$

$$C_{mLT} = \max(0.2 + 0.8 \times \alpha_{s,LT}, 0.4) = \max(0.2 + 0.8 \times 0.05, 0.4) = \max(0.24, 0.4) = 0.40$$

$$N_{Rk} = A \times f_y = 1.8900 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}] = 6709.50 [\text{kN}]$$

$$M_{y,Rk} = W_{pl,y} \times f_y = 2.3640 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}] = 839.22 [\text{kNm}]$$

$$M_{z,Rk} = W_{pl,z} \times f_y = 2.3640 \cdot 10^{-3} [\text{m}^3] \times 355.0 [\text{MPa}] = 839.22 [\text{kNm}]$$

$$k_{yy} = \min \left\{ C_{my} \times \left[ 1 + (\lambda_{rel,y} - 0.2) \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right], C_{my} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right\}$$

$$= \min \left\{ 0.90 \times \left[ 1 + (0.66 - 0.2) \times \frac{636.53 [\text{kN}]}{0.87 \times \frac{6709.50 [\text{kN}]}{1.00}} \right], 0.90 \times \left( 1 + 0.8 \times \frac{636.53 [\text{kN}]}{0.87 \times \frac{6709.50 [\text{kN}]}{1.00}} \right) \right\} = \min \{0.94, 0.98\} = 0.94$$

$$k_{yz} = 0.6 \times k_{zz} = 0.6 \times 0.40 = 0.24$$

$$k_{zy} = 0.6 \times k_{yy} = 0.6 \times 0.94 = 0.57$$

$$k_{zz} = \min \left[ C_{mz}, C_{mz} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right] = \min \left[ 0.40, 0.40 \times \left( 1 + 0.8 \times \frac{636.53 [\text{kN}]}{1.00 \times \frac{6709.50 [\text{kN}]}{1.00}} \right) \right] = \min [0.40, 0.43] = 0.40$$

$$\text{Unity check (6.61)} = \frac{|N_{Ed}|}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{yy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\lambda_{LT} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{yz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{636.53 [\text{kN}]}{0.87 \times \frac{6709.50 [\text{kN}]}{1.00}} + 0.94 \times \frac{58.64 [\text{kNm}] + |0.00 [\text{kNm}]|}{1.00 \times \frac{839.22 [\text{kNm}]}{1.00}} + 0.24 \times \frac{20.39 [\text{kNm}] + |0.00 [\text{kNm}]|}{\frac{839.22 [\text{kNm}]}{1.00}} = \mathbf{0.18} \leq \mathbf{1.00}$$

(EC3-1-1: 6.61)

$$\text{Unity check (6.62)} = \frac{|N_{Ed}|}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\lambda_{LT} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{636.53 [\text{kN}]}{1.00 \times \frac{6709.50 [\text{kN}]}{1.00}} + 0.57 \times \frac{58.64 [\text{kNm}] + |0.00 [\text{kNm}]|}{1.00 \times \frac{839.22 [\text{kNm}]}{1.00}} + 0.40 \times \frac{20.39 [\text{kNm}] + |0.00 [\text{kNm}]|}{\frac{839.22 [\text{kNm}]}{1.00}} = \mathbf{0.14} \leq \mathbf{1.00}$$

(EC3-1-1: 6.62)

$$\text{Unity check} = \max(\text{Unity check (6.61)}, \text{Unity check (6.62)}) = \max(0.18, 0.14) = \mathbf{0.18} \leq \mathbf{1.00}$$

The member satisfies the stability check.

#### EC-EN 1993 Steel check ULS

Values: **UC<sub>Overall</sub>**  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: B812

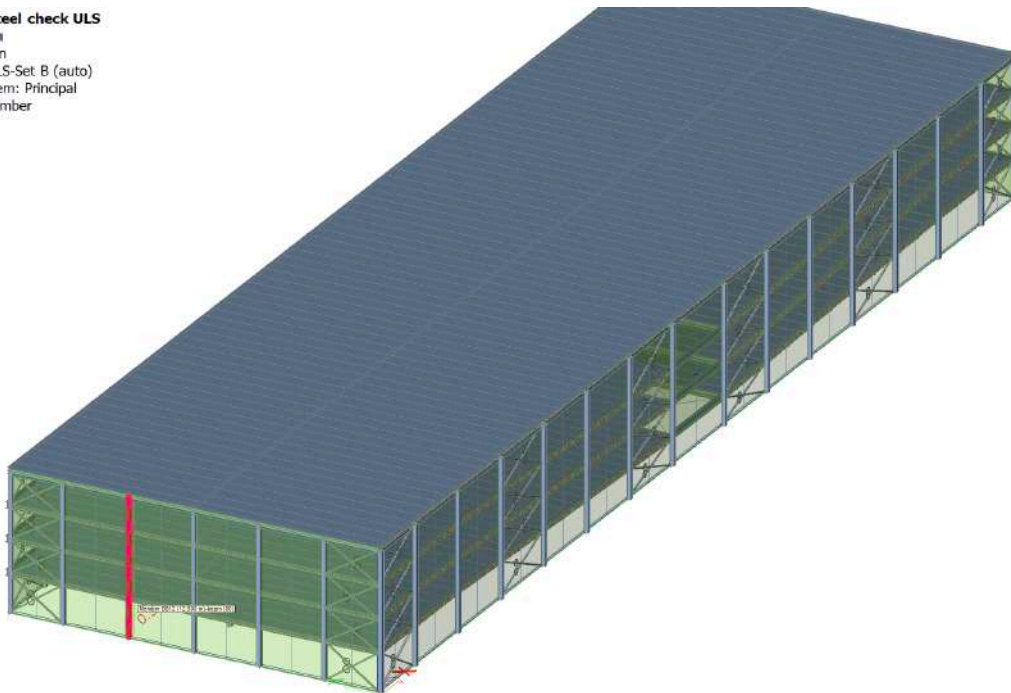


Figure 2.2.3.3. 8: Edge column

**EN 1993-1-1 Code Check**

National annex: Czech CSN-EN NA

Member	B812	12.600 / 12.600 m	SHS350/350/14.2	S 355	ULS-Set B (auto)	0.39 -
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Combination key	ULS-Set B (auto) / 1.15*LC1 + 1.15*LC2 + 1.05*LC3 + 1.05*LC4 + 1.50*3DWind16
-----------------	--

Partial safety factors	
$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

Material			
Yield strength	$f_y$	355.0	MPa
Ultimate strength	$f_u$	490.0	MPa
Fabrication		Rolled	

**....SECTION CHECK:....**

The critical check is on position 12.600 m

Internal forces		Calculated	Unit
Normal force	$N_{Ed}$	-111.36	kN
Shear force	$V_{y,Ed}$	-1.82	kN
Shear force	$V_{z,Ed}$	109.07	kN
Torsion	$T_{Ed}$	14.52	kNm
Bending moment	$M_{y,Ed}$	306.60	kNm
Bending moment	$M_{z,Ed}$	-5.14	kNm

**Classification for cross-section design**

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\psi$ [-]	$k_o$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	307	14	-1.381e+05	-1.426e+05								
3	I	307	14	-1.304e+05	1.373e+05	-0.9		0.5	21.6	56.3	65.1	95.6	1
5	I	307	14	1.499e+05	1.544e+05	1.0		1.0	21.6	22.8	27.7	31.2	1
7	I	307	14	1.422e+05	-1.255e+05	-0.9		0.5	21.6	53.3	62.0	89.2	1

**Note:** The Classification limits have been set according to Semi-Comp+.  
The cross-section is classified as Class 1

**Compression check**

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	1.8900e-02	m <sup>2</sup>
Compression resistance	$N_{c,Rd}$	6709.50	kN
Unity check		0.02	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{1.8900 \cdot 10^{-2} [m^2] \times 355.0 [MPa]}{1.00} = 6709.50 [kN] \quad (EC3-1-1: 6.10)$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-111.36 [kN]|}{6709.50 [kN]} = 0.02 \leq 1.00 \quad (EC3-1-1: 6.9)$$

**Bending moment check for  $M_y$**

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,y}$	2.3640e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,y,Rd}$	839.22	kNm
Unity check		0.37	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{2.3640 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 839.22 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|306.60 [kNm]|}{839.22 [kNm]} = 0.37 \leq 1.00 \quad (EC3-1-1: 6.12)$$

**Bending moment check for  $M_z$**

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	$W_{pl,z}$	2.3640e-03	m <sup>3</sup>
Plastic bending moment	$M_{pl,z,Rd}$	839.22	kNm
Unity check		0.01	-

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{2.3640 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 839.22 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{z,Ed}|}{M_{pl,z,Rd}} = \frac{|-5.14 [kNm]|}{839.22 [kNm]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.12)$$

**Shear check for  $V_y$**

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.4500e-03	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	1936.87	kN
Unity check		0.00	-

### Shear check for $V_y$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.4500e-03	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	1936.87	kN
Unity check		0.00	-

$$V_{pl,y,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.4500 \cdot 10^{-3} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 1936.87 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{y,Ed}|}{V_{c,y,Rd}} = \frac{|-1.82 [\text{kN}]|}{1936.87 [\text{kN}]} = 0.00 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Shear check for $V_z$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	9.4500e-03	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	1936.87	kN
Unity check		0.06	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{9.4500 \cdot 10^{-3} [\text{m}^2] \times \frac{355.0 [\text{MPa}]}{\sqrt{3}}}{1.00} = 1936.87 [\text{kN}] \quad (\text{EC3-1-1: 6.18})$$

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{c,z,Rd}} = \frac{|109.07 [\text{kN}]|}{1936.87 [\text{kN}]} = 0.06 \leq 1.00 \quad (\text{EC3-1-1: 6.17})$$

### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	1	
Total torsional moment	$T_{Ed}$	4.5	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.02	-

$$T_{Ed} = |T_{Ed}| \times T_{Ed,unit} = |14521.49| \times 3.123 \cdot 10^{-1} [\text{kN/m}^2] = 4.5 [\text{MPa}]$$

$$T_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [\text{MPa}]}{\sqrt{3} \times 1.00} = 205.0 [\text{MPa}]$$

$$\text{Unity check} = \frac{T_{Ed}}{T_{Rd}} = \frac{4.5 [\text{MPa}]}{205.0 [\text{MPa}]} = 0.02 \leq 1.00 \quad (\text{EC3-1-1: 6.23})$$

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.41)

Design plastic moment resistance reduced due to $N_{Ed}$	$M_{N,y,Rd}$	839.22	kNm
Exponent of bending ratio $y$	$\alpha$	1.66	
Design plastic moment resistance reduced due to $N_{Ed}$	$M_{N,z,Rd}$	839.22	kNm
Exponent of bending ratio $z$	$\beta$	1.66	

$$\text{Unity check (6.41)} = 0.19 + 0.00 = 0.19 <$$

$$M_{N,y,Rd} = \min \left[ \frac{M_{pl,y,Rd} \times (1-n)}{1-0.5 \times \text{ratio}_{A_w}}, M_{pl,y,Rd} \right] = \min \left[ \frac{839.22 [\text{kNm}] \times (1-0.02)}{1-0.5 \times 0.47}, 839.22 [\text{kNm}] \right] = \min [1081.69 [\text{kNm}], 839.22 [\text{kNm}]] = 839.22 [\text{kNm}] \quad (\text{EC3-1-1: 6.39})$$

$$\alpha = \max \left[ \min \left( \frac{1.66}{1-1.13 \times n^2}, 6 \right), 1 \right] = \max \left[ \min \left( \frac{1.66}{1-1.13 \times 0.02^2}, 6 \right), 1 \right] = \max [\min(1.66, 6), 1] = 1.66$$

$$M_{N,z,Rd} = \min \left[ \frac{M_{pl,z,Rd} \times (1-n)}{1-0.5 \times \text{ratio}_{A_f}}, M_{pl,z,Rd} \right] = \min \left[ \frac{839.22 [\text{kNm}] \times (1-0.02)}{1-0.5 \times 0.47}, 839.22 [\text{kNm}] \right] = \min [1081.69 [\text{kNm}], 839.22 [\text{kNm}]] = 839.22 [\text{kNm}] \quad (\text{EC3-1-1: 6.40})$$

$$\beta = \max \left[ \min \left( \frac{1.66}{1-1.13 \times n^2}, 6 \right), 1 \right] = \max \left[ \min \left( \frac{1.66}{1-1.13 \times 0.02^2}, 6 \right), 1 \right] = \max [\min(1.66, 6), 1] = 1.66$$

$$\text{Unity check} = \left( \frac{|M_{y,Ed}|}{M_{N,y,Rd}} \right)^\alpha + \left( \frac{|M_{z,Ed}|}{M_{N,z,Rd}} \right)^\beta = \left( \frac{|306.60 [\text{kNm}]|}{839.22 [\text{kNm}]} \right)^{1.66} + \left( \frac{|-5.14 [\text{kNm}]|}{839.22 [\text{kNm}]} \right)^{1.66} = 0.19 \leq 1.00 \quad (\text{EC3-1-1: 6.41})$$

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

The member satisfies the section check.

### ....:STABILITY CHECK:....

#### Classification for member buckling design

Decisive position for stability classification: 12.600 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\sigma$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	307	14	-1.381e+05	-1.426e+05								
3	I	307	14	-1.304e+05	1.373e+05	-0.9		0.5	21.6	56.3	65.1	95.6	1
5	I	307	14	1.499e+05	1.544e+05	1.0		1.0	21.6	22.8	27.7	31.2	1
7	I	307	14	1.422e+05	-1.255e+05	-0.9		0.5	21.6	53.3	62.0	89.2	1

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

### Flexural Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters		yy	zz	
Sway type		sway	non-sway	
System length	L	12.600	12.600	m
Buckling factor	k	1.33	0.50	
Buckling length	$l_{cr}$	16.729	6.313	m
Critical Euler load	$N_{cr}$	2607.66	18313.53	kN
Slenderness	$\lambda$	122.56	46.25	
Relative slenderness	$\lambda_{rel}$	1.60	0.61	
Limit slenderness	$\lambda_{rel,0}$	0.20	0.20	
Buckling curve	a			
Imperfection	$\alpha$	0.21	0.21	
Reduction factor	$\chi$	0.33	0.89	
Buckling resistance	$N_{b,Rd}$	2225.94	5957.91	kN

Flexural Buckling verification			
Cross-section area	A	1.8900e-02	m <sup>2</sup>
Buckling resistance	$N_{b,Rd}$	2225.94	kN
Unity check		0.05	-

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 3.5210 \cdot 10^{-4}[\text{m}^4]}{16.729[\text{m}]^2} = 2607.66[\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 3.5210 \cdot 10^{-4}[\text{m}^4]}{6.313[\text{m}]^2} = 18313.53[\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{16.729[\text{m}]}{136[\text{mm}]} = 122.56$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{6.313[\text{m}]}{136[\text{mm}]} = 46.25$$

$$\lambda_{rel,y} = \frac{\lambda_y}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{122.56}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{355.0[\text{MPa}]}}} = 1.60 \quad (\text{EC3-1-1: 6.50})$$

$$\lambda_{rel,z} = \frac{\lambda_z}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{46.25}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{355.0[\text{MPa}]}}} = 0.61 \quad (\text{EC3-1-1: 6.50})$$

$$\varphi_y = 0.5 \times [1 + \alpha_y \times (\lambda_{rel,y} - \lambda_{rel,y,0}) + \lambda_{rel,y}^2] = 0.5 \times [1 + 0.21 \times (1.60 - 0.20) + 1.60^2] = 1.93$$

$$\varphi_z = 0.5 \times [1 + \alpha_z \times (\lambda_{rel,z} - \lambda_{rel,z,0}) + \lambda_{rel,z}^2] = 0.5 \times [1 + 0.21 \times (0.61 - 0.20) + 0.61^2] = 0.73$$

$$\chi_y = \min \left( \frac{1}{\varphi_y + \sqrt{\varphi_y^2 - \lambda_{rel,y}^2}}, \frac{1}{\lambda_{rel,y}^2}, 1 \right) = \min \left( \frac{1}{1.93 + \sqrt{1.93^2 - 1.60^2}}, \frac{1}{1.60^2}, 1 \right) = \min(0.33, 0.39, 1) = 0.33 \quad (\text{EC3-1-1: 6.49})$$

$$\chi_z = \min \left( \frac{1}{\varphi_z + \sqrt{\varphi_z^2 - \lambda_{rel,z}^2}}, \frac{1}{\lambda_{rel,z}^2}, 1 \right) = \min \left( \frac{1}{0.73 + \sqrt{0.73^2 - 0.61^2}}, \frac{1}{0.61^2}, 1 \right) = \min(0.89, 2.73, 1) = 0.89 \quad (\text{EC3-1-1: 6.49})$$

$$N_{b,y,Rd} = \frac{\chi_y \times A \times f_y}{\gamma_{M1}} = \frac{0.33 \times 1.8900 \cdot 10^{-2}[\text{m}^2] \times 355.0[\text{MPa}]}{1.00} = 2225.94[\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,z,Rd} = \frac{\chi_z \times A \times f_y}{\gamma_{M1}} = \frac{0.89 \times 1.8900 \cdot 10^{-2}[\text{m}^2] \times 355.0[\text{MPa}]}{1.00} = 5957.91[\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,Rd} = \min(N_{b,y,Rd}, N_{b,z,Rd}) = \min(2225.94[\text{kN}], 5957.91[\text{kN}]) = 2225.94[\text{kN}]$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{b,Rd}} = \frac{|-111.36[\text{kN}]|}{2225.94[\text{kN}]} = 0.05 \leq 1.00 \quad (\text{EC3-1-1: 6.46})$$

### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** The cross-section concerns a RHS section which is not susceptible to Torsional(-Flexural) Buckling.

### Lateral Torsional Buckling check

According to EN 1993-1-1 article 6.3.2.1

**Note:** The cross-section concerns an RHS section with 'h / b < 10 /  $\lambda_{rel,z}$ '. This section is thus not susceptible to Lateral Torsional Buckling.

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

Bending and axial compression check parameters			
Interaction method		alternative method 2	
Cross-section area	A	1.8900e-02	m <sup>2</sup>
Plastic section modulus	W <sub>pl,y</sub>	2.3640e-03	m <sup>3</sup>
Plastic section modulus	W <sub>pl,z</sub>	2.3640e-03	m <sup>3</sup>
Design compression force	N <sub>Ed</sub>	111.36	kN
Design bending moment (maximum)	M <sub>y,Ed</sub>	306.60	kNm
Design bending moment (maximum)	M <sub>z,Ed</sub>	-5.14	kNm
Characteristic compression resistance	N <sub>Rk</sub>	6709.50	kN
Characteristic moment resistance	M <sub>y,Rk</sub>	839.22	kNm
Characteristic moment resistance	M <sub>z,Rk</sub>	839.22	kNm
Reduction factor	χ <sub>y</sub>	0.33	
Reduction factor	χ <sub>z</sub>	0.89	
Reduction factor	χ <sub>LT</sub>	1.00	
Interaction factor	k <sub>yy</sub>	0.94	
Interaction factor	k <sub>yz</sub>	0.24	
Interaction factor	k <sub>zy</sub>	0.56	
Interaction factor	k <sub>zz</sub>	0.40	

Maximum moment M<sub>y,Ed</sub> is derived from beam B812 position 12.600 m.  
Maximum moment M<sub>z,Ed</sub> is derived from beam B812 position 12.600 m.

Interaction method 2 parameters			
Method for interaction factors		Table B.1	
Sway type y		sway	
Equivalent moment factor	C <sub>my</sub>	0.90	
Resulting load type z		point load F	
End moment	M <sub>h,z</sub>	-5.14	kNm
Field moment	M <sub>s,z</sub>	1.21	kNm
Factor	α <sub>s,z</sub>	-0.24	
Ratio of end moments	ψ <sub>z</sub>	-0.81	
Equivalent moment factor	C <sub>mz</sub>	0.40	
Resulting load type LT		line load q	
End moment	M <sub>h,LT</sub>	306.60	kNm
Field moment	M <sub>s,LT</sub>	-115.94	kNm
Factor	α <sub>s,LT</sub>	-0.38	
Ratio of end moments	ψ <sub>LT</sub>	0.11	
Equivalent moment factor	C <sub>mLT</sub>	0.40	

Unity check (6.61) = 0.05 + 0.34 + 0.00 = 0.39 -  
Unity check (6.62) = 0.02 + 0.21 + 0.00 = 0.23 -

$$C_{my} = 0.90$$

$$\alpha_{s,z} = \frac{M_{s,z}}{M_{h,z}} = \frac{1.21 [kNm]}{5.14 [kNm]} = -0.24$$

$$C_{mz} = \max(-0.2 \times \psi_z - 0.8 \times \alpha_{s,z}, 0.4) = \max(-0.2 \times -0.81 - 0.8 \times -0.24, 0.4) = \max(0.35, 0.4) = 0.40$$

$$\alpha_{s,LT} = \frac{M_{s,LT}}{M_{h,LT}} = \frac{-115.94 [kNm]}{306.60 [kNm]} = -0.38$$

$$C_{mLT} = \max(0.1 - 0.8 \times \alpha_{s,LT}, 0.4) = \max(0.1 - 0.8 \times -0.38, 0.4) = \max(0.40, 0.4) = 0.40$$

$$N_{Rk} = A \times f_y = 1.8900 \cdot 10^{-2} [m^2] \times 355.0 [MPa] = 6709.50 [kN]$$

$$M_{y,Rk} = W_{pl,y} \times f_y = 2.3640 \cdot 10^{-3} [m^3] \times 355.0 [MPa] = 839.22 [kNm]$$

$$M_{z,Rk} = W_{pl,z} \times f_y = 2.3640 \cdot 10^{-3} [m^3] \times 355.0 [MPa] = 839.22 [kNm]$$

$$k_{yy} = \min \left\{ C_{my} \times \left[ 1 + (\lambda_{rel,y} - 0.2) \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right], C_{my} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right\}$$

$$= \min \left\{ 0.90 \times \left[ 1 + (1.60 - 0.2) \times \frac{111.36 [kN]}{0.33 \times \frac{6709.50 [kN]}{1.00}} \right], 0.90 \times \left( 1 + 0.8 \times \frac{111.36 [kN]}{0.33 \times \frac{6709.50 [kN]}{1.00}} \right) \right\} = \min \{0.96, 0.94\} = 0.94$$

$$k_{yz} = 0.6 \times k_{zz} = 0.6 \times 0.40 = 0.24$$

$$k_{zy} = 0.6 \times k_{yy} = 0.6 \times 0.94 = 0.56$$

$$k_{zz} = \min \left\{ C_{mz} \times \left[ 1 + (\lambda_{rel,z} - 0.2) \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right], C_{mz} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right\}$$

$$= \min \left\{ 0.40 \times \left[ 1 + (0.61 - 0.2) \times \frac{111.36 [kN]}{0.89 \times \frac{6709.50 [kN]}{1.00}} \right], 0.40 \times \left( 1 + 0.8 \times \frac{111.36 [kN]}{0.89 \times \frac{6709.50 [kN]}{1.00}} \right) \right\} = \min \{0.40, 0.41\} = 0.40$$

$$\text{Unity check (6.61)} = \frac{|N_{Ed}|}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{yy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{yz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{|111.36[\text{kN}]|}{0.33 \times \frac{6709.50[\text{kN}]}{1.00}} + 0.94 \times \frac{306.60[\text{kNm}] + |0.00[\text{kNm}]|}{1.00 \times \frac{839.22[\text{kNm}]}{1.00}} + 0.24 \times \frac{|-5.14[\text{kNm}]| + |0.00[\text{kNm}]|}{\frac{839.22[\text{kNm}]}{1.00}} = \mathbf{0.39} \leq \mathbf{1.00} \quad (\text{EC3-1-1: 6.61})$$

$$\text{Unity check (6.62)} = \frac{|N_{Ed}|}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

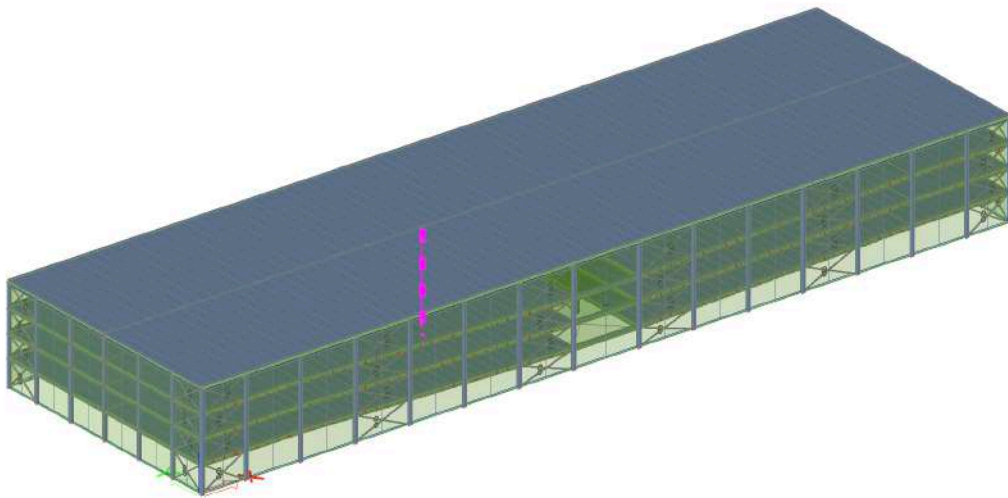
$$= \frac{|111.36[\text{kN}]|}{0.89 \times \frac{6709.50[\text{kN}]}{1.00}} + 0.56 \times \frac{|306.60[\text{kNm}]| + |0.00[\text{kNm}]|}{1.00 \times \frac{839.22[\text{kNm}]}{1.00}} + 0.40 \times \frac{|-5.14[\text{kNm}]| + |0.00[\text{kNm}]|}{\frac{839.22[\text{kNm}]}{1.00}} = \mathbf{0.23} \leq \mathbf{1.00} \quad (\text{EC3-1-1: 6.62})$$

Unity check = max(Unity check (6.61), Unity check (6.62)) = max(0.39, 0.23) = **0.39** ≤ **1.00**

The member satisfies the stability check.

**EC-EN 1993 Steel check ULS**

Values: **U<sub>Coverall</sub>**  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: B761



*Figure 2.2.3.3. 9: Interior column*



**EN 1993-1-1 Code Check**

National annex: Czech CSN-EN NA

<b>Member B761</b>	<b>12.600 / 12.600 m</b>	<b>SHS350/350/16.0</b>	<b>S 355</b>	<b>ULS-Set B (auto)</b>	<b>0.74 -</b>
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<b>Combination key</b>	
ULS-Set B (auto) / 1.15*LC1 + 1.15*LC2 + 1.50*LC3 + 1.50*LC4 + 0.90*3DWind8	

<b>Partial safety factors</b>	
γ <sub>M0</sub> for resistance of cross-sections	1.00
γ <sub>M1</sub> for resistance to instability	1.00
γ <sub>M2</sub> for resistance of net sections	1.25

<b>Material</b>			
Yield strength	f <sub>y</sub>	355.0	MPa
Ultimate strength	f <sub>u</sub>	490.0	MPa
Fabrication		Rolled	

....SECTION CHECK:....

The critical check is on position 12.600 m

Internal forces		Calculated	Unit
Normal force	N <sub>Ed</sub>	-2794.76	kN
Shear force	V <sub>y,Ed</sub>	-4.12	kN
Shear force	V <sub>z,Ed</sub>	12.98	kN
Torsion	T <sub>Ed</sub>	13.56	kNm
Bending moment	M <sub>y,Ed</sub>	10.69	kNm
Bending moment	M <sub>z,Ed</sub>	112.82	kNm

**Classification for cross-section design**

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	σ <sub>1</sub> [kN/m <sup>2</sup> ]	σ <sub>2</sub> [kN/m <sup>2</sup> ]	ψ [-]	k <sub>σ</sub> [-]	α [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	302	16	8.412e+04	-1.716e+05	0.5		1.0	18.9	22.8	27.7	37.6	1
3	I	302	16	1.767e+05	1.850e+05	1.0		1.0	18.9	22.8	27.7	31.4	1
5	I	302	16	1.808e+05	9.329e+04	0.5		1.0	18.9	22.8	27.7	37.2	1
7	I	302	16	8.821e+04	7.992e+04	0.9		1.0	18.9	22.8	27.7	32.0	1

**Note:** The Classification limits have been set according to Semi-Comp+. The cross-section is classified as Class 1

**Compression check**

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	2.1100e-02	m <sup>2</sup>
Compression resistance	N <sub>c,Rd</sub>	7490.50	kN
Unity check		0.37	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{2.1100 \cdot 10^{-2} [m^2] \times 355.0 [MPa]}{1.00} = 7490.50 [kN] \quad (EC3-1-1: 6.10)$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-2794.76 [kN]|}{7490.50 [kN]} = 0.37 \leq 1.00 \quad (EC3-1-1: 6.9)$$

**Bending moment check for M<sub>y</sub>**

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	W <sub>pl,y</sub>	2.6300e-03	m <sup>3</sup>
Plastic bending moment	M <sub>pl,y,Rd</sub>	933.65	kNm
Unity check		0.01	-

$$M_{pl,y,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{2.6300 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 933.65 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{|10.69 [kNm]|}{933.65 [kNm]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.12)$$

**Bending moment check for M<sub>z</sub>**

According to EN 1993-1-1 article 6.2.5 and formula (6.12),(6.13)

Plastic section modulus	W <sub>pl,z</sub>	2.6300e-03	m <sup>3</sup>
Plastic bending moment	M <sub>pl,z,Rd</sub>	933.65	kNm
Unity check		0.12	-

$$M_{pl,z,Rd} = \frac{W_{pl,z} \times f_y}{\gamma_{M0}} = \frac{2.6300 \cdot 10^{-3} [m^3] \times 355.0 [MPa]}{1.00} = 933.65 [kNm] \quad (EC3-1-1: 6.13)$$

$$\text{Unity check} = \frac{|M_{y,Ed}|}{M_{pl,y,Rd}} = \frac{112.82 [kNm]}{933.65 [kNm]} = 0.12 \leq 1.00 \quad (EC3-1-1: 6.12)$$

#### Shear check for $V_y$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	1.0550e-02	m <sup>2</sup>
Plastic shear resistance for $V_y$	$V_{pl,y,Rd}$	2162.32	kN
Unity check		0.00	-

$$V_{pl,y,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{1.0550 \cdot 10^{-2} [m^2] \times \frac{355.0 [MPa]}{\sqrt{3}}}{1.00} = 2162.32 [kN] \quad (EC3-1-1: 6.18)$$

$$\text{Unity check} = \frac{|V_{y,Ed}|}{V_{pl,y,Rd}} = \frac{|-4.12 [kN]|}{2162.32 [kN]} = 0.00 \leq 1.00 \quad (EC3-1-1: 6.17)$$

#### Shear check for $V_z$

According to EN 1993-1-1 article 6.2.6 and formula (6.17)

Shear correction factor	$\eta$	1.20	
Shear area	$A_v$	1.0550e-02	m <sup>2</sup>
Plastic shear resistance for $V_z$	$V_{pl,z,Rd}$	2162.32	kN
Unity check		0.01	-

$$V_{pl,z,Rd} = \frac{A_v \times \frac{f_y}{\sqrt{3}}}{\gamma_{M0}} = \frac{1.0550 \cdot 10^{-2} [m^2] \times \frac{355.0 [MPa]}{\sqrt{3}}}{1.00} = 2162.32 [kN] \quad (EC3-1-1: 6.18)$$

$$\text{Unity check} = \frac{|V_{z,Ed}|}{V_{pl,z,Rd}} = \frac{12.98 [kN]}{2162.32 [kN]} = 0.01 \leq 1.00 \quad (EC3-1-1: 6.17)$$

#### Torsion check

According to EN 1993-1-1 article 6.2.7 and formula (6.23)

Index of fibre	Fibre	1	
Total torsional moment	$T_{Ed}$	3.8	MPa
Elastic shear resistance	$T_{Rd}$	205.0	MPa
Unity check		0.02	-

$$\tau_{Ed} = \frac{|T_{Ed}| \times \gamma_{Ed,unit}}{\gamma_{M0}} = \frac{13557.47 \times 2.801 \cdot 10^{-1} [kN/m^2]}{1.00} = 3.8 [MPa]$$

$$\tau_{Rd} = \frac{f_y}{\sqrt{3} \times \gamma_{M0}} = \frac{355.0 [MPa]}{\sqrt{3} \times 1.00} = 205.0 [MPa]$$

$$\text{Unity check} = \frac{\tau_{Ed}}{\tau_{Rd}} = \frac{3.8 [MPa]}{205.0 [MPa]} = 0.02 \leq 1.00 \quad (EC3-1-1: 6.23)$$

**Note:** The unity check for torsion is lower than the limit value of 0.05. Therefore torsion is considered as insignificant and is ignored in the combined checks.

#### Combined bending, axial force and shear force check

According to EN 1993-1-1 article 6.2.9.1 and formula (6.41)

Design plastic moment resistance reduced due to $N_{Ed}$	$M_{N,y,Rd}$	764.69	kNm
Exponent of bending ratio $y$	$\alpha$	1.97	
Design plastic moment resistance reduced due to $N_{Ed}$	$M_{N,z,Rd}$	764.69	kNm
Exponent of bending ratio $z$	$\beta$	1.97	

$$\text{Unity check (6.41)} = 0.00 + 0.02 = 0.02 -$$

$$M_{N,y,Rd} = \min \left[ \frac{M_{pl,y,Rd} \times (1-n)}{1-0.5 \times \text{ratio}_{A,w}}, M_{pl,y,Rd} \right] = \min \left[ \frac{933.65 [kNm] \times (1-0.37)}{1-0.5 \times 0.47}, 933.65 [kNm] \right] = \min [764.69 [kNm], 933.65 [kNm]] = 764.69 [kNm] \quad (EC3-1-1: 6.39)$$

$$\alpha = \max \left[ \min \left( \frac{1.66}{1-1.13 \times n^2}, 6 \right), 1 \right] = \max \left[ \min \left( \frac{1.66}{1-1.13 \times 0.37^2}, 6 \right), 1 \right] = \max [\min (1.97, 6), 1] = 1.97$$

$$M_{N,z,Rd} = \min \left[ \frac{M_{pl,z,Rd} \times (1-n)}{1-0.5 \times \text{ratio}_{A,r}}, M_{pl,z,Rd} \right] = \min \left[ \frac{933.65 [kNm] \times (1-0.37)}{1-0.5 \times 0.47}, 933.65 [kNm] \right] = \min [764.69 [kNm], 933.65 [kNm]] = 764.69 [kNm] \quad (EC3-1-1: 6.40)$$

$$\beta = \max \left[ \min \left( \frac{1.66}{1-1.13 \times n^2}, 6 \right), 1 \right] = \max \left[ \min \left( \frac{1.66}{1-1.13 \times 0.37^2}, 6 \right), 1 \right] = \max [\min (1.97, 6), 1] = 1.97$$

$$\text{Unity check} = \left( \frac{|M_{y,Ed}|}{M_{N,y,Rd}} \right)^\alpha + \left( \frac{|M_{z,Ed}|}{M_{N,z,Rd}} \right)^\beta = \left( \frac{112.82 [kNm]}{764.69 [kNm]} \right)^{1.97} + \left( \frac{112.82 [kNm]}{764.69 [kNm]} \right)^{1.97} = 0.02 \leq 1.00 \quad (EC3-1-1: 6.41)$$

**Note:** Since the shear forces are less than half the plastic shear resistances their effect on the moment resistances is neglected.

The member satisfies the section check.

....:STABILITY CHECK:....

**Classification for member buckling design**

Decisive position for stability classification: 8.220 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\phi$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	302	16	4.134e+04	1.915e+05	0.2		1.0	18.9	22.8	27.7	42.5	1
3	I	302	16	1.994e+05	1.979e+05	1.0		1.0	18.9	22.8	27.7	31.0	1
5	I	302	16	1.899e+05	3.976e+04	0.2		1.0	18.9	22.8	27.7	42.6	1
7	I	302	16	3.188e+04	3.331e+04	1.0		1.0	18.9	22.8	27.7	31.4	1

**Note:** The Classification limits have been set according to Semi-Comp+.

The cross-section is classified as Class 1

**Flexural Buckling check**

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters		yy	zz	
Sway type		sway	non-sway	
System length	L	2.620	12.600	m
Buckling factor	k	1.39	0.70	
Buckling length	$l_{cr}$	3.651	8.820	m
Critical Euler load	$N_{cr}$	60559.46	10374.75	kN
Slenderness	$\lambda$	26.87	64.92	
Relative slenderness	$\lambda_{rel}$	0.35	0.85	
Limit slenderness	$\lambda_{rel,0}$	0.20	0.20	
Buckling curve	a	a	a	
Imperfection	$\alpha$	0.21	0.21	
Reduction factor	$\chi$	0.97	0.77	
Buckling resistance	$N_{b,Rd}$	7229.00	5738.48	kN

Flexural Buckling verification			
Cross-section area	A	2.1100e-02	m <sup>2</sup>
Buckling resistance	$N_{b,Rd}$	5738.48	kN
Unity check		0.49	-

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 3.8940 \cdot 10^{-4} [\text{m}^4]}{3.651 [\text{m}]^2} = 60559.45 [\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 3.8940 \cdot 10^{-4} [\text{m}^4]}{8.820 [\text{m}]^2} = 10374.75 [\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{3.651 [\text{m}]}{136 [\text{mm}]} = 26.87$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{8.820 [\text{m}]}{136 [\text{mm}]} = 64.92$$

$$\lambda_{rel,y} = \frac{\lambda_y}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{26.87}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{355.0 [\text{MPa}]}}} = 0.35 \quad (\text{EC3-1-1: 6.50})$$

$$\lambda_{rel,z} = \frac{\lambda_z}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{64.92}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{355.0 [\text{MPa}]}}} = 0.85 \quad (\text{EC3-1-1: 6.50})$$

$$\varphi_y = 0.5 \times [1 + \alpha_y \times (\lambda_{rel,y} - \lambda_{rel,y,0}) + \lambda_{rel,y}^2] = 0.5 \times [1 + 0.21 \times (0.35 - 0.20) + 0.35^2] = 0.58$$

$$\varphi_z = 0.5 \times [1 + \alpha_z \times (\lambda_{rel,z} - \lambda_{rel,z,0}) + \lambda_{rel,z}^2] = 0.5 \times [1 + 0.21 \times (0.85 - 0.20) + 0.85^2] = 0.93$$

$$\chi_y = \min \left( \frac{1}{\varphi_y + \sqrt{\varphi_y^2 - \lambda_{rel,y}^2}}, \frac{1}{\lambda_{rel,y}^2}, 1 \right) = \min \left( \frac{1}{0.58 + \sqrt{0.58^2 - 0.35^2}}, \frac{1}{0.35^2}, 1 \right) = \min(0.97, 8.08, 1) = 0.97 \quad (\text{EC3-1-1: 6.49})$$

$$\chi_z = \min \left( \frac{1}{\varphi_z + \sqrt{\varphi_z^2 - \lambda_{rel,z}^2}}, \frac{1}{\lambda_{rel,z}^2}, 1 \right) = \min \left( \frac{1}{0.93 + \sqrt{0.93^2 - 0.85^2}}, \frac{1}{0.85^2}, 1 \right) = \min(0.77, 1.39, 1) = 0.77 \quad (\text{EC3-1-1: 6.49})$$

$$N_{b,y,Rd} = \frac{\chi_y \times A \times f_y}{\gamma_{M1}} = \frac{0.97 \times 2.1100 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 7229.00 [\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,z,Rd} = \frac{\chi_z \times A \times f_y}{\gamma_{M1}} = \frac{0.77 \times 2.1100 \cdot 10^{-2} [\text{m}^2] \times 355.0 [\text{MPa}]}{1.00} = 5738.48 [\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,Rd} = \min(N_{b,y,Rd}, N_{b,z,Rd}) = \min(7229.00 [\text{kN}], 5738.48 [\text{kN}]) = 5738.48 [\text{kN}]$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{b,Rd}} = \frac{|-2794.76 [\text{kN}]|}{5738.48 [\text{kN}]} = 0.49 \leq 1.00 \quad (\text{EC3-1-1: 6.46})$$

### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** The cross-section concerns a RHS section which is not susceptible to Torsional(-Flexural) Buckling.

### Lateral Torsional Buckling check

According to EN 1993-1-1 article 6.3.2.1

**Note:** The cross-section concerns an RHS section with  $h/b < 10 / \lambda_{rel,z}$ . This section is thus not susceptible to Lateral Torsional Buckling.

### Bending and axial compression check

According to EN 1993-1-1 article 6.3.3 and formula (6.61),(6.62)

Bending and axial compression check parameters			
Interaction method		alternative method 2	
Cross-section area	A	2.1100e-02	m <sup>2</sup>
Plastic section modulus	W <sub>pl,y</sub>	2.6300e-03	m <sup>3</sup>
Plastic section modulus	W <sub>pl,z</sub>	2.6300e-03	m <sup>3</sup>
Design compression force	N <sub>Ed</sub>	2794.76	kN
Design bending moment (maximum)	M <sub>y,Ed</sub>	-23.32	kNm
Design bending moment (maximum)	M <sub>z,Ed</sub>	178.53	kNm
Characteristic compression resistance	N <sub>Rk</sub>	7490.50	kN
Characteristic moment resistance	M <sub>y,Rk</sub>	933.65	kNm
Characteristic moment resistance	M <sub>z,Rk</sub>	933.65	kNm
Reduction factor	χ <sub>y</sub>	0.97	
Reduction factor	χ <sub>z</sub>	0.77	
Reduction factor	χ <sub>LT</sub>	1.00	
Interaction factor	k <sub>yy</sub>	0.95	
Interaction factor	k <sub>yz</sub>	0.76	
Interaction factor	k <sub>zy</sub>	0.57	
Interaction factor	k <sub>zz</sub>	1.27	

Maximum moment M<sub>y,Ed</sub> is derived from beam B761 position 9.980 m.

Maximum moment M<sub>z,Ed</sub> is derived from beam B761 position 8.660 m.

Interaction method 2 parameters			
Method for interaction factors		Table B.1	
Sway type y		sway	
Equivalent moment factor	C <sub>my</sub>	0.90	
Resulting load type z		point load F	
End moment	M <sub>h,z</sub>	112.82	kNm
Field moment	M <sub>s,z</sub>	178.53	kNm
Factor	α <sub>h,z</sub>	0.63	
Ratio of end moments	ψ <sub>z</sub>	0.49	
Equivalent moment factor	C <sub>mz</sub>	0.96	
Resulting load type LT		point load F	
End moment	M <sub>h,LT</sub>	-39.67	kNm
Field moment	M <sub>s,LT</sub>	50.87	kNm
Factor	α <sub>h,LT</sub>	-0.78	
Ratio of end moments	ψ <sub>LT</sub>	-0.27	
Equivalent moment factor	C <sub>mLT</sub>	0.86	

Unity check (6.61) = 0.39 + 0.02 + 0.15 = 0.56 -

Unity check (6.62) = 0.49 + 0.01 + 0.24 = 0.74 -

C<sub>my</sub> = 0.90

$$\alpha_{h,z} = \frac{M_{h,z}}{M_{s,z}} = \frac{112.82[\text{kNm}]}{178.53[\text{kNm}]} = 0.63$$

$$C_{mz} = 0.9 + 0.1 \times \alpha_{h,z} = 0.9 + 0.1 \times 0.63 = 0.96$$

$$\alpha_{h,LT} = \frac{M_{h,LT}}{M_{s,LT}} = \frac{-39.67[\text{kNm}]}{50.87[\text{kNm}]} = -0.78$$

$$C_{mLT} = 0.9 + 0.1 \times \alpha_{h,LT} \times (1 + 2 \times \psi_{LT}) = 0.9 + 0.1 \times -0.78 \times (1 + 2 \times -0.27) = 0.86$$

$$N_{Rk} = A \times f_y = 2.1100 \cdot 10^{-2}[\text{m}^2] \times 355.0[\text{MPa}] = 7490.50[\text{kN}]$$

$$M_{y,Rk} = W_{pl,y} \times f_y = 2.6300 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}] = 933.65[\text{kNm}]$$

$$M_{z,Rk} = W_{pl,z} \times f_y = 2.6300 \cdot 10^{-3}[\text{m}^3] \times 355.0[\text{MPa}] = 933.65[\text{kNm}]$$

$$k_{yy} = \min \left\{ C_{my} \times \left[ 1 + (\lambda_{rel,y} - 0.2) \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right], C_{my} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right\}$$

$$= \min \left\{ 0.90 \times \left[ 1 + (0.35 - 0.2) \times \frac{2794.76[\text{kN}]}{0.97 \times \frac{7490.50[\text{kN}]}{1.00}} \right], 0.90 \times \left( 1 + 0.8 \times \frac{2794.76[\text{kN}]}{0.97 \times \frac{7490.50[\text{kN}]}{1.00}} \right) \right\} = \min \{0.95; 1.18\} = 0.95$$

$$k_{yz} = 0.6 \times k_{zz} = 0.6 \times 1.27 = 0.76$$

$$k_{zy} = 0.6 \times k_{yy} = 0.6 \times 0.95 = 0.57$$

$$k_{zz} = \min \left\{ C_{mz} \times \left[ 1 + (\lambda_{rel,z} - 0.2) \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right], C_{mz} \times \left( 1 + 0.8 \times \frac{N_{Ed}}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} \right) \right\}$$

$$= \min \left\{ 0.96 \times \left[ 1 + (0.85 - 0.2) \times \frac{2794.76[kN]}{0.77 \times \frac{7490.50[kN]}{1.00}} \right], 0.96 \times \left( 1 + 0.8 \times \frac{2794.76[kN]}{0.77 \times \frac{7490.50[kN]}{1.00}} \right) \right\} = \min \{ 1.27, 1.34 \} = 1.27$$

$$\text{Unity check (6.61)} = \frac{|N_{Ed}|}{\chi_y \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{yy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{|2794.76[kN]|}{0.97 \times \frac{7490.50[kN]}{1.00}} + 0.95 \times \frac{|-23.32[kNm]| + |0.00[kNm]|}{1.00 \times \frac{933.65[kNm]}{1.00}} + 0.76 \times \frac{|178.53[kNm]| + |0.00[kNm]|}{\frac{933.65[kNm]}{1.00}} = \mathbf{0.56 \leq 1.00}$$

(EC3-1-1: 6.61)

$$\text{Unity check (6.62)} = \frac{|N_{Ed}|}{\chi_z \times \frac{N_{Rk}}{\gamma_{M1}}} + k_{zy} \times \frac{|M_{y,Ed}| + |\Delta M_{y,Ed}|}{\chi_{LT} \times \frac{M_{y,Rk}}{\gamma_{M1}}} + k_{zz} \times \frac{|M_{z,Ed}| + |\Delta M_{z,Ed}|}{\frac{M_{z,Rk}}{\gamma_{M1}}}$$

$$= \frac{|2794.76[kN]|}{0.77 \times \frac{7490.50[kN]}{1.00}} + 0.57 \times \frac{|-23.32[kNm]| + |0.00[kNm]|}{1.00 \times \frac{933.65[kNm]}{1.00}} + 1.27 \times \frac{|178.53[kNm]| + |0.00[kNm]|}{\frac{933.65[kNm]}{1.00}} = \mathbf{0.74 \leq 1.00}$$

(EC3-1-1: 6.62)

Unity check = max (Unity check (6.61), Unity check (6.62)) = max(0.56, 0.74) = **0.74 ≤ 1.00**

The member satisfies the stability check.

### 2.2.3.3.5. Bracing Check

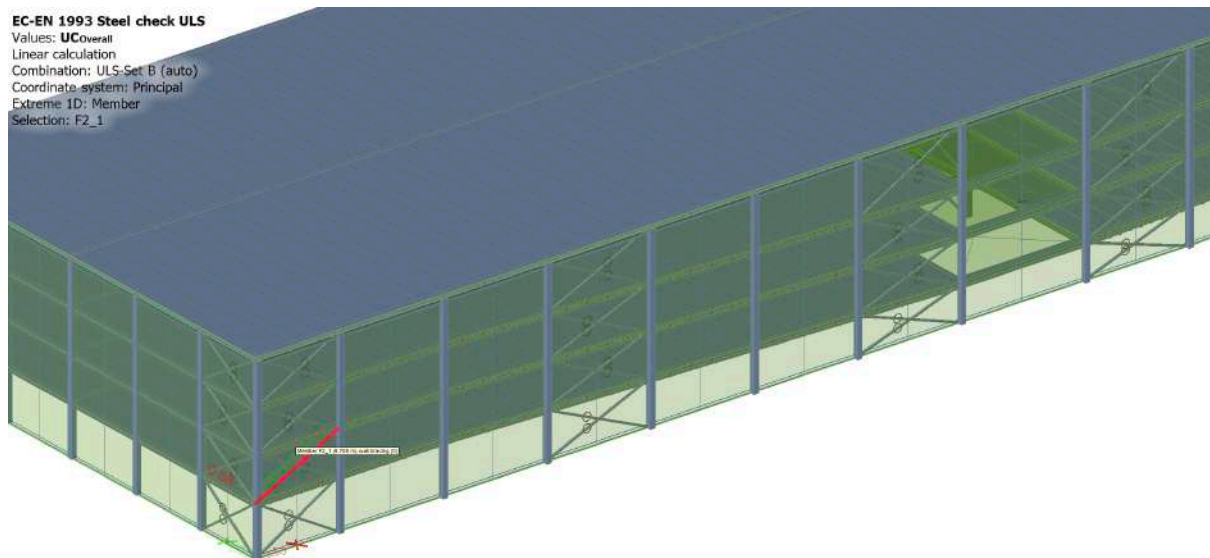


Figure 2.2.3.3. 10: Corner brace

## EC-EN 1993 Steel check ULS

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: F2\_1

### EN 1993-1-1 Code Check

National annex: Czech CSN-EN NA

Member F2_1	6.708 / 6.708 m	SHS110/110/14.2	S 275 JR (EN 10025-2)	ULS-Set B (auto)	0.58 -
-------------	-----------------	-----------------	-----------------------	------------------	--------

<b>Combination key</b>	
ULS-Set B (auto) / 1.15*LC1 + 1.15*LC2 + 1.05*LC3 + 1.05*LC4 + 1.50*3DWind4	

<b>Partial safety factors</b>	
$\gamma_{M0}$ for resistance of cross-sections	1.00
$\gamma_{M1}$ for resistance to instability	1.00
$\gamma_{M2}$ for resistance of net sections	1.25

<b>Material</b>			
Yield strength	$f_y$	275.0	MPa
Ultimate strength	$f_u$	410.0	MPa
Fabrication		Rolled	

....:SECTION CHECK:....

The critical check is on position 6.708 m

<b>Internal forces</b>	<b>Calculated</b>	<b>Unit</b>
Normal force	$N_{Ed}$	-184.29 kN
Shear force	$V_{y,Ed}$	0.00 kN
Shear force	$V_{z,Ed}$	0.00 kN
Torsion	$T_{Ed}$	0.00 kNm
Bending moment	$M_{y,Ed}$	0.00 kNm
Bending moment	$M_{z,Ed}$	0.00 kNm

### Classification for cross-section design

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\sigma$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
3	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
5	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
7	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1

**Note:** The Classification limits have been set according to Semi-Comp+.  
 The cross-section is classified as Class 1

### Compression check

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	5.2300e-03	m <sup>2</sup>
Compression resistance	$N_{c,Rd}$	1438.25	kN
Unity check		0.13	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{5.2300 \cdot 10^{-3} [\text{m}^2] \times 275.0 [\text{MPa}]}{1.00} = 1438.25 [\text{kN}] \quad (\text{EC3-1-1: 6.10})$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-184.29 [\text{kN}]|}{1438.25 [\text{kN}]} = 0.13 \leq 1.00 \quad (\text{EC3-1-1: 6.9})$$

The member satisfies the section check.

....:STABILITY CHECK:....

### Classification for member buckling design

Decisive position for stability classification: 6.708 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_\sigma$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
3	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
5	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
7	I	67	14	3.528e+04	3.528e+04	1.0		1.0	4.7	25.9	31.4	35.1	1

**Note:** The Classification limits have been set according to Semi-Comp+.  
 The cross-section is classified as Class 1

### Flexural Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters		yy	zz	
Sway type		sway	non-sway	
System length	L	6.708	6.708	m
Buckling factor	k	1.00	1.00	
Buckling length	$l_{cr}$	6.708	6.708	m
Critical Euler load	$N_{cr}$	357.41	357.43	kN
Slenderness	$\lambda$	174.15	174.15	
Relative slenderness	$\lambda_{rel}$	2.01	2.01	
Limit slenderness	$\lambda_{rel,0}$	0.20	0.20	
Buckling curve	a	a	a	
Imperfection	$\alpha$	0.21	0.21	
Reduction factor	$\chi$	0.22	0.22	
Buckling resistance	$N_{b,Rd}$	318.81	318.82	kN

Flexural Buckling verification			
Cross-section area	A	5.2300e-03	m <sup>2</sup>
Buckling resistance	$N_{b,Rd}$	318.81	kN
Unity check		0.58	-

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 7.7600 \cdot 10^{-6}[\text{m}^4]}{6.708[\text{m}]^2} = 357.41[\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0[\text{MPa}] \times 7.7600 \cdot 10^{-6}[\text{m}^4]}{6.708[\text{m}]^2} = 357.43[\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{6.708[\text{m}]}{39[\text{mm}]} = 174.15$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{6.708[\text{m}]}{39[\text{mm}]} = 174.15$$

$$\lambda_{rel,y} = \frac{\lambda_y}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{174.15}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{275.0[\text{MPa}]}}} = 2.01 \quad (\text{EC3-1-1: 6.50})$$

$$\lambda_{rel,z} = \frac{\lambda_z}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{174.15}{\pi \times \sqrt{\frac{210000.0[\text{MPa}]}{275.0[\text{MPa}]}}} = 2.01 \quad (\text{EC3-1-1: 6.50})$$

$$\varphi_y = 0.5 \times [1 + \alpha_y \times (\lambda_{rel,y} - \lambda_{rel,y,0}) + \lambda_{rel,y}^2] = 0.5 \times [1 + 0.21 \times (2.01 - 0.20) + 2.01^2] = 2.70$$

$$\varphi_z = 0.5 \times [1 + \alpha_z \times (\lambda_{rel,z} - \lambda_{rel,z,0}) + \lambda_{rel,z}^2] = 0.5 \times [1 + 0.21 \times (2.01 - 0.20) + 2.01^2] = 2.70$$

$$\chi_y = \min \left( \frac{1}{\varphi_y + \sqrt{\varphi_y^2 - \lambda_{rel,y}^2}}, \frac{1}{\lambda_{rel,y}^2}, 1 \right) = \min \left( \frac{1}{2.70 + \sqrt{2.70^2 - 2.01^2}}, \frac{1}{2.01^2}, 1 \right) = \min(0.22, 0.25, 1) = 0.22 \quad (\text{EC3-1-1: 6.49})$$

$$\chi_z = \min \left( \frac{1}{\varphi_z + \sqrt{\varphi_z^2 - \lambda_{rel,z}^2}}, \frac{1}{\lambda_{rel,z}^2}, 1 \right) = \min \left( \frac{1}{2.70 + \sqrt{2.70^2 - 2.01^2}}, \frac{1}{2.01^2}, 1 \right) = \min(0.22, 0.25, 1) = 0.22 \quad (\text{EC3-1-1: 6.49})$$

$$N_{b,y,Rd} = \frac{\chi_y \times A \times f_y}{\gamma_{M1}} = \frac{0.22 \times 5.2300 \cdot 10^{-3}[\text{m}^2] \times 275.0[\text{MPa}]}{1.00} = 318.81[\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,z,Rd} = \frac{\chi_z \times A \times f_y}{\gamma_{M1}} = \frac{0.22 \times 5.2300 \cdot 10^{-3}[\text{m}^2] \times 275.0[\text{MPa}]}{1.00} = 318.82[\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,Rd} = \min(N_{b,y,Rd}, N_{b,z,Rd}) = \min(318.81[\text{kN}], 318.82[\text{kN}]) = 318.81[\text{kN}]$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{b,Rd}} = \frac{|-184.29[\text{kN}]|}{318.81[\text{kN}]} = 0.58 \leq 1.00 \quad (\text{EC3-1-1: 6.46})$$

### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** The cross-section concerns a RHS section which is not susceptible to Torsional(-Flexural) Buckling.

The member satisfies the stability check.

**EC-EN 1993 Steel check ULS**  
 Values: U<sub>Coverall</sub>  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: B866

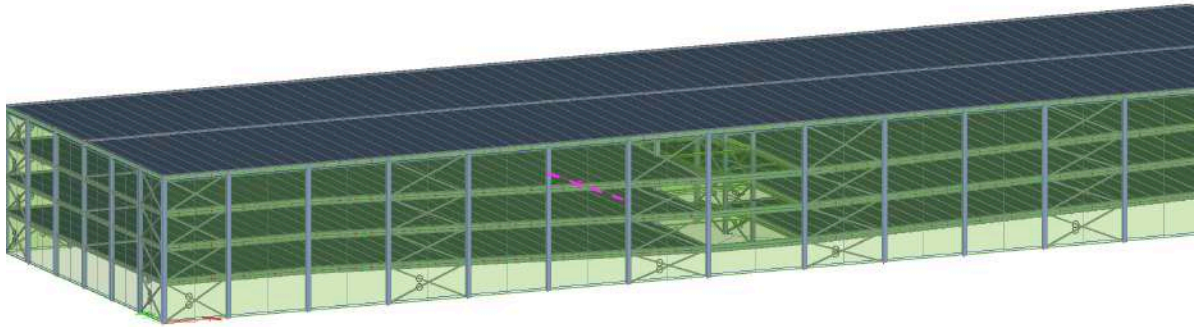


Figure 2.2.3.3. 11: Internal brace

**EC-EN 1993 Steel check ULS**

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Member  
 Selection: B866

**EN 1993-1-1 Code Check**

National annex: Czech CSN-EN NA

<b>Member B866</b>	<b>8.078 / 8.078 m</b>	<b>SHS110/110/14.2</b>	<b>S 275 JR (EN 10025-2)</b>	<b>ULS-Set B (auto)</b>	<b>0.81 -</b>
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<b>Combination key</b>	
ULS-Set B (auto) / 1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind16	

<b>Partial safety factors</b>	
γ <sub>M0</sub> for resistance of cross-sections	1.00
γ <sub>M1</sub> for resistance to instability	1.00
γ <sub>M2</sub> for resistance of net sections	1.25

<b>Material</b>		
Yield strength	f <sub>y</sub>	275.0 MPa
Ultimate strength	f <sub>u</sub>	410.0 MPa
Fabrication		Rolled

...:SECTION CHECK:...:

The critical check is on position 8.078 m

Internal forces		Calculated	Unit
Normal force	N <sub>Ed</sub>	-183.41	kN
Shear force	V <sub>y,Ed</sub>	0.00	kN
Shear force	V <sub>z,Ed</sub>	0.00	kN
Torsion	T <sub>Ed</sub>	0.00	kNm
Bending moment	M <sub>y,Ed</sub>	0.00	kNm
Bending moment	M <sub>z,Ed</sub>	0.00	kNm

**Classification for cross-section design**

Classification according to EN 1993-1-1 article 5.5.2  
 Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2



Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_{\sigma}$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
3	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
5	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
7	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1

**Note:** The Classification limits have been set according to Semi-Comp+.  
The cross-section is classified as Class 1

#### Compression check

According to EN 1993-1-1 article 6.2.4 and formula (6.9)

Cross-section area	A	5.2300e-03	m <sup>2</sup>
Compression resistance	$N_{c,Rd}$	1438.25	kN
Unity check		0.13	-

$$N_{c,Rd} = \frac{A \times f_y}{\gamma_{M0}} = \frac{5.2300 \cdot 10^{-3} [\text{m}^2] \times 275.0 [\text{MPa}]}{1.00} = 1438.25 [\text{kN}] \quad (\text{EC3-1-1: 6.10})$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{c,Rd}} = \frac{|-183.41 [\text{kN}]|}{1438.25 [\text{kN}]} = \mathbf{0.13} \leq \mathbf{1.00} \quad (\text{EC3-1-1: 6.9})$$

The member satisfies the section check.

...:STABILITY CHECK:...:

#### Classification for member buckling design

Decisive position for stability classification: 8.078 m

Classification according to EN 1993-1-1 article 5.5.2

Classification of Internal and Outstand parts according to EN 1993-1-1 Table 5.2 Sheet 1 & 2

Id	Type	c [mm]	t [mm]	$\sigma_1$ [kN/m <sup>2</sup> ]	$\sigma_2$ [kN/m <sup>2</sup> ]	$\Psi$ [-]	$k_{\sigma}$ [-]	$\alpha$ [-]	c/t [-]	Class 1 Limit [-]	Class 2 Limit [-]	Class 3 Limit [-]	Class
1	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
3	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
5	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1
7	I	67	14	3.511e+04	3.511e+04	1.0		1.0	4.7	25.9	31.4	35.1	1

**Note:** The Classification limits have been set according to Semi-Comp+.  
The cross-section is classified as Class 1

#### Flexural Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

Buckling parameters		yy	zz	
Sway type		sway	non-sway	
System length	L	8.078	8.078	m
Buckling factor	k	1.00	1.00	
Buckling length	$l_{cr}$	8.078	8.078	m
Critical Euler load	$N_{cr}$	246.49	246.50	kN
Slenderness	$\lambda$	209.71	209.70	
Relative slenderness	$\lambda_{rel}$	2.42	2.42	
Limit slenderness	$\lambda_{rel,0}$	0.20	0.20	
Buckling curve	a	a	a	
Imperfection	$\alpha$	0.21	0.21	
Reduction factor	$\chi$	0.16	0.16	
Buckling resistance	$N_{b,Rd}$	225.20	225.21	kN

#### Flexural Buckling verification

Cross-section area	A	5.2300e-03	m <sup>2</sup>
Buckling resistance	$N_{b,Rd}$	225.20	kN
Unity check		0.81	-

$$N_{cr,y} = \frac{\pi^2 \times E \times I_y}{l_{cr,y}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 7.7600 \cdot 10^{-6} [\text{m}^4]}{8.078 [\text{m}]^2} = 246.49 [\text{kN}]$$

$$N_{cr,z} = \frac{\pi^2 \times E \times I_z}{l_{cr,z}^2} = \frac{\pi^2 \times 210000.0 [\text{MPa}] \times 7.7600 \cdot 10^{-6} [\text{m}^4]}{8.078 [\text{m}]^2} = 246.50 [\text{kN}]$$

$$\lambda_y = \frac{l_{cr,y}}{i_y} = \frac{8.078 [\text{m}]}{39 [\text{mm}]} = 209.71$$

$$\lambda_z = \frac{l_{cr,z}}{i_z} = \frac{8.078 [\text{m}]}{39 [\text{mm}]} = 209.70$$

$$\lambda_{rel,y} = \frac{\lambda_y}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{209.71}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{275.0 [\text{MPa}]}}} = 2.42 \quad (\text{EC3-1-1: 6.50})$$

$$\lambda_{rel,z} = \frac{\lambda_z}{\pi \times \sqrt{\frac{E}{f_y}}} = \frac{209.70}{\pi \times \sqrt{\frac{210000.0 [\text{MPa}]}{275.0 [\text{MPa}]}}} = 2.42 \quad (\text{EC3-1-1: 6.50})$$

$$\varphi_y = 0.5 \times [1 + \alpha_y \times (\lambda_{rel,y} - \lambda_{rel,y,0}) + \lambda_{rel,y}^2] = 0.5 \times [1 + 0.21 \times (2.42 - 0.20) + 2.42^2] = 3.65$$

$$\varphi_z = 0.5 \times [1 + \alpha_z \times (\lambda_{rel,z} - \lambda_{rel,z,0}) + \lambda_{rel,z}^2] = 0.5 \times [1 + 0.21 \times (2.42 - 0.20) + 2.42^2] = 3.65$$

$$\chi_y = \min \left( \frac{1}{\varphi_y + \sqrt{\varphi_y^2 - \lambda_{rel,y}^2}}, \frac{1}{\lambda_{rel,y}^2}, 1 \right) = \min \left( \frac{1}{3.65 + \sqrt{3.65^2 - 2.42^2}}, \frac{1}{2.42^2}, 1 \right) = \min(0.16, 0.17, 1) = 0.16 \quad (\text{EC3-1-1: 6.49})$$

$$\chi_z = \min \left( \frac{1}{\varphi_z + \sqrt{\varphi_z^2 - \lambda_{rel,z}^2}}, \frac{1}{\lambda_{rel,z}^2}, 1 \right) = \min \left( \frac{1}{3.65 + \sqrt{3.65^2 - 2.42^2}}, \frac{1}{2.42^2}, 1 \right) = \min(0.16, 0.17, 1) = 0.16 \quad (\text{EC3-1-1: 6.49})$$

$$N_{b,y,Rd} = \frac{\chi_y \times A \times f_y}{\gamma_{M1}} = \frac{0.16 \times 5.2300 \cdot 10^{-3} [\text{m}^2] \times 275.0 [\text{MPa}]}{1.00} = 225.20 [\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,z,Rd} = \frac{\chi_z \times A \times f_y}{\gamma_{M1}} = \frac{0.16 \times 5.2300 \cdot 10^{-3} [\text{m}^2] \times 275.0 [\text{MPa}]}{1.00} = 225.21 [\text{kN}] \quad (\text{EC3-1-1: 6.47})$$

$$N_{b,Rd} = \min(N_{b,y,Rd}, N_{b,z,Rd}) = \min(225.20 [\text{kN}], 225.21 [\text{kN}]) = 225.20 [\text{kN}]$$

$$\text{Unity check} = \frac{|N_{Ed}|}{N_{b,Rd}} = \frac{|-183.41 [\text{kN}]|}{225.20 [\text{kN}]} = 0.81 \leq 1.00 \quad (\text{EC3-1-1: 6.46})$$

#### Torsional(-Flexural) Buckling check

According to EN 1993-1-1 article 6.3.1.1 and formula (6.46)

**Note:** The cross-section concerns a RHS section which is not susceptible to Torsional(-Flexural) Buckling.

The member satisfies the stability check.

## 2.2.3.4. Results

### 2.2.3.4.1. Stress Distribution

#### 1D stresses

Values:  $\sigma_x$

Linear calculation

Combination: ULS-Set B (auto)

Coordinate system: Principal

Extreme 1D: Global

Selection: All

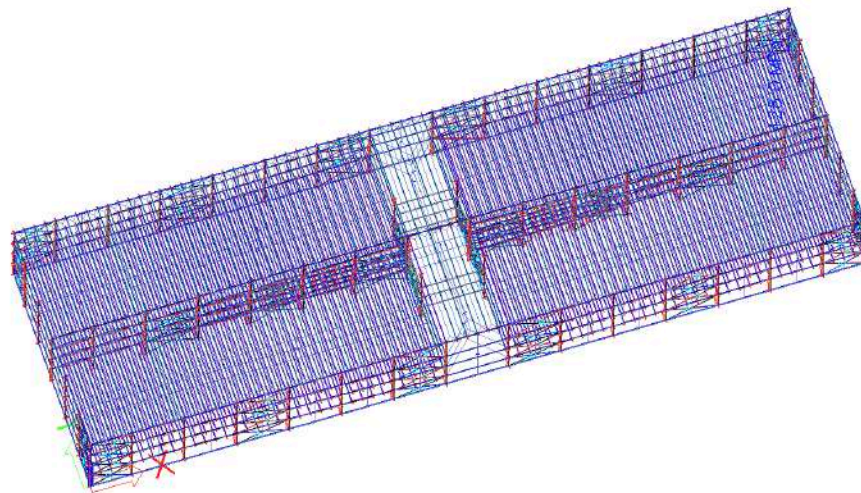


Figure 2.2.3.4. 1: Stress distribution

The maximum and minimum stress distribution on the structure is shown above. The red colour represents compression and blue represents tension. The stress values are in the acceptable range.

### 1D stresses

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: All

Name	dx [m]	Fibre	Case	$\sigma_x$ [MPa]	$\tau_{xy}$ [MPa]	$\tau_{xz}$ [MPa]	$\tau_{tor}$ [MPa]
B769	8.660+	8	ULS-Set B (auto)/1	<b>-510.5</b>	-1.4	-42.7	-15.3
Bx28_3	4.500+	3	ULS-Set B (auto)/2	<b>428.0</b>	0.0	0.0	0.0

Name	Combination key
ULS-Set B (auto)/1	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind16
ULS-Set B (auto)/2	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind12

### 2.2.3.4.2. Deformation

#### 1D deformations

Values:  $u_y$   
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: All

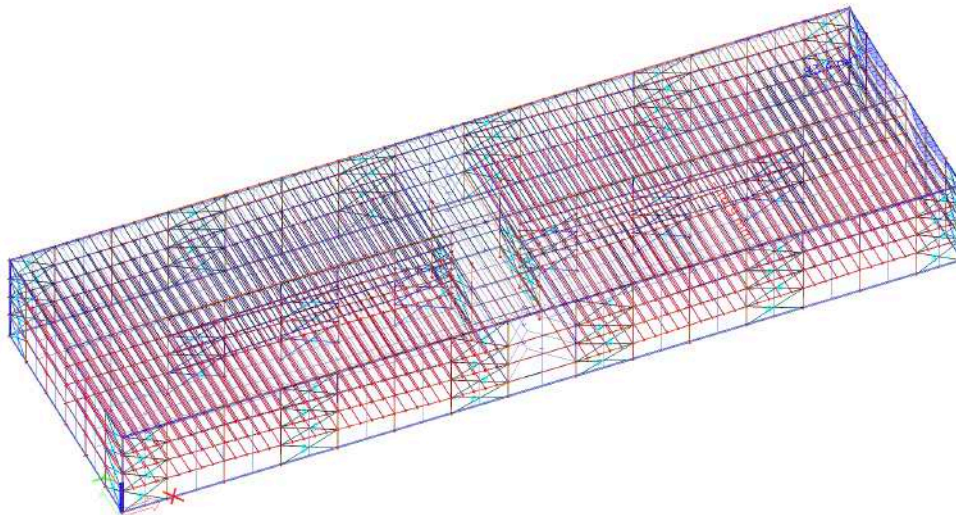


Figure 2.2.3.4. 2: Deformation  $u_y$

**1D deformations**  
Values:  $u_x$   
Linear calculation  
Combination: ULS-Set B (auto)  
Coordinate system: Principal  
Extreme 1D: Global  
Selection: All

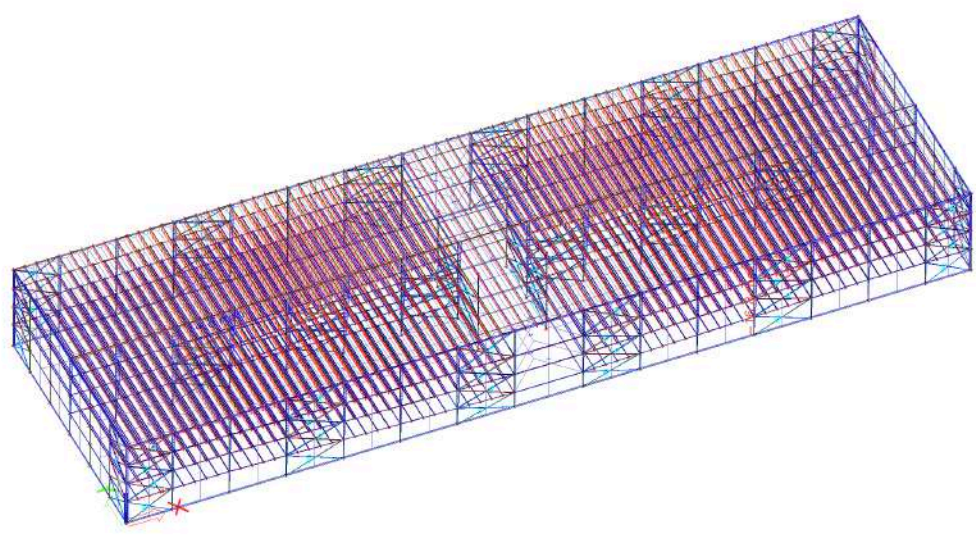


Figure 2.2.3.4. 3: Deformation  $u_x$

**1D deformations**  
Values:  $u_z$   
Linear calculation  
Combination: ULS-Set B (auto)  
Coordinate system: Principal  
Extreme 1D: Global  
Selection: All

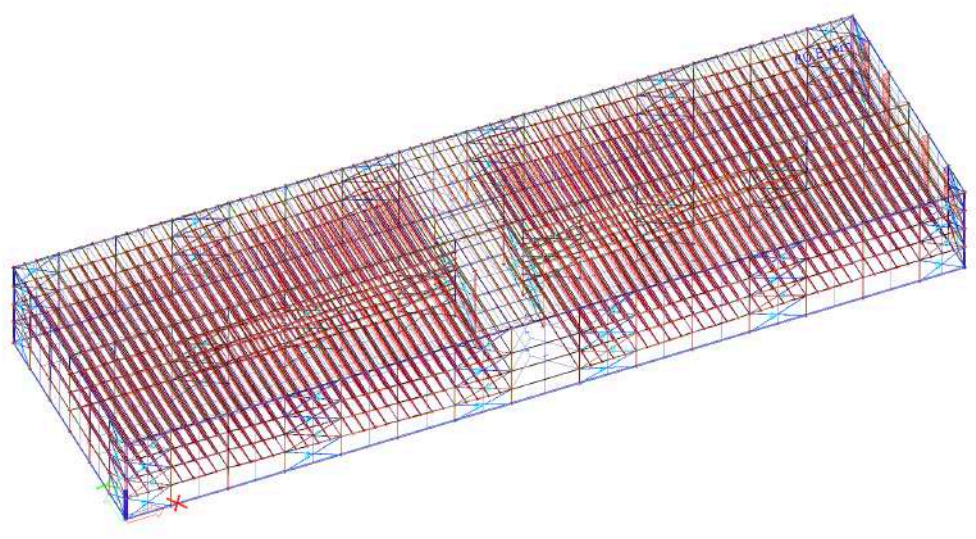


Figure 2.2.3.4. 4: Deformation  $u_z$

## 1D deformations

Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Global  
 Selection: All  
**Deformations**

Name	dx [m]	Case	u <sub>x</sub> [mm]	u <sub>y</sub> [mm]	u <sub>z</sub> [mm]	φ <sub>x</sub> [mrad]	φ <sub>y</sub> [mrad]	φ <sub>z</sub> [mrad]	U <sub>total</sub> [mm]
B1056	17.000	ULS-Set B (auto)/1	<b>-16.8</b>	0.0	-6.6	-0.3	-3.6	-0.2	18.0
B763	0.000	ULS-Set B (auto)/2	<b>14.8</b>	9.0	2.4	0.0	0.3	-1.6	17.5
Bx12_4	7.500	ULS-Set B (auto)/1	0.0	<b>-16.9</b>	-6.4	3.6	-0.3	-0.2	18.1
By43_4	2.571	ULS-Set B (auto)/3	4.0	<b>33.4</b>	0.3	-1.1	-0.1	-1.3	33.5
B1413	8.947	ULS-Set B (auto)/4	-10.4	1.7	<b>-34.4</b>	-2.1	-0.3	0.0	35.1
F48_1	5.883	ULS-Set B (auto)/5	-3.7	2.3	<b>30.8</b>	0.0	0.0	0.0	31.0
Bx55_3	4.500	ULS-Set B (auto)/2	-0.8	-7.8	-10.1	<b>-12.5</b>	-0.7	0.2	12.8
B1070	0.000	ULS-Set B (auto)/6	-2.9	1.9	-26.3	<b>12.1</b>	-0.7	0.0	26.5
Bx28_3	7.500	ULS-Set B (auto)/7	1.9	2.4	-10.7	-0.4	<b>-15.8</b>	-0.5	11.1
Bx28_2	0.000	ULS-Set B (auto)/8	0.8	2.2	-9.5	-1.1	<b>15.2</b>	-0.3	9.8
By41_4	2.571	ULS-Set B (auto)/9	3.9	39.3	-1.9	-1.2	0.7	<b>-16.4</b>	39.5
By42_4	3.429	ULS-Set B (auto)/10	4.3	35.7	-2.2	-0.9	-0.8	<b>16.7</b>	36.0
B1413	8.947	ULS-Set B (auto)/1	-11.3	1.5	-34.4	-2.1	-0.3	0.0	<b>35.2</b>

Name	Combination key
ULS-Set B (auto)/1	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind12
ULS-Set B (auto)/2	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind4
ULS-Set B (auto)/3	LC1 + LC2 + 1.05*LC3 + 1.50*3DWind12
ULS-Set B (auto)/4	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind8
ULS-Set B (auto)/5	1.15*LC1 + 1.15*LC2 + 1.05*LC3 + 1.05*LC4 + 1.50*3DWind12
ULS-Set B (auto)/6	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 1.05*LC4 + 0.90*3DWind16
ULS-Set B (auto)/7	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 0.90*3DWind16
ULS-Set B (auto)/8	1.35*LC1 + 1.35*LC2 + 1.05*LC3 + 0.90*3DWind4
ULS-Set B (auto)/9	1.15*LC1 + 1.15*LC2 + 1.05*LC4 + 1.50*3DWind12
ULS-Set B (auto)/10	1.15*LC1 + 1.15*LC2 + 1.05*LC3 + 1.50*3DWind12

Table 2.2.3.4. 1: Extreme values of deformations

### 2.2.3.4.3. Internal Forces and Reactions

**1D internal forces**  
 Values:  $M_y$   
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Cross-section  
 Selection: All

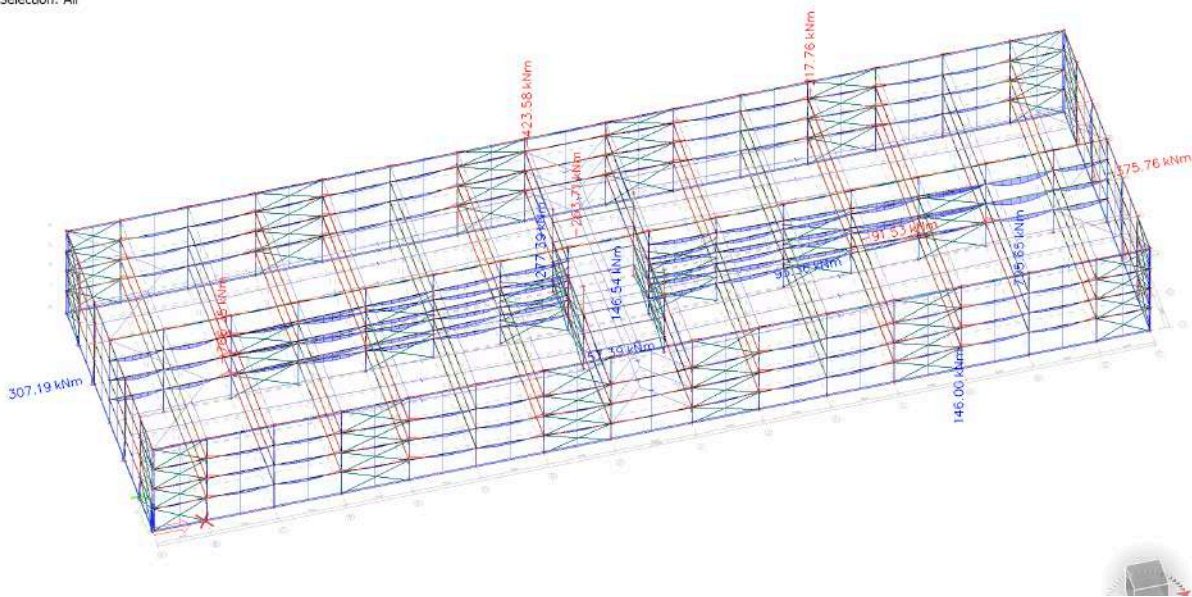


Figure 2.2.3.4. 5:  $M_y$  bending moments

**1D internal forces**  
 Values:  $M_x$   
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Cross-section  
 Selection: All

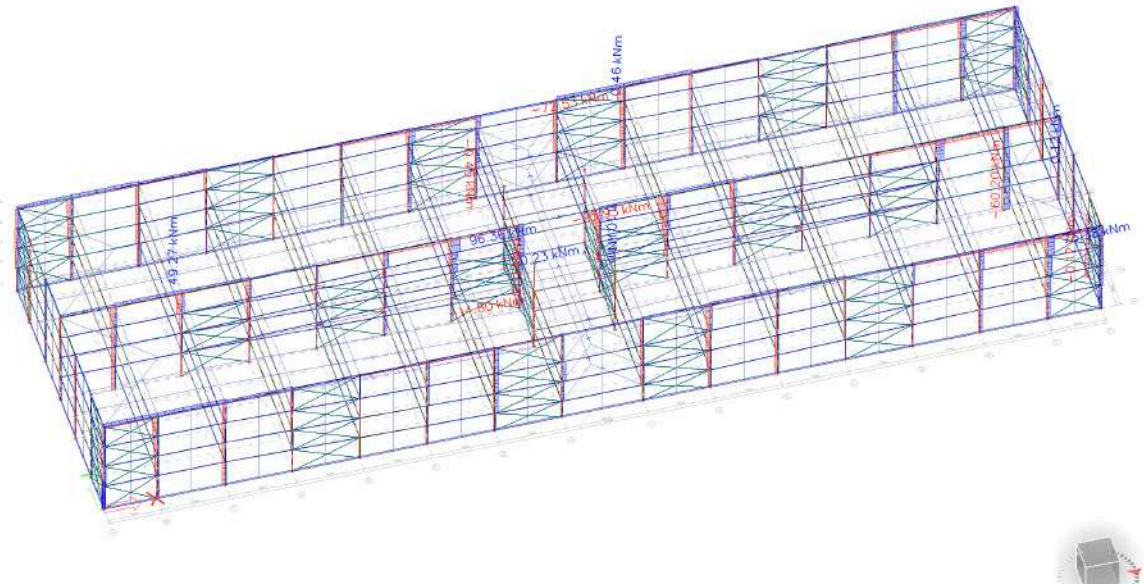


Figure 2.2.3.4. 6:  $M_x$  bending moments

**1D internal forces**  
 Values: N  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 Coordinate system: Principal  
 Extreme 1D: Cross-section  
 Selection: All

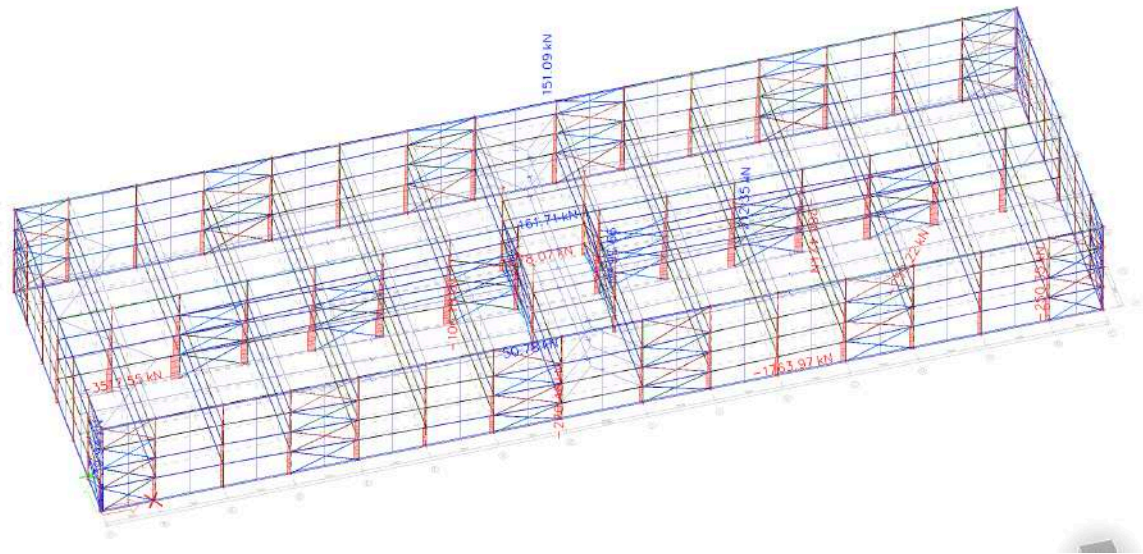


Figure 2.2.3.4. 7:N axial forces

**Reactions**  
 Values: Rz  
 Linear calculation  
 Combination: ULS-Set B (auto)  
 System: Global  
 Extreme: Member  
 Selection: All

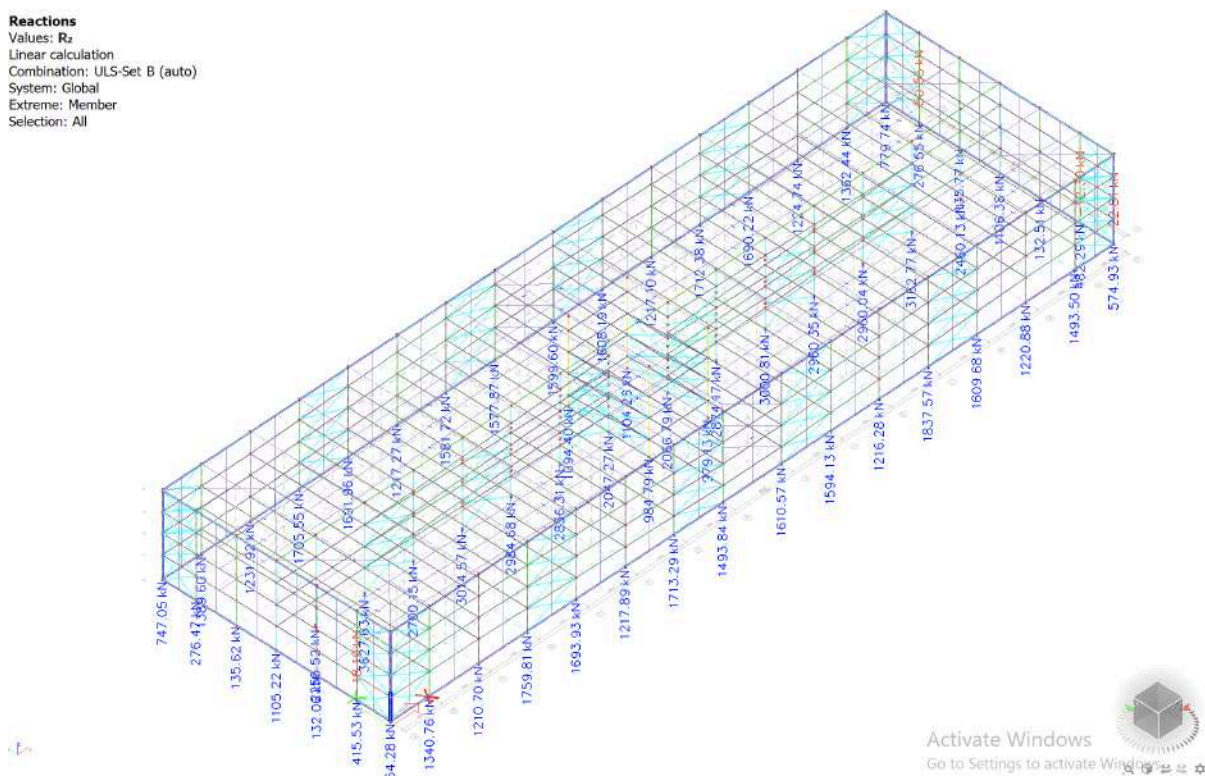


Figure 2.2.3.4. 8:Reactions

The cross-sectional extreme internal forces are shown below table.

Name	dx [m]	Case	Cross-section	N [kN]	Vy [kN]	Vz [kN]	Mx [kNm]	My [kNm]	Mz [kNm]
Bx26_3	0	ULS-Set B (auto)/1	CS12-Beam2 - HEA600	-284.42	0	180.08	0.02	0	0
Bx39_1	7.503	ULS-Set B (auto)/2	CS12-Beam2 - HEA600	212.35	0	-215.94	0.03	0	0
Bx52_3	0	ULS-Set B (auto)/2	CS12-Beam2 - HEA600	-90.95	-11.18	219.49	0.46	-282.02	34.58
Bx15_3	0	ULS-Set B (auto)/3	CS12-Beam2 - HEA600	-130.5	16.41	164.7	-0.34	-207.39	-28.85
Bx28_1	0	ULS-Set B (auto)/4	CS12-Beam2 - HEA600	-31.34	0.5	520.76	-0.18	-723.11	-3.75
Bx17_1	7.5	ULS-Set B (auto)/5	CS12-Beam2 - HEA600	-25.27	0.28	-526.52	0.07	-766.25	2.09
Bx28_2	3.75	ULS-Set B (auto)/6	CS12-Beam2 - HEA600	1.59	0	0	-0.27	795.65	0
Bx50_3	0	ULS-Set B (auto)/2	CS12-Beam2 - HEA600	-91.1	11.19	213.55	-0.45	-262.41	-51.1
Bx15_3	6	ULS-Set B (auto)/1	CS12-Beam2 - HEA600	-129.57	16.39	-124.58	-0.36	-86.2	70.15

Name	dx [m]	Case	Cross-section	N [kN]	Vy [kN]	Vz [kN]	Mx [kNm]	My [kNm]	Mz [kNm]
By21_3	0	ULS-Set B (auto)/13	CS13_Secondary beam2 - HEA320	-98.45	0	4.14	-0.11	0	0
By24_3	0	ULS-Set B (auto)/2	CS13_Secondary beam2 - HEA320	-119.28	-1.83	115.16	0.1	-198.95	7.04
By22_3	11	ULS-Set B (auto)/15	CS13_Secondary beam2 - HEA320	-71.75	-1.24	-150.74	0.06	-423.58	-13.59
By22_3	4.583	ULS-Set B (auto)/5	CS13_Secondary beam2 - HEA320	-107.82	-0.74	-10.07	0.06	277.39	-3.41
By20_3	6.013	ULS-Set B (auto)/2	CS13_Secondary beam2 - HEA320	-182.57	-5.61	-70.48	0.09	-66.84	-22.45
By23_3	11	ULS-Set B (auto)/2	CS13_Secondary beam2 - HEA320	-251.66	2.71	-194.44	-0.05	-379.42	13.76
B763	12.6	ULS-Set B (auto)/2	CS1-Column1 - SHS350/350/16.0	-3517.55	19.8	-5.99	-13.16	-10	205.42
B772	0	ULS-Set B (auto)/16	CS1-Column1 - SHS350/350/16.0	-3.75	-29	-18.89	40.13	27.92	18.86
B770	2.880+	ULS-Set B (auto)/17	CS1-Column1 - SHS350/350/16.0	-585.64	-101.83	12.65	0.36	-3.63	145.51
B770	9.320+	ULS-Set B (auto)/1	CS1-Column1 - SHS350/350/16.0	-2388.86	61.04	2.85	3.07	-9.44	145.2
B769	8.660+	ULS-Set B (auto)/2	CS1-Column1 - SHS350/350/16.0	-2460.93	-13.9	-308.02	-64.84	78.94	282.21
B767	0	ULS-Set B (auto)/2	CS1-Column1 - SHS350/350/16.0	-292.7	-15.97	-51.62	-95.93	53.57	57.59
B761	0	ULS-Set B (auto)/2	CS1-Column1 - SHS350/350/16.0	-292.34	-16.72	37.76	96.36	-32.32	56.76
B769	8.660+	ULS-Set B (auto)/5	CS1-Column1 - SHS350/350/16.0	-1790.43	-32.64	-55.35	9.67	-91.53	48.66
B769	9.540+	ULS-Set B (auto)/4	CS1-Column1 - SHS350/350/16.0	-2531.41	16.23	-45.26	-0.05	93.36	247.84
B770	12.6	ULS-Set B (auto)/11	CS1-Column1 - SHS350/350/16.0	-1105.34	-75.22	3.03	-0.94	3.64	-247.65
B770	9.100+	ULS-Set B (auto)/2	CS1-Column1 - SHS350/350/16.0	-2468.93	20.13	216.56	69.89	18.95	357.83
B788	12.6	ULS-Set B (auto)/8	CS3-Column3 - SHS350/350/16.0	-1763.97	-45.21	-16.87	9.58	-25.92	87.53
B783	0	ULS-Set B (auto)/4	CS3-Column3 - SHS350/350/16.0	50.78	-6.89	11.13	41.18	-12.46	2.84
B778	4.200+	ULS-Set B (auto)/2	CS3-Column3 - SHS350/350/16.0	-679.14	-185.65	-31.65	-22.17	49.01	349.38
B803	2.000+	ULS-Set B (auto)/2	CS3-Column3 - SHS350/350/16.0	-717.58	115.73	-25.67	-39.7	36.89	-156.72
B804	0	ULS-Set B (auto)/2	CS3-Column3 - SHS350/350/16.0	-117.76	-16.65	-38.29	-72.53	25.4	20.31
B791	0	ULS-Set B (auto)/1	CS3-Column3 - SHS350/350/16.0	-52.96	-44.94	25.28	73.38	-32.78	25.84
B820	12.6	ULS-Set B (auto)/3	CS3-Column3 - SHS350/350/16.0	-52.82	2.19	-134.12	-25.18	-375.76	7.94
B812	12.6	ULS-Set B (auto)/18	CS3-Column3 - SHS350/350/16.0	-55.59	-1.91	109.17	14.21	307.19	-5.42
B797	12.6	ULS-Set B (auto)/11	CS3-Column3 - SHS350/350/16.0	-666.53	-108.01	-2.76	2.47	-2.99	-266.39
B817	12.6	ULS-Set B (auto)/2	CS4-Column4 - SHS350/350/16.0	-1078.07	64.14	-6.43	5.29	8.97	42.88
B818	5.000+	ULS-Set B (auto)/15	CS4-Column4 - SHS350/350/16.0	-466.32	-66.45	-6.04	-2.57	39.02	104.38
B815	10.200+	ULS-Set B (auto)/8	CS4-Column4 - SHS350/350/16.0	-922.03	24.84	-8.42	-10.02	48.77	-45.26
B817	2.000+	ULS-Set B (auto)/2	CS4-Column4 - SHS350/350/16.0	-129.65	12.68	0.76	-1.21	13.26	22.1
B815	4.200+	ULS-Set B (auto)/4	CS4-Column4 - SHS350/350/16.0	-295.74	72.02	-5.73	-14.8	31.72	-125.36
B818	11.000+	ULS-Set B (auto)/17	CS4-Column4 - SHS350/350/16.0	-723.37	26.5	-5.45	10.23	35.98	-38.31
B815	0	ULS-Set B (auto)/19	CS4-Column4 - SHS350/350/16.0	0	0	0	0	0	0
B816	10.200+	ULS-Set B (auto)/20	CS4-Column4 - SHS350/350/16.0	-865.64	31.01	-6.5	-8.22	57.39	-52.04
B816	4.200+	ULS-Set B (auto)/2	CS4-Column4 - SHS350/350/16.0	-282.13	75.77	-4.54	-7.31	32.27	-132.32

Table 2.2.3.4. 2: Extreme internal forces

### 3. Conclusion

The study of multi-storey steel carpark design is focused on designing process , calculation and analysis of framed structure both by hand calculation and software.

The slab floor system used in study makes construction faster ,besides it provides structural stability during the construction. The interior sloped floor system design is provided to stay in Eurocode boundary while designing interior ramp.

The study provides calculations which can be utilized in the future design. Besides the study shows material quantity that are used in design in order to give an idea about financial cost.



PROFILE	MATERIAL	NUMBER	LENGTH [mm]	WEIGHT [kg/m]	TOTAL WEIGHT [kg]
SHS 350*350*16	S355	43	12600	165.6	89722.08
		2	12850	165.6	4255.92
		2	13150	165.6	4355.28
		16	13450	165.6	35637.12
SHS 110*110*14.2	S275JR	104	7500	41	31980
HEA 600	S355	60	7550	178	80634
		4	5785	178	4118.92
		8	5765	178	8209.36
		22	6000	178	23496
		6	5000	178	5340
		8	9000	178	12816
		4	4885	178	3478.12
		8	6015	178	8565.36
		6	4500	178	4806
		24	7500	178	32040
HEA 320	S275	12	5000	97.6	5856
		24	6000	97.6	14054.4
TOTAL					<b>369364.56</b>

Table 4. 1:List of profiles

## Appendix A - Analysis Procedure

In this study, the calculations are prepared on Jupyter notebook so that it can be utilized in the future design.

### Python Code of Beam Design

```

1. import handcalcs.render
2. from math import sqrt
3. import forallpeople
4. forallpeople.environment('structural', top_level = True)
5. %%render
6. L =(8.5*m) #for half span
7. g_f= 2.43*kN/(m**2)*L #Floor dead load
8. m_b=(0.66*kN/m) #SelfWeight,HEA 600(S355),Assumption
9. g=(g_f + m_b) #Total Dead Load
10. q_f=2.5*(kN/(m**2))*L #Floor Live load
11. q=q_f #Total Live Load
12. w_uls = g*1.35 + q*1.5 #Ultimate Limit State
13. w_sls = g+q #Serviceability Limit State
14. %%render
15. #Parameters
16. h=590*mm #depth
17. b=300*mm #width
18. t_w=13*mm #thickness
19. t_f=25*mm #flange thickness
20. r=27*mm #root radius
21. m_w=177.8*kg/m #weight
22. %%render
23.
24. d=h-2*(t_f+r) #depth between fillets

```

```

25. I_y=1412*10**6*mm**4 #second moment of area
26. pi=3.14159265359 #π pi constant
27. i_y=249.7*mm #radius of gyration
28. I_T=4075*10**3*mm**4 #torsion constant
29. I_w=8.8790*10**12*mm**6 #warping constant
30. G=81000*MPa #shear modulus
31. %%render
32. #Parameters
33. I_z=112.7*10**6*mm**4 #second moment of area
34. i_z=70.5*mm #radius of gyration
35. %%render
36. #Parameters
37. Wpl_y=5350*10**3*mm**3 #plastic section modulus
38. Wpl_z=1156*10**3*mm**3 #plastic section modulus
39. %%render
40. #Parameters
41. Wel_y=4787*10**3*mm**3 #elastic section modulus
42. Wel_z=751.4*10**3*mm**3 #elastic section modulus
43. %%render
44. alpha= 0.49
45. A=22646*mm**2 #area
46. E=210*MPa #elasticity modulus
47. f_y=355*MPa #yield strength
48. f_u=490*MPa #ultimate strength
49. %%render
50. #Parameters
51. gamma_M0= 1.0 #partial safety factor
52. gamma_M1= 1.0 #partial safety factor
53. %%render
54. epsilon= ((235*MPa)/f_y)**(1/2)
55. %%render
56. c= (b-t_w-2*r)/2
57. limit= c/t_f
58. print("The flange in compression is Class 1") if c/t_f < 9*epsilon else print
("The flange in compression is Class 2 ")
59. %%render
60. c_b=d
61. limit= c/t_w
62. print("The web under bending is Class 1") if c_b/t_w < 72*epsilon else print(
"The web under bending is Class 2")
63. %%render
64. eta= 1
65. h_w= h-2*t_f
66. limit= h_w/t_w
67. limit_2= 72*epsilon/eta
68. print("The shear buckling resistance of the web does not need to be verified.
") if h_w/t_w < 72*epsilon/eta else print("The shear buckling resistance of t
he web need to be verified.")
69. %%render
70. A_v= A-(2*b*t_f)+t_f*(t_w+2*r)
71. Vpl_Rd = (A_v*(f_y/3**(1/2)))/gamma_M0
72. %%render
73. V_Ed =334.18*kN
74. Vc_Rd=Vpl_Rd
75. limit= V_Ed/Vc_Rd
76. print("The shear resistance of the section is adequate.") if V_Ed/Vc_Rd <1.0
else print("The shear resistance of the section is not adequate.")
77. %%render
78. limit= Vc_Rd/2
79.
80. #Shear force at maximum bending moment V_Ed= 334.180 kN
81. print("Therefore,no reduction in resistance to bending due to shear is requir
ed.") if Vc_Rd>V_Ed else print("Need reduction in resistance to bending due
to shear is required.")
82. %%render
83. Mc_Rd= Mpl_Rd= (Wpl_y*f_y)/gamma_M0

```

```

84. %%render
85. M_Ed= 496.94*kN*m
86. limit= M_Ed/Mc_Rd
87. print("Therefore,the bending moment capacity is adequate.") if M_Ed/Mc_Rd <1.
    0 else print("Therefore,the bending moment capacity is not adequate.")
88. %%render
89. L_cr=9000*mm
90. epsilon=((235*MPa)/f_y)**(1/2)
91. lambda_1=93.9*epsilon
92. i=i_z
93. lambdabar= (L_cr/i)*(1/lambda_1)
94. UV=0.9
95. beta_w=1
96. C_1=3.75
97. beta=0.75
98. lambdabar_LT0=0.4
99. lambdabar_LT= 1*UV*lambdabar*(beta_w**1/2)/(C_1**1/2)
100. phi_LT = 0.5*(1+alpha*((lambdabar_LT-
    lambdabar_LT0)+(beta*lambdabar_LT**2)))
101. chi_LT = 1/(phi_LT+((phi_LT**2)-(beta*lambdabar_LT**2))**(1/2))
102. chi_LT=1
103. M_b_Rd= chi_LT*W_e1_y*f_y/gamma_M1
104. print("Therefore,the lateral torsional buckling moment capacity is adequate.
    ") if M_Ed/M_b_Rd <1.0 else print("Therefore,the lateral torsional buckling
    moment capacity is not adequate.")

```

### Python Code of Column Design

```

1. import handcalcs.render
2. from math import sqrt
3. import forallpeople
4. forallpeople.environment('structural', top_level = True)
5. %%render
6. # Parameters
7. g_k=0.00243*MPa
8. q_k=0.0025*MPa
9. g_sk=0.0007*MPa
10. q_w=0.00032*MPa
11. B=7500*mm
12. H=8500*mm
13. n_floor=3
14. L_total=34000*mm
15. H_total=12600*mm
16. g_steel=7.85e-7*N/mm**3
17. A_col=7524*mm**2
18. gamma_G=1.35
19. gamma_Q=1.5
20. gamma_M0=1.0
21. psi_0_snow=0.5
22. psi_0_wind=0.6
23. b_c=350*mm #SHS 350*350*16
24. %%render
25. # Parameters
26. #Section properties (SHS 350*350*16)
27. h=350*mm #width
28. b=350*mm #side dimension
29. t=16*mm #thickness
30. L=3500*mm #height
31. r_0=24*mm #outer rounding radius
32. r_i=16*mm #inner rounding radius
33. A=21101*mm**2 #area
34. A_v=10551*mm**2 #shear area
35. %%render
36.
37. I=389.4*10**6*mm**4 #second moment of inertia

```

```

38. W_el=2225*10**3*mm**3 #elastic section modulus
39. W_pl=2630*10**3*mm**3 #plastic section modulus
40. N_pl_Rd=7490.96*kN #design plastic axial force resistance
41. V_pl_Rd=2162.46*kN #design plastic shear axial force resistance
42. M_el_Rd=789.97*kN*m #design elastic bending moment resistance
43. M_pl_Rd=933.53*kN*m #design plastic bending moment resistance
44. T_pl_Rd=668.91*kN #design plastic torsional moment resistance
45. i=i_y=i_z=117*mm #radius of gyration
46. y_0=0
47. W_T=3264*10**3*mm**3 #torsion modulus
48. I_T=609900*10**3*mm**4 #torsion constant
49. f_y=355*MPa #yield strength
50. f_u=490*MPa #ultimate strength
51. gamma_M0=1.00 #resistance of cross-section
52. gamma_M1=1.00 #resistance of members to instability
53. G=81000*MPa #shear modulus
54. alpha=0.21 # buckling curve
55. pi=3.14159265359 #pi constant
56. I_w=1.058*10**12*mm**6 #warping constant
57. E=210*GPa #modulus of elasticity
58. %%render
59. N_c_Rd= A*f_y/gamma_M0#for class 1
60. %%render
61. condition_1= N_Ed/N_c_Rd
62. %%render
63. L_cr=3500*mm
64. epsilon=((235*MPa)/f_y)**(1/2)
65. lambda_1=93.9*epsilon
66. lambdabar= (L_cr/i)*(1/lambda_1)
67. phi = 0.5*(1+alpha*(lambdabar-(0.2))+(lambdabar**2))
68. chi= 1/(phi+((phi**2)-(lambdabar**2))**(1/2))
69. N_b_Rd= chi*A*f_y/gamma_M1
70. %%render
71. condition_2= N_Ed/N_b_Rd
72. %%render
73. i_0=(i_y**2+i_z**2+y_0**2)**(1/2)
74. N_cr_T=(1/i_0**2)*(G*I_T+((pi**2)*E*I_w)/L**2))
75. lambdabar_T= ((A*f_y)/N_cr_T)**(1/2)
76. phi_T = 0.5*(1+alpha*((lambdabar_T-0.2)+(lambdabar_T**2)))
77. chi_T = 1/(phi_T+((phi_T**2)-(lambdabar_T**2))**(1/2))
78. %%render
79. chi_T = 1.00
80. N_bT_Rd= (chi_T*A*f_y)/gamma_M1
81. %%render
82. e=268.5*mm
83. V_Ed_6=107.97*kN
84. M_Ed=M_Ed_6y= V_Ed_6*e
85. %%render
86. UV=0.9
87. beta_w=1
88. C_1=1
89. beta=0.75
90. lambdabar_LT0=0.4
91. lambdabar_LT=1*UV*lambdabar*(beta_w**1/2)/(C_1**1/2)
92. %%render
93. phi_LT = 0.5*(1+alpha*((lambdabar_LT-
lambdabar_LT0)+(beta*lambdabar_LT**2)))
94. chi_LT = 1/(phi_LT+((phi_LT**2)-(beta*lambdabar_LT**2))**(1/2))
95. %%render
96. chi_LT=1
97. M_b_Rd= chi_LT*W_el*f_y/gamma_M1
98. %%render
99. condition_3= (N_Ed/N_b_Rd)+(M_Ed/M_b_Rd)

```

## Python Code of Secondary Beam Design

```
1. import handcalcs.render
2. from math import sqrt
3. import forallpeople
4. forallpeople.environment('structural', top_level = True)
5. %%render
6. L =(9*m) #for longest span
7. g_f= 2.43*kN/(m**2)*L #Floor dead load
8. m_b=(0.96*kN/m) #SelfWeight,HEA 320(S355),Assumption
9. g=(g_f + m_b) #Total Dead Load
10. q_f=2.5*(kN/(m**2))*L #Floor Live load
11. q=q_f #Total Live Load
12. w_uls = g*1.35 + q*1.5 #Ultimate Limit State
13. w_sls = g+q #Serviceability Limit State
14. %%render
15. #Parameters
16. h=310*mm #depth
17. b=300*mm #width
18. t_w=9*mm #thickness
19. t_f=15.5*mm #flange thickness
20. r=27*mm #root radius
21. m_w=97.6*kg/m #weight
22. %%render
23.
24. d=h-2*(t_f+r) #depth between fillets
25. I_y=229.3*10**6*mm**4 #second moment of area
26.
27. i_y=135.8*mm #radius of gyration
28. %%render
29. I_z=69.85*10**6*mm**4 #second moment of area
30. i_z=74.9*mm #radius of gyration
31. %%render
32. Wpl_y=1628*10**3*mm**3 #plastic section modulus
33. Wpl_z=709.7*10**3*mm**3 #plastic section modulus
34. %%render
35. Wel_y=1479*10**3*mm**3 #elastic section modulus
36. Wel_z=465.7*10**3*mm**3 #elastic section modulus
37. %%render
38. A=12437*mm**2 #area
39. E=210*MPa #elasticity modulus
40. f_y=355*MPa #yield strength
41. f_u=490*MPa #ultimate strength
42. %%render
43. #Parameters
44. gamma_M0= 1.0 #partial safety factor
45. gamma_M1= 1.0 #partial safety factor
46. %%render
47. epsilon= ((235*MPa)/f_y)**(1/2)
48. %%render
49. c= (b-t_w-2*r)/2
50. limit= c/t_f
51. print("The flange in compression is Class 1") if c/t_f < 9*epsilon else print
("The flange in compression is Class 2 ")
52. %%render
53. c_b=d
54. limit= c/t_w
55. print("The web under bending is Class 1") if c_b/t_w < 72*epsilon else print(
"The web under bending is Class 2")
56. %%render
57. eta= 1
58. h_w= h-2*t_f
59. limit= h_w/t_w
60. limit_2= 72*epsilon/eta
```

```

61. print("The shear buckling resistance of the web does not need to be verified.
    ") if h_w/t_w < 72*epsilon/eta else print("The shear buckling resistance of t
    he web need to be verified.")
62. %%render
63. A_v= A-(2*b*t_f)+t_f*(t_w+2*r)
64. Vp1_Rd = (A_v*(f_y/3**(1/2)))/gamma_M0
65. %%render
66. V_Ed =195*kN
67. Vc_Rd=Vp1_Rd
68. limit= V_Ed/Vc_Rd
69. print("The shear resistance of the section is adequate.") if V_Ed/Vc_Rd <1.0
    else print("The shear resistance of the section is not adequate.")
70. %%render
71. limit= Vc_Rd/2
72.
73. #Shear force at maximum bending moment V_Ed= 334.180 kN
74. print("Therefore,no reduction in resistance to bending due to shear is requir
    ed.") if Vc_Rd>V_Ed else print("Need reduction in resistance to bending due
    to shear is required.")
75. %%render
76. Mc_Rd= Mpl_Rd= (Wpl_y*f_y)/gamma_M0
77. %%render
78. M_Ed= 496.94*kN*m
79. limit= M_Ed/Mc_Rd
80. print("Therefore,the bending moment capacity is adequate.") if M_Ed/Mc_Rd <1.
    0 else print("Therefore,the bending moment capacity is not adequate.")

```

### Python Code of Connection Design

```

1. import handcalcs.render
2. from math import sqrt
3. import forallpeople
4. forallpeople.environment('structural', top_level = True)
5. %%render
6. f_yb=640*(N/mm/mm)
7. f_ub=800*(N/mm/mm)
8. f_y=355*(MPa)
9. #Parameters
10. b=160*mm
11. d_p=300*mm
12. t=10*mm
13. %%render
14. alpha_v = 0.6
15. gamma_M2 = 1.25
16. A_s=157*(mm*mm)
17.
18. F_vRd = (alpha_v * A_s * f_ub )/ gamma_M2
19. %%render
20. e_2 = 80*mm
21. d_0 = 18*mm
22. k_1 = min (((2.8*e_2/d_0)-1.7) , 2.5)
23. %%render
24. e_1= (50*mm)
25. p_1= (60*mm)
26. f_u= (360*MPa)
27.
28. alpha_b = min((e_1/(3*d_0),(p_1/(3*d_0)-0.25),(f_ub/f_u),(1)))
29. %%render
30. e_1_1= (170*mm)
31. p_1= (60*mm)
32. f_u= (360*MPa)
33.
34. alpha_b = min((e_1_1/(3*d_0),(p_1/(3*d_0)-0.25),(f_ub/f_u),(1)))
35. %%render
36. d=16*mm

```

```

37. t=10*mm
38. F_bRd = k_1*alpha_b*d**t*f_u/gamma_M2
39. %%render
40. V_rd= 4*F_bRd
41. %%render
42. V_Ed=107.97*kN
43. e_boltgroup=268.5*mm
44. M_ed = e_boltgroup * V_Ed
45. %%render
46. a=(6*mm)
47. l=(300*mm)
48. W_el_w= ((2*a*1**2)/6)
49. sigma_w = M_ed/ W_el_w
50. %%render
51. tau_L =sigma_L = sigma_w/2**(1/2)
52. tau_II= V_Ed/(2*a*1)
53. %%render
54. beta_w = 0.9
55. C_1= ((sigma_L**2+3*(tau_L**2+tau_II**2))**(1/2))
56. C_2= f_u/(beta_w*gamma_M2)
57. %%render
58. sigma_L=sigma_w/2**(1/2)
59. C_3= f_u/gamma_M2
60. %%render
61. A_nt = t*(e_2-(d_0/2))
62. A_nv = t*(e_2+p_1-d_0-d_0/2)
63. gamma_M0=1.00
64. V_eff2Rd= ((0.5* A_nt*f_u)/gamma_M2)+ (A_nv*f_y)/(3**(1/2)*gamma_M0)
65. %%render
66. A_v=t*d_p
67. V_plRd= (A_v*f_y)/(3**(1/2)*gamma_M0)
68. %%render
69. t_w=13*mm
70. A_ntw= t_w*(e_2-d_0/2)
71. A_nvw= t_w*(120*mm+e_1+p_1-d_0-(d_0/2))
72. %%render
73. V_eff2Rd= ((0.5*A_ntw*f_u)/gamma_M2)+(A_nvw*f_y)/(3**(1/2)*gamma_M0)
74. %%render
75. M_elRd = (W_el_w * f_y)/gamma_M0

```

## Python Code of Column Base Plate Design

```

1. import handcalcs.render
2. from math import sqrt
3. import forallpeople
4. forallpeople.environment('structural', top_level = True)
5. %%render
6. # Parameters
7. g_k=2.43*kN/m**2
8. q_k=2.5*kN/m**2
9. g_sk=0.7*kN/m**2
10. q_w=0.32*kN/m**2
11. B=7.5*m
12. H=8.5*m
13. n_floor=3
14. L_total=34*m
15. H_total=12.6*m
16. g_steel=785*N/m**3
17. A_col=7524*mm**2
18. gamma_G=1.35
19. gamma_Q=1.5
20. gamma_M0=1.0
21. psi_0_snow=0.5
22. psi_0_wind=0.6

```

```

23. b_c=350*mm #SHS 350*350*16
24. %%render
25. A_load= B*H*2
26. %%render
27. N_self= A_col*g_steel*H_total
28. %%render
29. N_dead= A_load*g_k*n_floor
30. %%render
31. N_live= A_load*q_k*n_floor
32. %%render
33. N_snow= A_load*g_sk
34. %%render
35. M_wind= q_w*(L_total/2)*((H_total**2)/2)
36. N_wind= M_wind/B #Tension force
37. %%render
38. N_Ed= gamma_Q*(N_self+N_dead)+gamma_Q*(N_live+N_snow*psi_0_snow+N_wind*psi_0_
wind)
39. %%render
40. # Parameters
41. t=30*mm
42. b_1=450*mm
43. d_1=450*mm
44. b=500*mm
45. d=500*mm
46. h=500*mm
47. b_r=425*mm
48. #Column Base Material
49. f_y=355*MPa
50. E=210*GPa
51. gamma_0=1.0
52. #Pad footing material: concrete C25/30
53. f_ck=25*MPa
54. gamma_c=1.5
55. #Grout not specified
56. %%render
57. b_2= min(b_1+2*b_r,3*b_1,b_1+h)
58. d_2= min(d_1+2*b_r,3*d_1,d_1+h)
59. %%render
60. k_j= ((b_2*d_2)/(b_1*d_1))**(1/2)
61. beta_j =2/3 #joint coeff.
62. %%render
63. f_jd= (beta_j*k_j*f_ck)/gamma_c
64. %%render
65. c= t*(f_y/(3*f_jd*gamma_M0))**(1/2)
66. %%render
67. A_eff=(2*c+b_c)**2
68. %%render
69. N_b_Rd= A_eff*f_jd
70. %%render
71. R_condition = N_Ed/N_b_Rd
72. %%render
73. V_Ed= gamma_Q*q_w*(L_total/2)*(H_total/2)
74. %%render
75. # Parameters
76. f_ub=490*MPa
77. f_yb=355*MPa
78. A_s=245*mm**2
79. n_b=2
80. alpha_v=0.5
81. gamma_M2=1.25
82. %%render
83. alpha_b=0.44-0.0003*1/MPa*f_yb
84. F_1vb_Rd= 0.85*(alpha_v*f_ub*A_s)/gamma_M2
85. F_2vb_Rd= (alpha_b*f_ub*A_s)/gamma_M2
86. %%render
87. F_vb_Rd= min(F_1vb_Rd,F_2vb_Rd)

```



```

88. %%render
89. V_Rd= n_b*F_vb_Rd
90. %%render
91. R_condition2= V_Ed/V_Rd

```

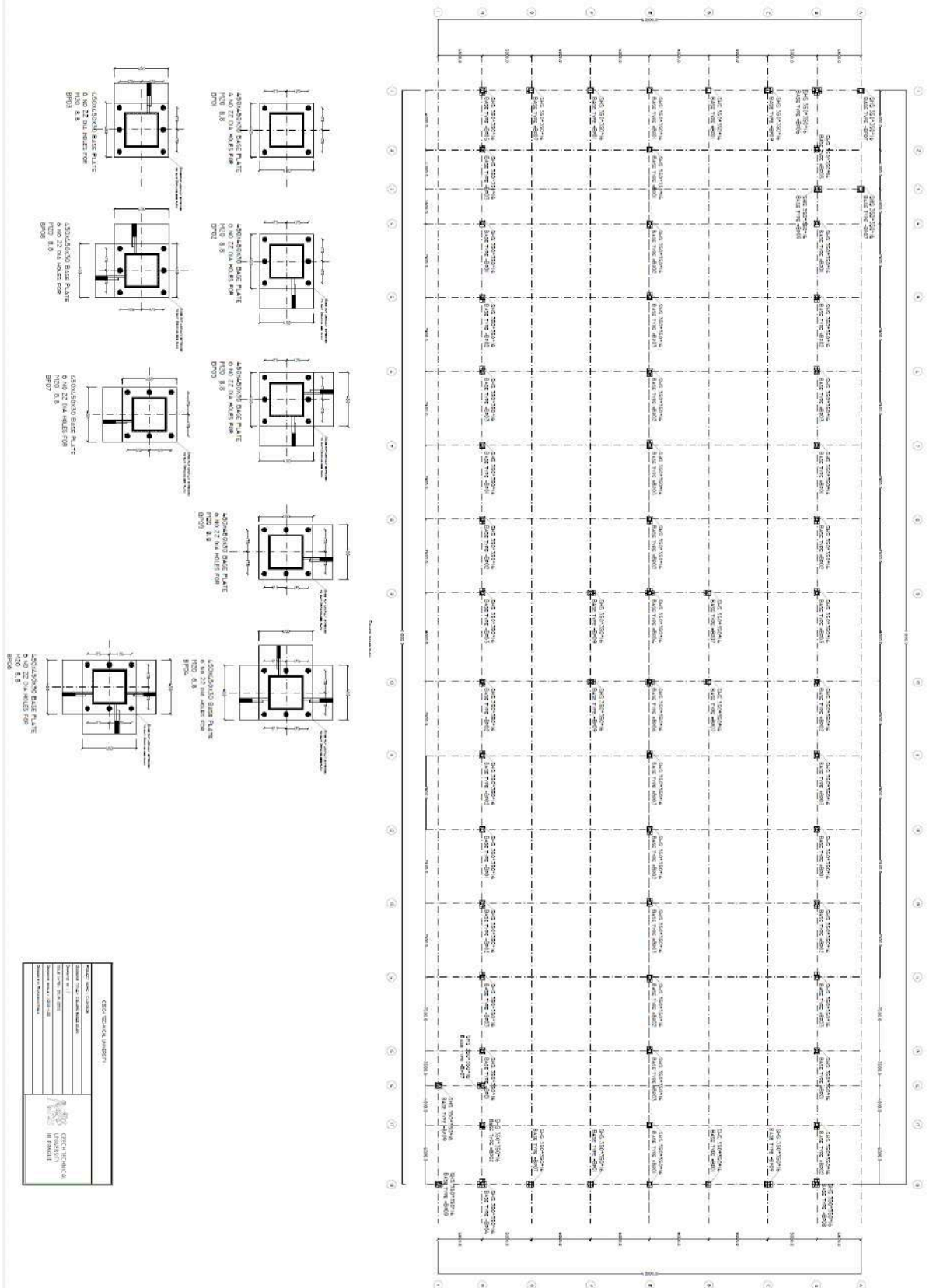
### Python Code of Bracing Member Design

```

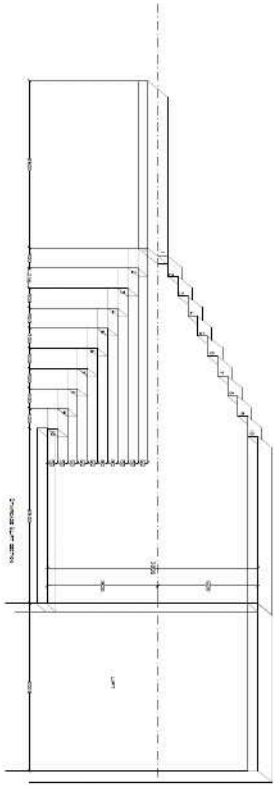
1. import handcalcs.render
2. from math import sqrt
3. import forallpeople
4. forallpeople.environment('structural', top_level = True)
5. %%render
6. b=120*mm
7. t=14.2*mm
8. A_s=5790*mm**2
9. f_y=275*MPa
10. L=8276*mm
11. E=210*GPa
12. I_y=10500000*mm**4
13. I_z=10500000*mm**4
14. gamma_M0=1.0
15. gamma_M1=1.0
16. alpha=0.21
17. pi=3.14159265359
18. %%render
19. N_Ed=259.22*kN #from SCIA
20. N_c_Rd= (A_s*f_y)/gamma_M0
21. %%render
22. C_1= N_Ed/N_c_Rd
23. %%render
24. L_cr=L
25. N_cr= (pi**2*E*I_z)/L_cr**2
26. lambdabar= ((A_s*f_y)/N_cr)**(1/2)
27. phi = 0.5*(1+alpha*(lambdabar-(0.2))+(lambdabar**2))
28. chi= 1/(phi+((phi**2)-(lambdabar**2))**(1/2))
29. N_b_Rd= chi*A_s*f_y/gamma_M1
30. %%render
31. C_2= N_Ed/N_b_Rd

```

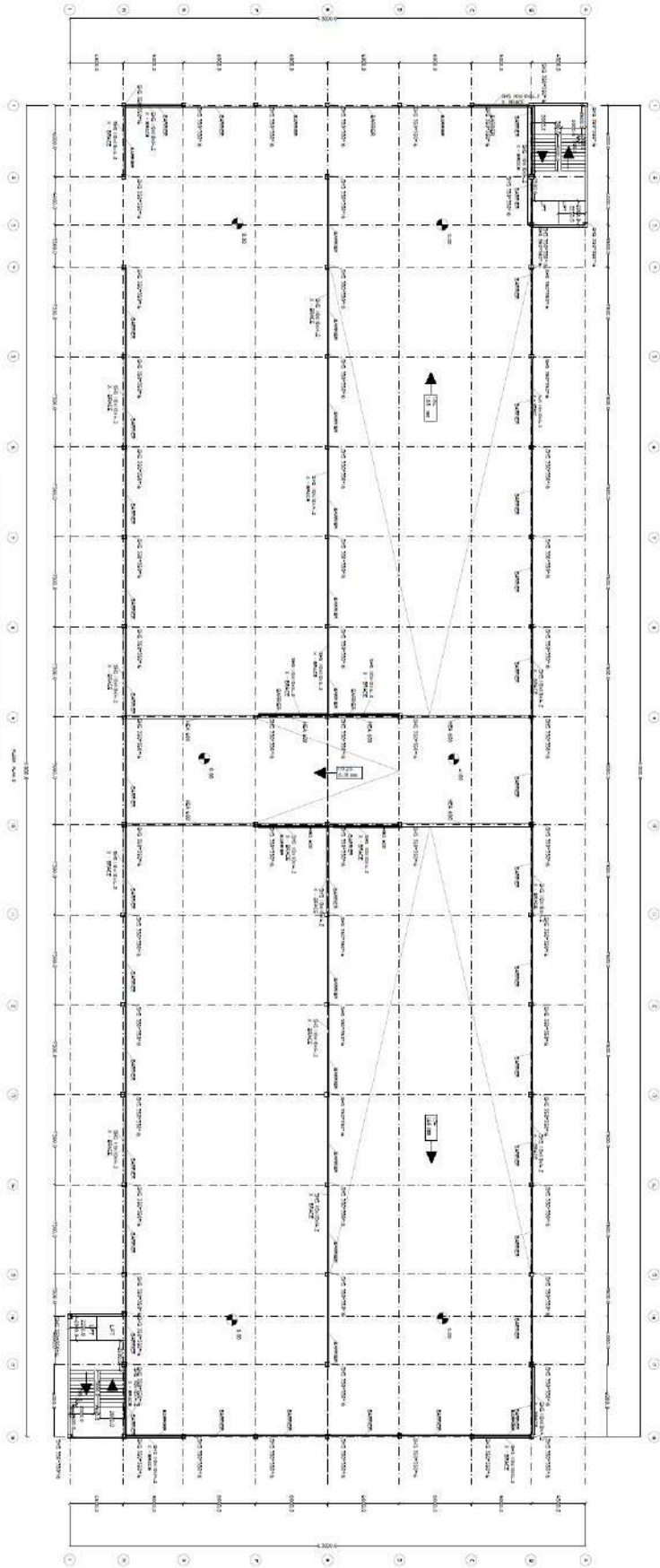
# Appendix B - Drawings

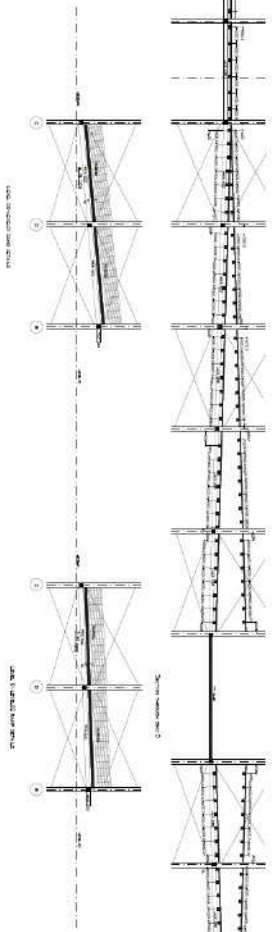
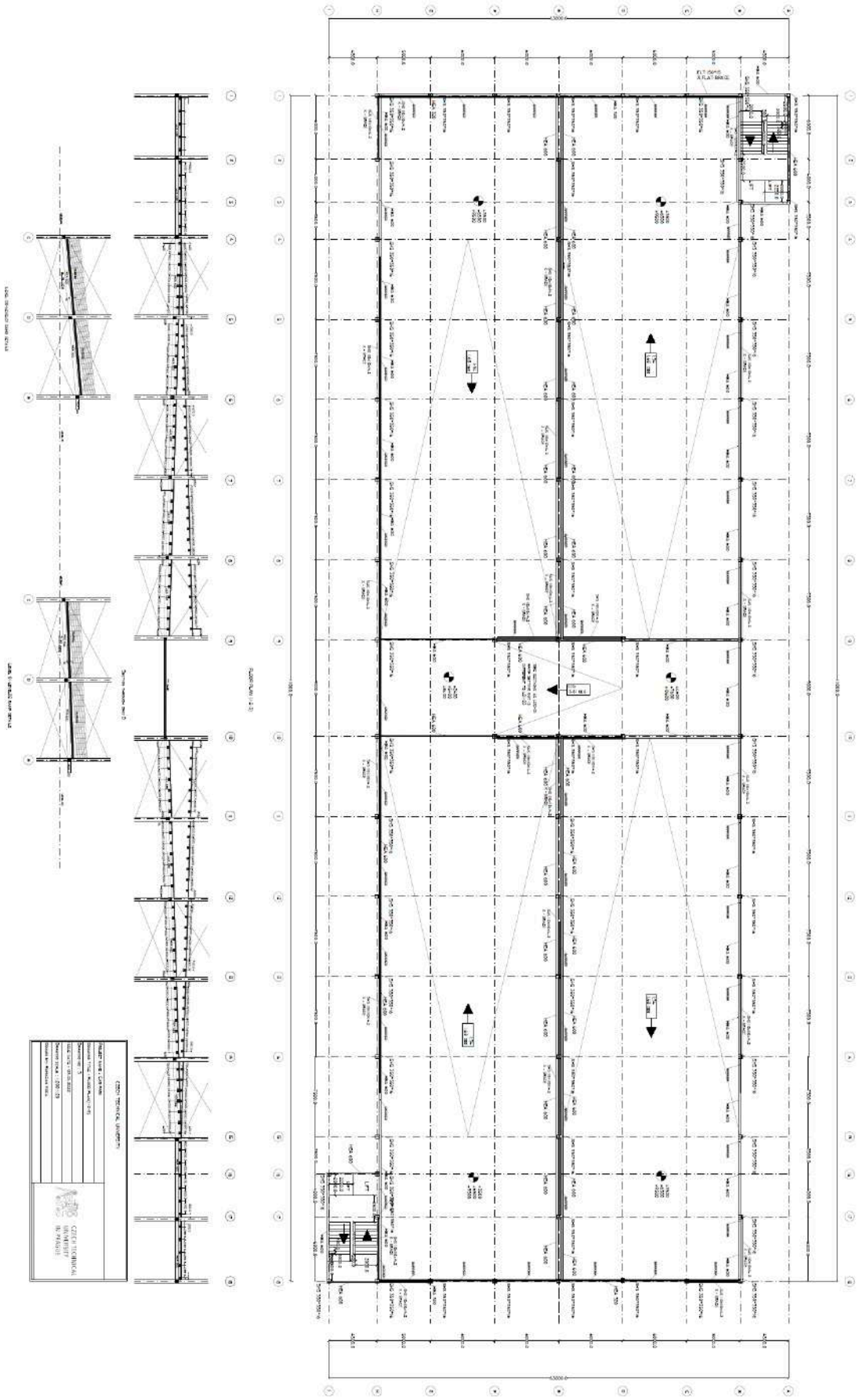


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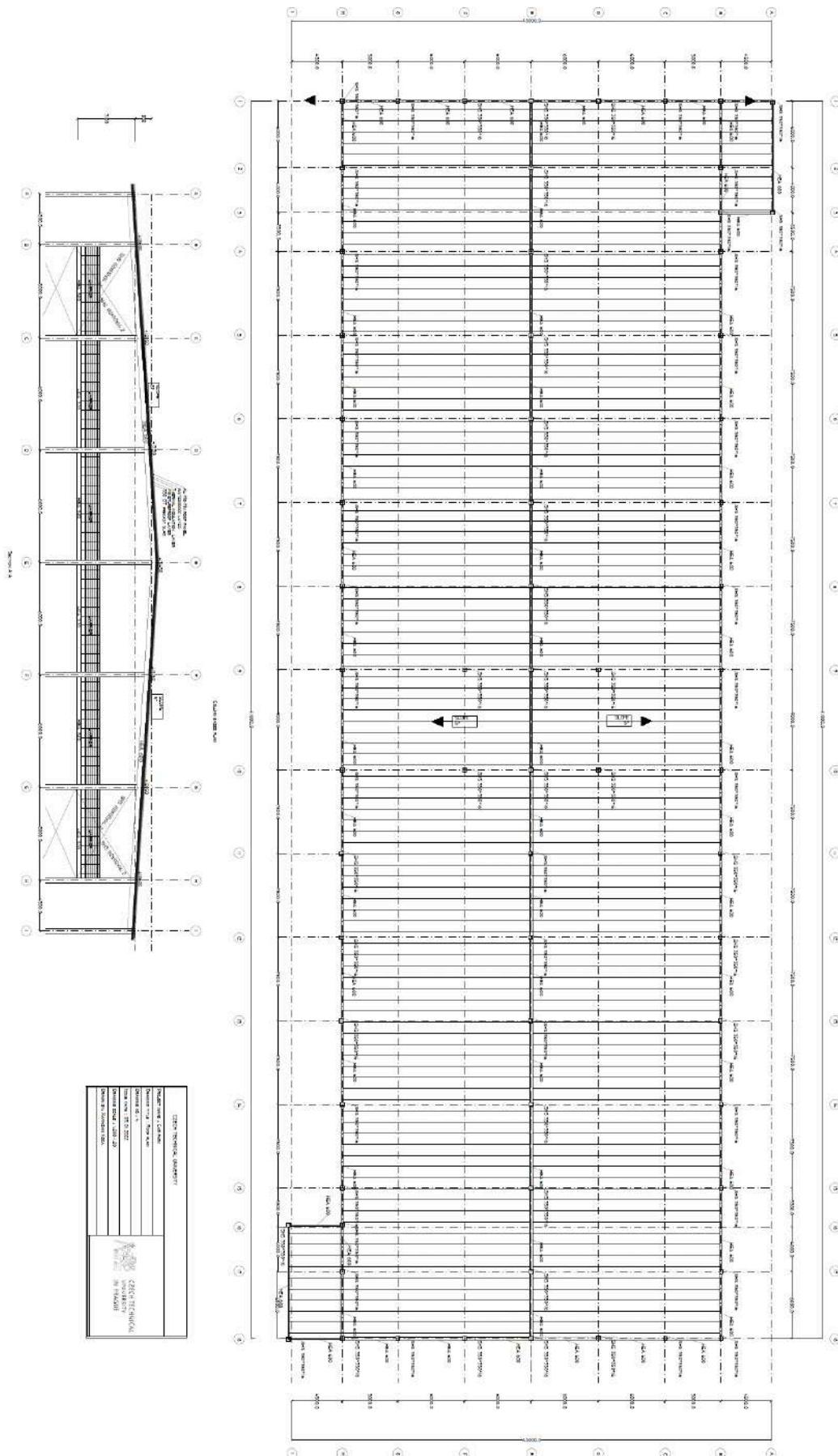


<p>PROYECTO: [illegible]</p> <p>CLIENTE: [illegible]</p> <p>FECHA: [illegible]</p> <p>ESCALA: [illegible]</p> <p>PROYECTISTA: [illegible]</p> <p>PROYECTO: [illegible]</p> <p>PROYECTISTA: [illegible]</p>	



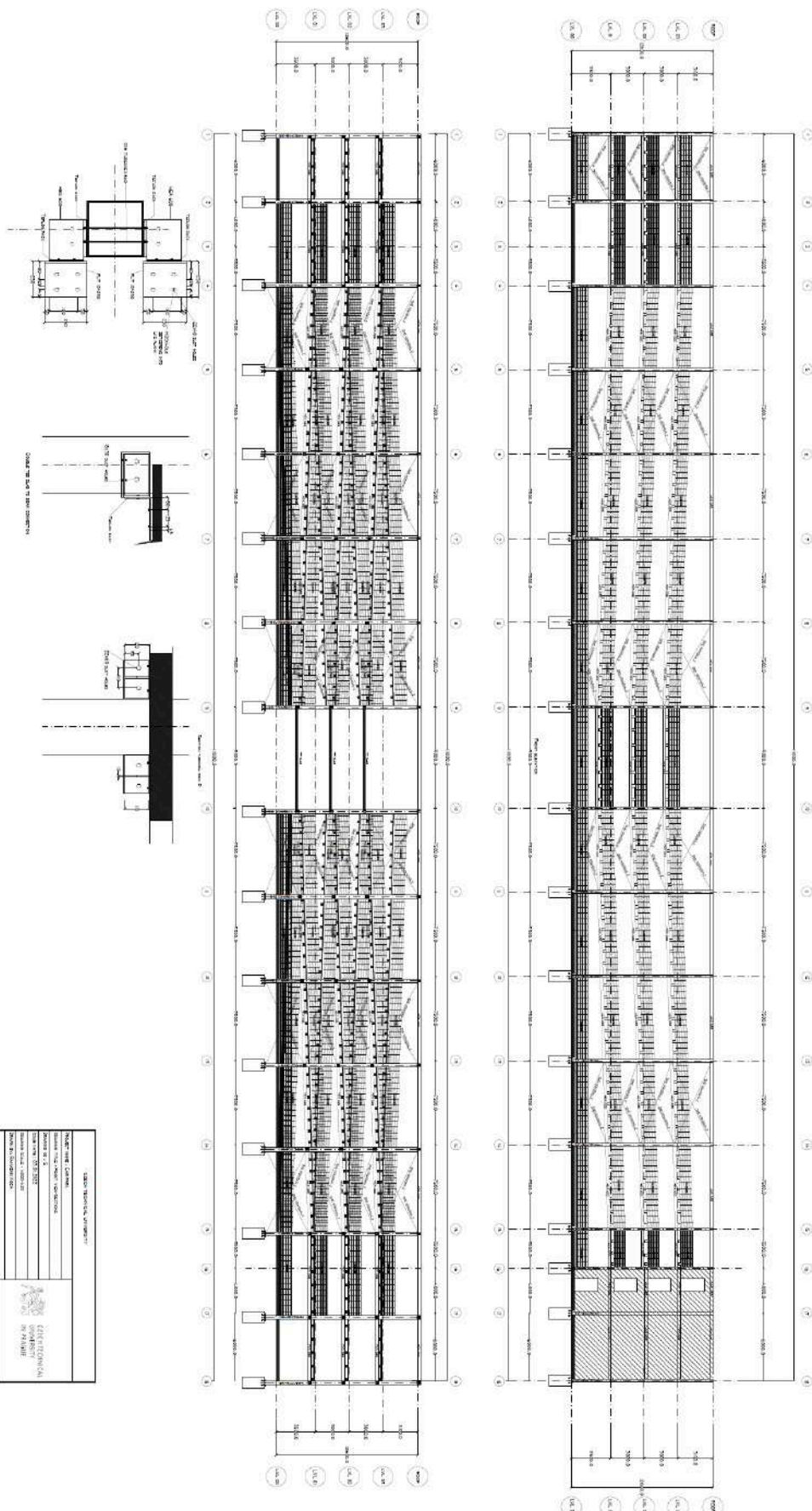


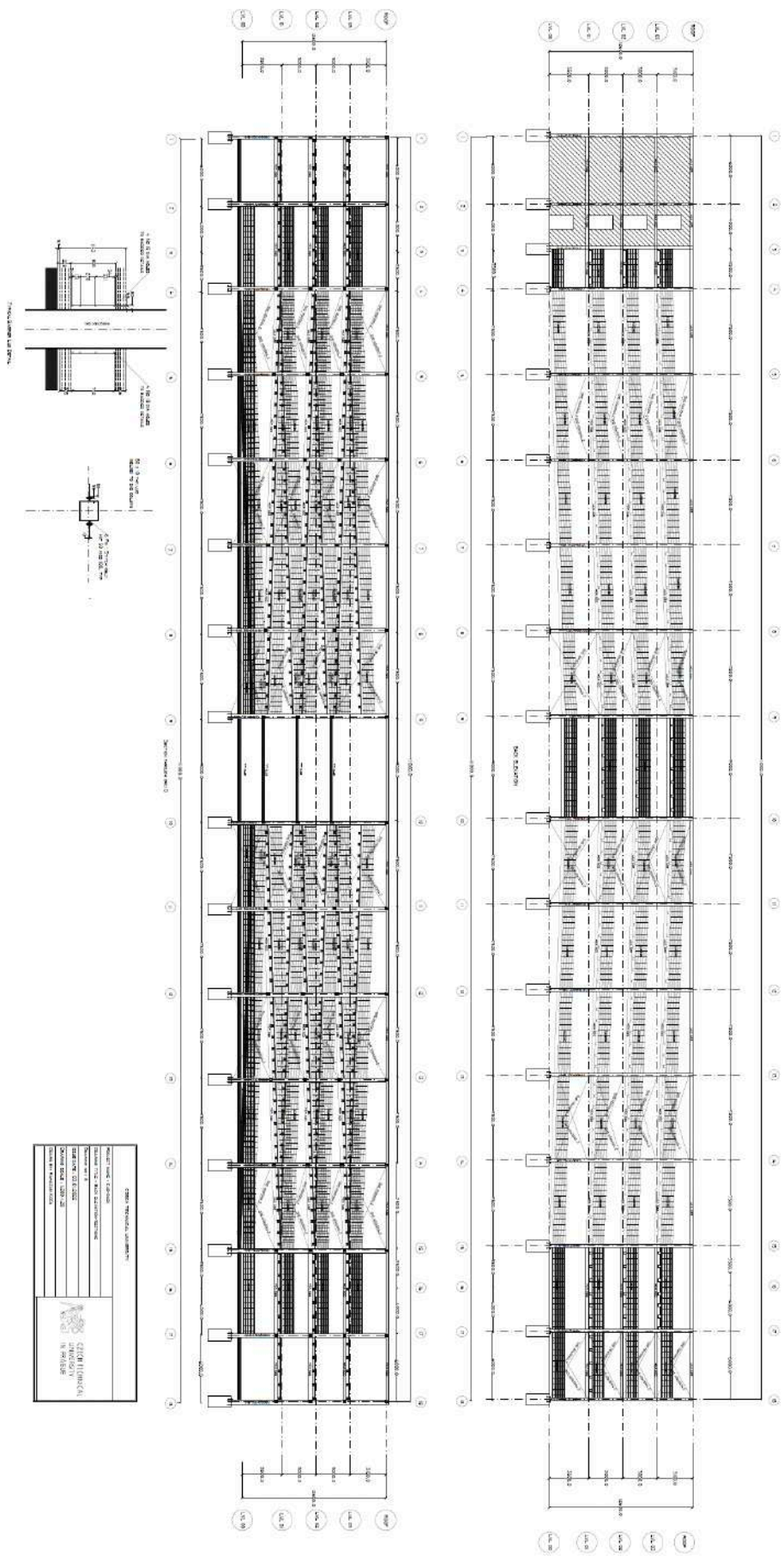
<p>PROJECT: TECHNICAL UNIVERSITY</p>	
<p>DATE: 15.05.2024</p>	<p>SCALE: 1/50</p>
<p>DESIGNER: [Name]</p>	<p>CHECKER: [Name]</p>
<p>APPROVER: [Name]</p>	<p>CLIENT: [Name]</p>



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[5] OneSteel(2004):Economical Carparks- A Design Guide,2<sup>nd</sup> Ed., [One Steel Market Mills,November 2004]

### Further Reading

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Brettle , M.E.,&Brown,D.G.(Eds.).(2009):*Steel building design: Worked examples for students in accordance with Eurocode and the UK National Annexes.* Berkshire: The Steel Construction Institution.

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